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CHAPTER 1-STEAM

ONE AND TWO PIPE SYSTEMS - MAINS AND RISERS

The steam system piping recommendations that follow are designed to help guide an individual working on existing systems; systems that may require alterations or additions for a variety of reasons. These recommendations are conservative but in light of many unknown variables, they will serve one well.

TWO PIPE STEAM

1. STEAM MAIN Capacity in Sq. Ft. E.D.R.

| Size of Pipe in In. | Distance of Main | | |
|------------------------|------------------|--------|--------|
| | 100' | 200' | 300' |
| 2 | 650 | 461 | 375 |
| 2-1/2 | 1,030 | 731 | 595 |
| 3 | 1,860 | 1,320 | 1,075 |
| 4 | 3,800 | 2,698 | 2,196 |
| 6 | 11,250 | 7,987 | 6,502 |
| 8 | 22,250 | 15,797 | 12,860 |
| 10 | 40,800 | 28,968 | 23,582 |
| 12 | 66,000 | 46,860 | 38,148 |

Note: Mains must be pitched for steam & condensate to flow in same direction.

2. STEAM RISERS

Maximum Capacity (Sq. Ft. E.D.R.) at Various Riser Pitch With
Steam and Condensate Flowing in Opposite Direction.

| Pipe Size in In. | Pitch of Pipe per 10' Length | | | | |
|---------------------|------------------------------|-------|-------|-------|-------|
| | 1" | 2" | 3" | 4" | 5" |
| 3/4 | 22.8 | 28.4 | 33.2 | 38.6 | 42.0 |
| 1 | 46.8 | 59.2 | 69.2 | 76.8 | 82.0 |
| 1-1/4 | 79.6 | 108.0 | 125.2 | 133.6 | 152.4 |
| 1-1/2 | 132.0 | 168.0 | 187.2 | 203.2 | 236.8 |
| 2 | 275.2 | 371.6 | 398.4 | 409.6 | 460.0 |

Note: Above E.D.R. is maximum. Greater load will cause problems with increased steam velocities.

3. CONDENSATE RETURN MAINS (Horizontal, Pitched)

| Capacity based on sq. ft. E.D.R. (2 oz PD per 100') | | |
|---|------------|------------|
| Pipe Size in In. | Dry Return | Wet Return |
| 1 | 370 | 900 |
| 1-1/4 | 780 | 1,530 |
| 1-1/2 | 1,220 | 2,430 |
| 2 | 2,660 | 5,040 |
| 2-1/2 | 4,420 | 8,460 |
| 3 | 8,100 | 13,500 |
| 4 | 17,380 | 27,900 |

CHAPTER 1-STEAM

4. CONDENSATE RETURN RISERS (Vertical, Horizontal Pitch)

| Capacity Based on Sq. Ft. (E.D.R.) (2 oz. PD per 100') | |
|--|------------|
| Pipe Size | Dry Return |
| 3/4" | 170 |
| 1 " | 410 |
| 1 1/4" | 890 |
| 1 1/2" | 1,350 |

ONE PIPE STEAM

1. STEAM MAIN

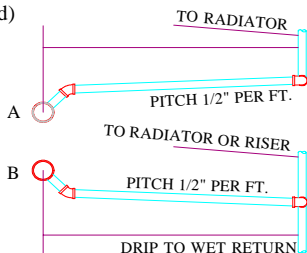
| Capacity in Sq. Ft. (E.D.R.) (Based on 2 oz. PD) | | |
|--|----------------|----------------|
| Pipe Size Inches | T.E.L. 100' | T.E.L. 200' |
| 2 " | 350 | 235 |
| 2 1/2" | 580 | 385 |
| 3 " | 1,050 | 700 |
| 4 " | 2,210 | 1,480 |
| 6 " | 6,880 | 4,480 |

Notes: Mains must be pitched for steam & condensate to flow in same direction. If steam and condensate counterflows, increase piping one size.

2. STEAM RISER - Capacity in Sq. Ft. (E.D.R.)

(Up-feed Riser - Horizontal Travel
8' Max. - Riser Not Dripped)

| Pipe Size Inches | Sq. Ft. E.D.R. |
|---------------------|-------------------|
| 1 " | 25 |
| 1 1/4" | 55 |
| 1 1/2" | 85 |
| 2 " | 150 |



Notes:

- (A) Horizontal travel beyond 10' increase one size
- (B) Horizontal travel sloped down and dripped decrease one size

CHAPTER 1-STEAM

3. DRY RETURN - Capacity in Sq. Ft. (E.D.R.)

| Pipe Size | Sq. Ft. (E.D.R.) |
|-----------|------------------|
| 1-1/4" | 640 |
| 1-1/2" | 1,000 |
| 2" | 2,180 |

4. WET RETURN - Capacity in Sq. Ft. (E.D.R.)

| Pipe Size | Sq. Ft. (E.D.R.) |
|-----------|------------------|
| 1-1/4" | 1,100 |
| 1-1/2" | 1,700 |
| 2" | 3,600 |

| NUMBER OF SMALLER PIPES EQUIVALENT TO ONE LARGER PIPE | | | | | | | | | | | | |
|---|-----------|------|------|--------|--------|------|--------|------|--------|------|------|------|
| PIPE SIZE | PIPE SIZE | | | | | | | | | | | |
| | 1/2" | 3/4" | 1" | 1-1/4" | 1-1/2" | 2" | 2-1/2" | 3" | 3-1/2" | 4" | 5" | 6" |
| 1/2" | 1 | 2.27 | 4.88 | 10 | 15.8 | 31.7 | 52.9 | 96.9 | 140 | 205 | 377 | 620 |
| 3/4" | | 1 | 2.05 | 4.3 | 6.97 | 14 | 23.2 | 42.5 | 65 | 90 | 166 | 273 |
| 1" | | | 1 | 2.25 | 3.45 | 6.82 | 11.4 | 20.9 | 30 | 44 | 81 | 133 |
| 1-1/4" | | | | 1 | 1.5 | 3.1 | 5.25 | 9.1 | 12 | 19 | 37 | 68 |
| 1-1/2" | | | | | 1 | 2 | 3.34 | 6.13 | 9 | 13 | 23 | 39 |
| 2" | | | | | | 1 | 1.67 | 3.06 | 4.5 | 6.5 | 11.9 | 19.6 |
| 2-1/2" | | | | | | | 1 | 1.82 | 2.7 | 3.87 | 7.12 | 11.7 |
| 3" | | | | | | | | 1 | 1.5 | 2.12 | 3.89 | 6.39 |
| 3-1/2" | | | | | | | | | 1 | 1.25 | 2.5 | 4.25 |
| 4" | | | | | | | | | | 1 | 1.84 | 3.02 |
| 5" | | | | | | | | | | | 1 | 1.65 |
| 6" | | | | | | | | | | | | 1 |

STEAM VELOCITIES IN STEAM PIPING

To obtain steam velocity in feet per second, multiply load by proper factor shown below.

| | Initial Pressure of Dry Steam | PIPE SIZE - INCHES | | | | | | | | | | | |
|--------------------|-------------------------------|--------------------|--------|--------|-------|--------|-------|-------|-------|-------|-------|-------|--------|
| | | 1" | 1-1/4" | 1-1/2" | 2" | 2-1/2" | 3" | 4" | 5" | 6" | 8" | 10" | 12" |
| Sq. Ft. EDR Factor | 5 PSI Gauge | | | .1000 | .0610 | .0425 | .0276 | .0160 | .0101 | .0071 | .0041 | .0026 | .00182 |
| | 2 PSI Gauge | .2750 | .1580 | .1170 | .0700 | .0490 | .0320 | .0186 | .0118 | .0082 | .0047 | .0030 | .0021 |
| | 0 PSI Gauge | .3094 | .1778 | .1316 | .0788 | .0551 | .0360 | .0209 | .0133 | .0093 | .0054 | .0034 | .0024 |

FRICTION LOSSES - FLOW OF STEAM THROUGH PIPES

Based On: Dry Steam at 2PSI Gauge Initial Pressure. Schedule 40 Steel Pipe.
Loads: Are in Sq. Ft. EDR.

Friction Losses: Are in ounces per 100 linear feet of pipe, or the equivalent.
For Capacities In Pounds Per Hour: Divide Sq. Ft. EDR by 4.

| LOAD SQ. FT. EDR | PRESSURE LOSS OUNCES - PER 100 FEET OF STEEL PIPE | | | | | | | |
|------------------------|---|------|--------|--------|------|--------|------|------|
| | PIPE SIZE - INCHES | | | | | | | |
| | 3/4" | 1" | 1 1/4" | 1 1/2" | 2" | 2 1/2" | 3" | 4" |
| 30 | 0.80 | | | | | | | |
| 35 | 1.05 | | | | | | | |
| 40 | 1.31 | | | | | | | |
| 45 | 1.62 | | | | | | | |
| 50 | 1.95 | .60 | | | | | | |
| 75 | 4.01 | 1.21 | | | | | | |
| 100 | 6.7 | 2.05 | .54 | | | | | |
| 125 | 10.2 | 3.07 | .80 | | | | | |
| 150 | 14.3 | 4.27 | 1.12 | .54 | | | | |
| 175 | 18.9 | 5.7 | 1.48 | .70 | | | | |
| 200 | 24.3 | 7.2 | 1.88 | .89 | | | | |
| 225 | | 9.0 | 2.33 | 1.11 | | | | |
| 250 | | 10.9 | 2.83 | 1.33 | | | | |
| 275 | | 13.2 | 3.35 | 1.59 | | | | |
| 300 | | 15.4 | 3.92 | 1.86 | | | | |
| 350 | | 20.6 | 5.2 | 2.47 | | | | |
| 400 | | 26.5 | 6.7 | 3.15 | .92 | | | |
| 450 | | | 8.3 | 3.90 | 1.14 | | | |
| 500 | | | 10.0 | 4.75 | 1.38 | | | |
| 550 | | | 12.0 | 5.6 | 1.64 | | | |
| 600 | | | 14.1 | 6.6 | 1.91 | | | |
| 650 | | | 16.4 | 7.7 | 2.22 | | | |
| 700 | | | 19.0 | 8.8 | 2.55 | 1.06 | | |
| 800 | | | 24.4 | 11.4 | 3.27 | 1.35 | | |
| 900 | | | | 14.1 | 4.05 | 1.68 | | |
| 1000 | | | | 17.2 | 4.93 | 2.04 | | |
| 1100 | | | | 20.6 | 5.9 | 2.43 | | |
| 1200 | | | | 24.3 | 6.9 | 2.85 | | |
| 1300 | | | | 28.3 | 8.1 | 3.32 | 1.14 | |
| 1400 | | | | | 9.3 | 3.81 | 1.30 | |
| 1500 | | | | | 10.6 | 4.31 | 1.48 | |
| 1600 | | | | | 11.9 | 4.89 | 1.66 | |
| 1700 | | | | | 13.4 | 5.5 | 1.86 | |
| 1800 | | | | | 14.9 | 6.1 | 2.07 | |
| 1900 | | | | | 16.6 | 6.8 | 2.28 | |
| 2000 | | | | | 18.2 | 7.5 | 2.51 | |
| 2200 | | | | | 21.8 | 8.9 | 3.00 | |
| 2400 | | | | | 25.8 | 10.6 | 3.53 | |
| 2600 | | | | | | 12.3 | 4.08 | 1.08 |
| 2800 | | | | | | 14.1 | 4.73 | 1.24 |
| 3000 | | | | | | 16.1 | 5.4 | 1.40 |
| 3200 | | | | | | 18.2 | 6.1 | 1.58 |
| 3500 | | | | | | 21.6 | 7.2 | 1.86 |
| 4000 | | | | | | 27.9 | 9.2 | 2.41 |
| 4500 | | | | | | | 11.5 | 2.98 |
| 5000 | | | | | | | 14.1 | 3.62 |
| 5500 | | | | | | | 16.9 | 4.34 |

CHAPTER 1-STEAM

PROPERTIES OF SATURATED STEAM

| | Pressure psig | Temp °F | Heat in BTU/lb. | | | Specific Volume Cu. ft. per lb. |
|------------|------------------|------------|-----------------|--------|-------|--|
| | | | Sensible | Latent | Total | |
| Inches Vac | 25 | 134 | 102 | 1017 | 1119 | 142 |
| | 20 | 162 | 129 | 1001 | 1130 | 73.9 |
| | 15 | 179 | 147 | 990 | 1137 | 51.3 |
| | 10 | 192 | 160 | 982 | 1142 | 39.4 |
| | 5 | 203 | 171 | 976 | 1147 | 31.8 |
| | 0 | 212 | 180 | 970 | 1150 | 26.8 |
| | 1 | 215 | 183 | 968 | 1151 | 25.2 |
| | 2 | 219 | 187 | 966 | 1153 | 23.5 |
| | 3 | 222 | 190 | 964 | 1154 | 22.3 |
| | 4 | 224 | 192 | 962 | 1154 | 21.4 |
| | 5 | 227 | 195 | 960 | 1155 | 20.1 |
| | 6 | 230 | 198 | 959 | 1157 | 19.4 |
| | 7 | 232 | 200 | 957 | 1157 | 18.7 |
| | 8 | 233 | 201 | 956 | 1157 | 18.4 |
| | 9 | 237 | 205 | 954 | 1159 | 17.1 |
| | 10 | 239 | 207 | 953 | 1160 | 16.5 |
| | 12 | 244 | 212 | 949 | 1161 | 15.3 |
| | 14 | 248 | 216 | 947 | 1163 | 14.3 |
| | 16 | 252 | 220 | 944 | 1164 | 13.4 |
| | 18 | 256 | 224 | 941 | 1165 | 12.6 |
| | 20 | 259 | 227 | 939 | 1166 | 11.9 |
| | 22 | 262 | 230 | 937 | 1167 | 11.3 |
| | 24 | 265 | 233 | 934 | 1167 | 10.8 |
| | 26 | 268 | 236 | 933 | 1169 | 10.3 |
| | 28 | 271 | 239 | 930 | 1169 | 9.85 |
| | 30 | 274 | 243 | 929 | 1172 | 9.46 |
| | 32 | 277 | 246 | 927 | 1173 | 9.10 |
| | 34 | 279 | 248 | 925 | 1173 | 8.75 |
| | 36 | 282 | 251 | 923 | 1174 | 8.42 |
| | 38 | 284 | 253 | 922 | 1175 | 8.08 |
| | 40 | 286 | 256 | 920 | 1176 | 7.82 |
| | 42 | 289 | 258 | 918 | 1176 | 7.57 |
| | 44 | 291 | 260 | 917 | 1177 | 7.31 |
| | 46 | 293 | 262 | 915 | 1177 | 7.14 |
| | 48 | 295 | 264 | 914 | 1178 | 6.94 |
| | 50 | 298 | 267 | 912 | 1179 | 6.68 |
| | 55 | 300 | 271 | 909 | 1180 | 6.27 |
| | 60 | 307 | 277 | 906 | 1183 | 5.84 |
| | 65 | 312 | 282 | 901 | 1183 | 5.49 |
| | 70 | 316 | 286 | 898 | 1184 | 5.18 |
| | 75 | 320 | 290 | 895 | 1185 | 4.91 |
| | 80 | 324 | 294 | 891 | 1185 | 4.67 |
| | 85 | 328 | 298 | 889 | 1187 | 4.44 |
| | 90 | 331 | 302 | 886 | 1188 | 4.24 |
| | 95 | 335 | 305 | 883 | 1188 | 4.05 |
| | 100 | 338 | 309 | 880 | 1189 | 3.89 |
| | 105 | 341 | 312 | 878 | 1190 | 3.74 |
| | 110 | 344 | 316 | 875 | 1191 | 3.59 |

CHAPTER 1-STEAM

RELATIONS OF ALTITUDE, PRESSURE & BOILING POINT

| Altitude Feet | Atmospheric Pressure Absolute | | Boiling Point of Water °F (Gage Pressure PSI) | | | | |
|------------------|-------------------------------------|------------------------|--|-------|-------|-------|-------|
| | Inches of Mercury (Barometer) | Lbs. per Sq. In. | 0 | 1 | 5 | 10 | 15 |
| -500 | 30.46 | 14.96 | 212.8 | 216.1 | 227.7 | 239.9 | 250.2 |
| -100 | 30.01 | 14.74 | 212.3 | 215.5 | 227.2 | 239.4 | 249.9 |
| Sea Level | 29.90 | 14.69 | 212.0 | 215.3 | 227.0 | 239.3 | 249.7 |
| 500 | 29.35 | 14.42 | 211.0 | 214.4 | 226.3 | 238.7 | 249.2 |
| 1000 | 28.82 | 14.16 | 210.1 | 213.5 | 225.5 | 238.1 | 248.6 |
| 1500 | 28.30 | 13.90 | 209.4 | 212.7 | 225.0 | 237.6 | 248.2 |
| 2000 | 27.78 | 13.65 | 208.2 | 211.7 | 224.1 | 236.8 | 247.7 |
| 2500 | 27.27 | 13.40 | 207.3 | 210.9 | 223.4 | 236.3 | 247.2 |
| 3000 | 26.77 | 13.15 | 206.4 | 210.1 | 222.7 | 235.7 | 246.7 |
| 3500 | 26.29 | 12.91 | 205.5 | 209.2 | 222.1 | 235.1 | 246.2 |
| 4000 | 25.81 | 12.68 | 204.7 | 208.4 | 221.4 | 234.6 | 245.7 |
| 4500 | 25.34 | 12.45 | 203.7 | 207.5 | 220.7 | 234.0 | 245.2 |
| 5000 | 24.88 | 12.22 | 202.7 | 206.8 | 220.1 | 233.4 | 244.7 |
| 6000 | 23.98 | 11.78 | 200.9 | 205.0 | 218.7 | 232.4 | 243.8 |
| 7000 | 23.11 | 11.35 | 199.1 | 203.3 | 217.3 | 231.3 | 242.9 |
| 8000 | 22.28 | 10.94 | 197.4 | 201.6 | 216.1 | 230.3 | 242.0 |
| 9000 | 21.47 | 10.55 | 195.7 | 200.0 | 214.8 | 229.3 | 241.3 |
| 10000 | 20.70 | 10.17 | 194.0 | 198.4 | 213.5 | 228.3 | 240.4 |
| 11000 | 19.95 | 9.80 | 192.2 | 196.8 | 212.3 | 227.3 | 239.6 |
| 12000 | 19.23 | 9.45 | 190.6 | 195.2 | 211.1 | 226.3 | 238.7 |
| 13000 | 18.53 | 9.10 | 188.7 | 193.6 | 209.9 | 225.4 | 237.9 |
| 14000 | 17.86 | 8.77 | 187.2 | 192.3 | 208.8 | 224.5 | 237.2 |
| 15000 | 17.22 | 8.46 | 185.4 | 190.6 | 207.6 | 223.6 | 236.4 |

CHAPTER 1-STEAM

EXPANSION OF PIPES IN INCHES PER 100 FT

| Temp. Degrees F. | Cast Iron | Wrought Iron | Steel | Brass or Copper |
|-----------------------------|----------------------|-------------------------|--------------|----------------------------|
| 0 | 0.00 | 0.00 | 0.00 | 0.00 |
| 50 | 0.36 | 0.40 | 0.38 | 0.57 |
| 100 | 0.72 | 0.79 | 0.76 | 1.14 |
| 125 | 0.88 | 0.97 | 0.92 | 1.40 |
| 150 | 1.10 | 1.21 | 1.15 | 1.75 |
| 175 | 1.28 | 1.41 | 1.34 | 2.04 |
| 200 | 1.50 | 1.65 | 1.57 | 2.38 |
| 225 | 1.70 | 1.87 | 1.78 | 2.70 |
| 250 | 1.90 | 2.09 | 1.99 | 3.02 |
| 275 | 2.15 | 2.36 | 2.26 | 3.42 |
| 300 | 2.35 | 2.58 | 2.47 | 3.74 |
| 325 | 2.60 | 2.86 | 2.73 | 4.13 |
| 350 | 2.80 | 3.08 | 2.94 | 4.45 |

CHAPTER 1-STEAM

STEAM SUSPENDED UNIT HEATERS/UNIT VENTILATORS

Output is normally based on 2PSI steam/60° entering air.

Output will increase or decrease with changes in steam pressure and/or entering air.

| FACTORS | | | FOR CONVERSION OF BASIC STEAM RATINGS @ 2 lbs. ga./60 deg. entering air to various conditions of steam pressure and air temperature | | | | | | | | | | | |
|--------------------------|------------|--------------------|---|------|------|------|------|------|------|------|------|------|-----|-----|
| STEAM | | | ENTERING AIR TEMP. DEG. F. | | | | | | | | | | | |
| lbs. per sq. in | Temp °F | l. heat Btu/lb. | -10 | 0 | 10 | 20 | 30 | 40 | 50 | 60 | 70 | 80 | 90 | 100 |
| 0 | 212.0 | 970.2 | | | | | 1.19 | 1.11 | 1.03 | .96 | .88 | .81 | .74 | .67 |
| 2 | 218.5 | 966.2 | | | | 1.32 | 1.24 | 1.16 | 1.08 | 1.00 | .93 | .85 | .78 | .71 |
| 5 | 227.2 | 960.5 | 1.64 | 1.55 | 1.46 | 1.37 | 1.29 | 1.21 | 1.13 | 1.05 | .97 | .90 | .83 | .76 |
| 10 | 239.4 | 952.5 | 1.73 | 1.64 | 1.55 | 1.46 | 1.38 | 1.29 | 1.21 | 1.13 | 1.06 | .98 | .91 | .84 |
| 15 | 249.7 | 945.5 | 1.80 | 1.71 | 1.61 | 1.53 | 1.44 | 1.34 | 1.28 | 1.19 | 1.12 | 1.04 | .97 | .90 |

Note: If it is found that operating conditions create a final air temperature below 90°, the output air stream may be uncomfortable and steam pressure should be increased.

Problem: Condensate will increase. How much?

Example: A unit heater rated at 100,000 btuh (2 psi/60° e.a.) is installed with 50° entering air but final air temperature is below 90°.

Factory Rating: 100,000 btuh ÷ 966.2 (latent heat at 2 psi) = 103.5 lbs. per hr.

Operating Conditions: 100,000 x 1.08 (50° e.a.) ÷ 966.2 = 111.8 lbs. per hr.

Increasing pressure to 5 psi: 100,000 x 1.13 ÷ 960.5 = 117.7 lbs. per hr.

CHAPTER 1-STEAM

STEAM CONVERSION FACTORS

| <u>MULTIPLY</u> | <u>BY</u> | <u>TO GET</u> |
|--|-----------|--|
| • Boiler Horsepower (BHP) | 34.5 | Lb. of Steam Water per hour (lb/hr) |
| • Boiler Horsepower (BHP) | 0.069 | Gallons of Water Per Minute (GPM) |
| • Sq. Feet of Equivalent Direct Radiation (EDR) | 0.000637 | Gallons of Water per Minute(GPM) |
| • Boiler Horsepower (BHP) | 33,479 | B.T.U. |
| • Boiler Horsepower (BHP) | 108 | Sq. Feet of Equivalent Direct Radiation (EDR) |
| • Lbs. of Steam Water per Hour (lb/hr) | 0.002 | Gallons of Water per Minute(GPM) |
| • Lbs. per square inch | 2.307 | Feet of Water |
| • Lbs. per square inch | 2.036 | Inch of Mercury |
| • Feet of Water (head) | 0.4335 | Lbs. per square inch |
| • Inch of Mercury | 13.6 | Inch of Water Column |
| • Gallons of Water | 8.34 | Lbs. of Water |
| • Cubic Feet of Water | 7.48 | Gallons of Water |
| • Cubic Feet per Minute | 62.43 | Lbs. of Water per Minute |
| • Cubic Feet per Minute | 448.8 | Gallons per hour |
| • Cubic Centimeters per ltr. of Oxygen | 1,400 | Parts per billion of Oxygen |
| • Lbs. of Condensate | 4 | Sq. ft. E.D.R. |

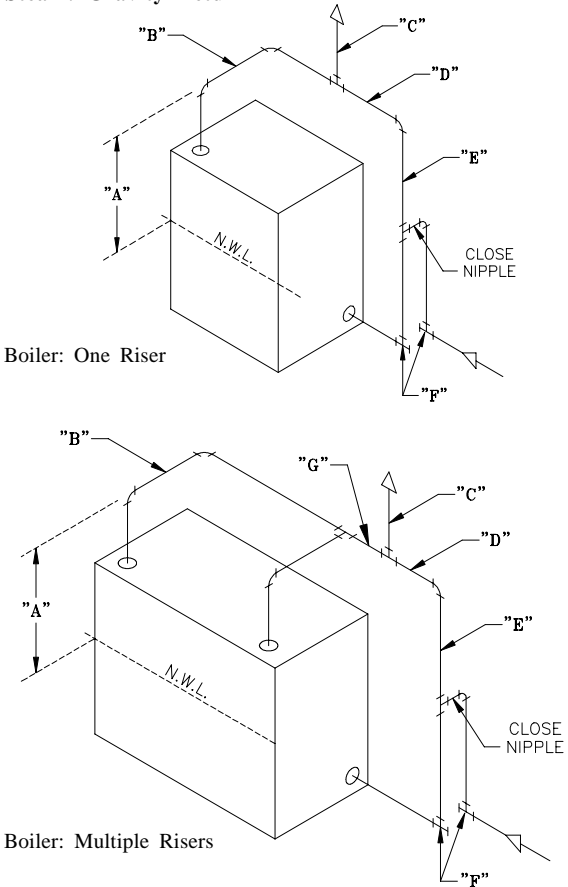
Note: Use above factors to match condensate return and boiler feed equipment with boiler size.

- Size Condensate Receivers for 1 min. net storage capacity based on return rate.
- Size Boiler Feed Receivers for system capacity (normally estimated at 10 min.)
- Size Condensate Pumps at 2 to 3 times condensate return rate.
- Size Boiler Feed Pumps at 2 times boiler evaporation rate or .14 GPM/boiler HP (continuous running boiler pumps may be sized at 1-1/2 times boiler evaporation rate or .104 GPM/boiler HP)

CHAPTER 1-STEAM

STEAM PIPING BOILER DETAIL

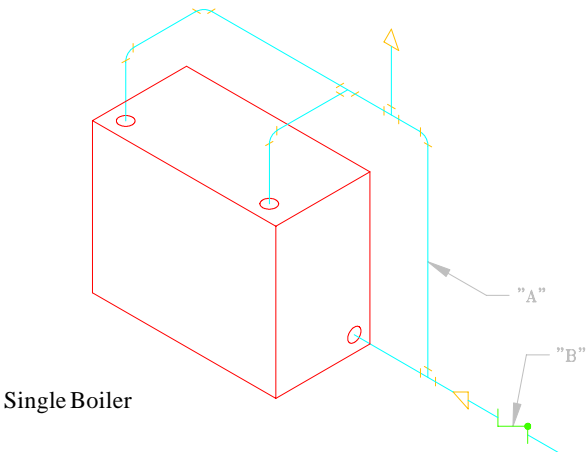
A. Steam: Gravity Return



CHAPTER 1-STEAM

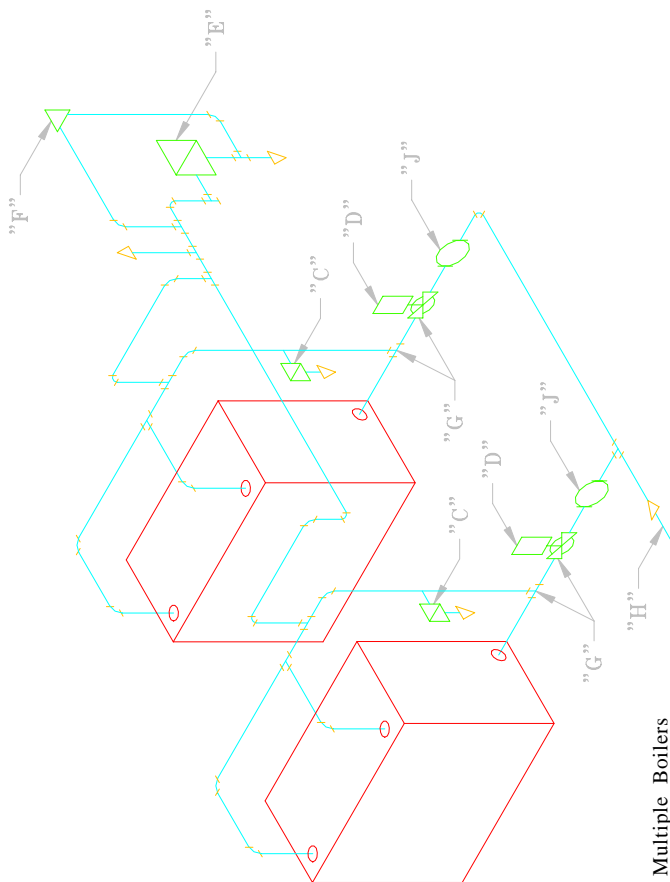
- A. 24" vertical rise.
- B. Horizontal traverse to line up header with rear return tapping.
- C. Vertical rise to existing main. Same size as header.
- D. Nipple: 6" residential; 12" commercial.
- E. Hartford Loop. Height/size, consult manufacturer.
- F. Preferable use plugged tees in lieu of 90° elbow in wet returns.
- G. Header all steam risers before moving steam to system.

B. Steam: Pump Return



Notes refer to Single & Multiple Boilers

- A. Header drip. Use in lieu of Hartford Loop when pump returns condensate. Size same as equalizer. May reduce below W.L.
- B. Install check valve. Preferably lift check in lieu of swing.
- C. Install F & T trap in header drip at N.W.L. Will maintain proper water level in unfired boilers. Return overflow to condensate receiver.
- D. Multiple boilers with common boiler feed system must have each boiler return isolated with motorized valves.
- E. Initial call for steam may create excess condensate in common drop header. F & T trap will safely guide condensate to condensate receiver.
- F. Thermostatic trap piped from header to F & T will help remove air from header.
- G. Size motorized valve by GPM flow. Increase pipe size between solenoid and boiler to reduce water velocity into boiler. See recommendations for water flow on condensate pump piping detail.
- H. See pump piping detail for condensate return to boiler.
- J. Plug valves must be installed next to each motorized valve to regulate GPM flow to each boiler from common pumping.

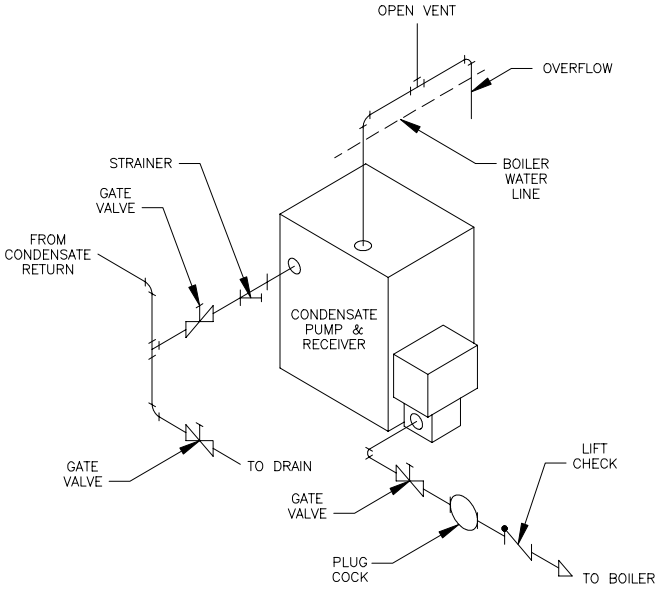


Multiple Boilers

CHAPTER 1-STEAM

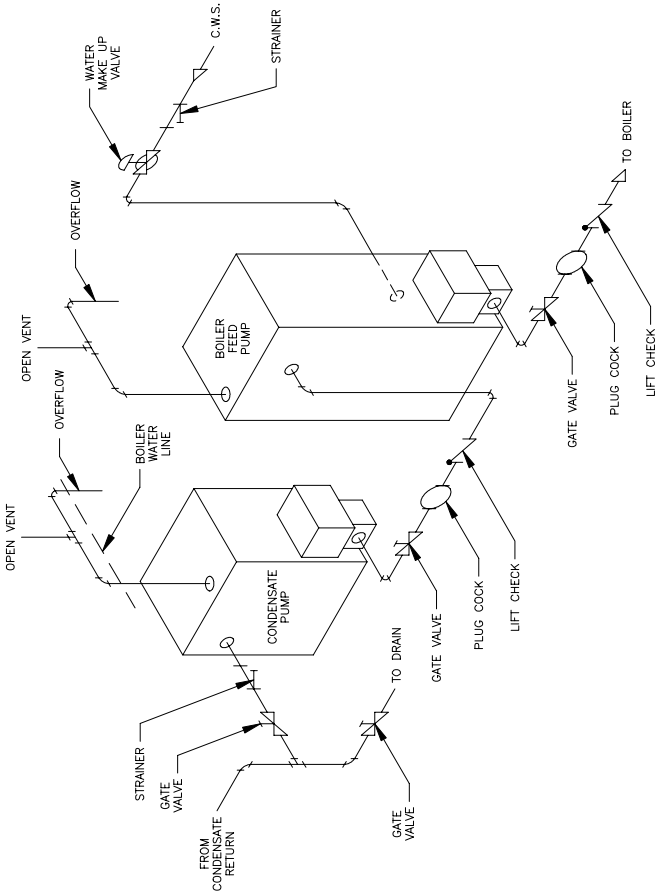
STEAM PIPING - PUMP DETAIL

A. Condensate Pump



CHAPTER 1-STEAM

B. Boiler Feed Pump and Condensate Pump



Control of Combination Units

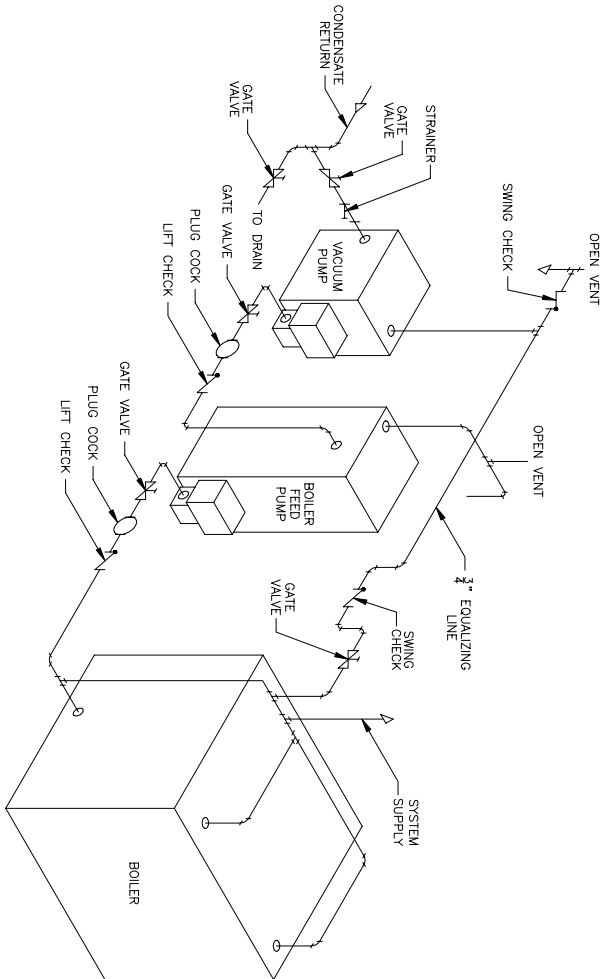
- Condensate Unit. Float mechanism in receiver operates pump. It will move water to the boiler feed receiver.
- Boiler Feed Unit. Pump Controller (#150/42A McDonnell-Miller are typical) will control the movement of water directly to the boiler. Should the water in the boiler feed receiver drop too low, an installed water make-up valve will raise the receiver to a safe level.

CHAPTER 1-STEAM

C. Boiler Feed and Vacuum Pumps

Control & Pipe (same as Boiler Feed & Condensate Pump)

Additional Piping: Equalizing line between boiler steam header and vacuum pump. Equalizing line will prevent a possible vacuum developing in steam supply that could be greater than vacuum in the return.



CHAPTER 1-STEAM

SIZING OF VACUUM PUMP

Vacuum condensate pumps are normally sized at 2 or 3 times the system condensing rate. This is the same procedure for sizing standard condensing rate. The vacuum pump must also remove air from the return system. The system size, the inches of vacuum desired, and tightness of the system piping must be considered. The following chart will help.

Air Removal Requirements

| System | Vac/In. Hg. | CFM/1,000 EDR |
|------------------------------------|-------------|---------------|
| • up to 10,000 EDR (tight) | 0-10 | .5 |
| • over 10,000 EDR (tight) | 0-10 | .3 |
| • all systems, some air in-leakage | 0-10 | 1.0 |

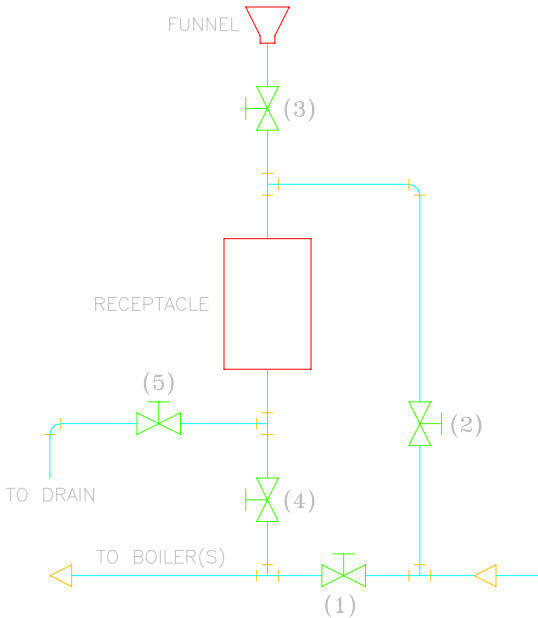
HIGHEST TEMPERATURE °F OF CONDENSATE PERMISSIBLE WITH VACUUM PUMPS

| Vacuum Inches of Mercury in Pump Receiver | Highest Permissible Temperature of Condensate °F |
|---|--|
| 15 | 179 |
| 12 | 187 |
| 10 | 192 |
| 8 | 196 |
| 6 | 201 |
| 4 | 205 |
| 2 | 209 |
| 1 | 210 |

CHAPTER1-STEAM

STEAM PIPING CHEMICAL FEED TO BOILER

Placement of Chemical Feed to Boiler



- Between boiler and boiler feed pump.
- Chemical treatment added ahead of the pump suction will increase the friction and shorten the life of the mechanical pump seals.

Procedure

- Open valve (3).
- Pour chemicals into funnel to rest in receptacle.
- Close valves (1) - (3) - (5); open valves (2) - (4). Condensate will move from the pump through the receptacle and take the chemicals to the boiler.
- When feeding is complete, close valves (2) and (4), then open valve (1).
- Residue in receptacle may be drained by opening valve (5).

CHAPTER 1-STEAM

CONVERTING A STEAM SYSTEM TO HOT WATER

GUIDELINES

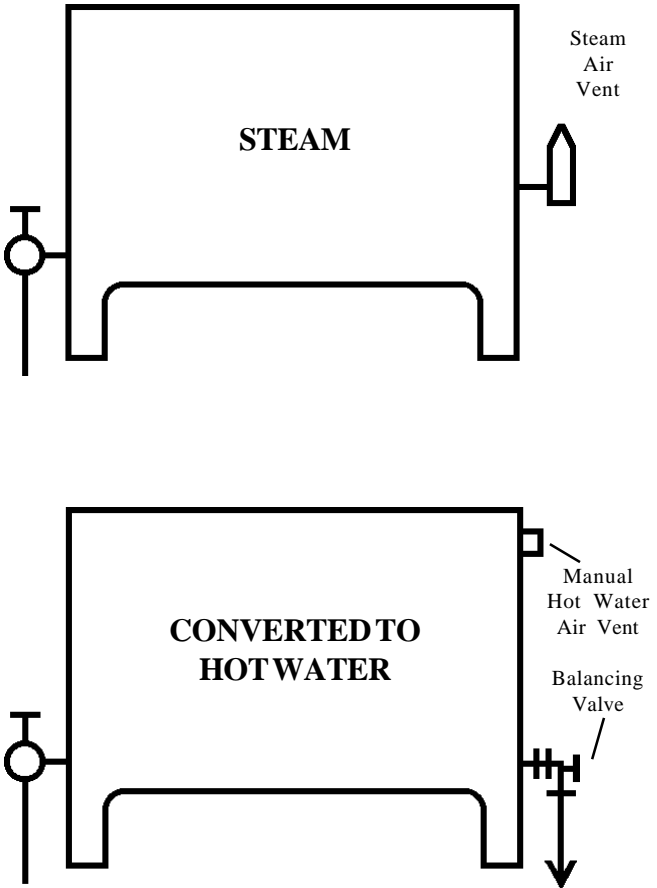
1. If the heat emitters are cast iron radiators make certain that the sections are nipped (connected) at both the top and bottom.
2. Size boiler based on a calculated heat loss not installed radiation.
3. Measure the installed radiation. Divide total E.D.R. (sq. ft.) into the calculated heat loss. The answer will indicate the number of BTU's each sq. ft. of radiation must develop at outdoor design temperature.

Refer to the chart on page 22. In the BTU column move horizontally to the number nearest your calculation. Above that number will be the maximum system temperature you need. Vertically below will be the amount of radiation (sq. ft.) that the various pipe sizes will handle.

4. **One-Pipe Steam System:** Existing steam mains and risers may be reused. However, new return risers and return main must be installed. Use the chart on page 13 to determine their size.
5. **Two-Pipe Steam System:** With the exception of the near boiler piping, all the system piping (steam mains, risers, and condensate returns) may be reused. Typically, a residential steam system that has 2" steam mains will have a minimum 1 1/4" dry return. If the heat loss is less than the 160 MBH that 1 1/4" dry return would be of sufficient size. The chart on page 13 gives the MBH capacity of pipe sizes from 1/2" to 2".
6. Regardless of the system, never reuse existing wet return piping.

CHAPTER 1-STEAM

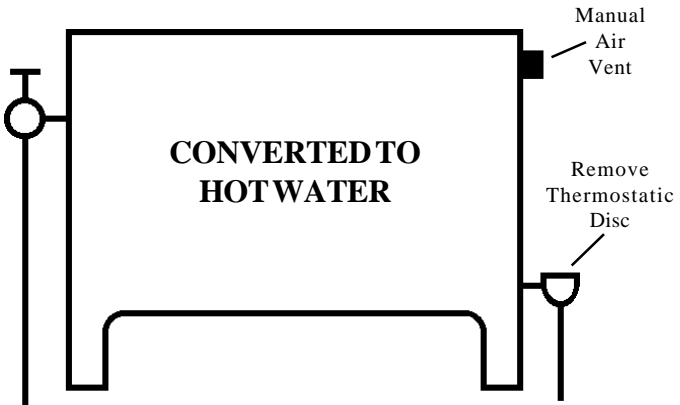
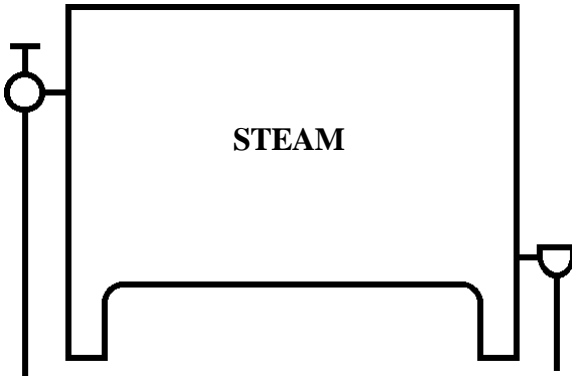
ONE PIPE STEAM (Figure 1)



NOTE: Radiator must be nipped top & bottom.

CHAPTER 1-STEAM

TWO PIPE STEAM (Figure 2)



CHAPTER 1-STEAM

| Capacity | | | | | | | | | | | | |
|--|------|------|------|------|------|------|------|------|------|------|------|-------|
| Basis: 70°F Room Temperature 20°F Temperature Drop | | | | | | | | | | | | |
| Sq. Ft. Radiation | | | | | | | | | | | | |
| Temp. | 215° | 200° | 190° | 180° | 170° | 160° | 150° | 140° | 130° | 120° | 110° | |
| BTU | 240 | 210 | 190 | 170 | 150 | 130 | 110 | 90 | 70 | 50 | 30 | |
| Pipe Size | MBH | | | | | | | | | | | |
| 1/2" | 17 | 71 | 81 | 90 | 100 | 113 | 131 | 155 | 189 | 243 | 340 | 567 |
| 3/4" | 39 | 163 | 186 | 205 | 229 | 260 | 300 | 355 | 433 | 557 | 780 | 1300 |
| 1" | 71 | 296 | 338 | 374 | 418 | 473 | 546 | 646 | 789 | 1014 | 1420 | 2367 |
| 1 1/4" | 160 | 667 | 762 | 842 | 914 | 1067 | 1231 | 1455 | 1778 | 2286 | 3200 | 5333 |
| 1 1/2" | 240 | 1000 | 1143 | 1263 | 1412 | 1600 | 1846 | 2182 | 2667 | 3429 | 4800 | 8000 |
| 2" | 450 | 1875 | 2143 | 2368 | 2647 | 3000 | 3462 | 4091 | 5000 | 6429 | 9000 | 15000 |
| <p>Use chart to determine BTU load on radiation converted from steam to H.W. Note: Heat loss of building will determine BTU load on system piping. Divide sq. ft. of installed radiation into heat loss = BTU load per sq. ft.</p> | | | | | | | | | | | | |

CHAPTER 2 - HOT WATER PIPING - RESIDENTIAL

Pipe Capacity in MBH at 500 Milinches Restriction Per Foot of Pipe

| Pipe Size | MBH | +Friction Head Feet Per 100' | G.P.M. at 20° T.D. | Velocity Flow of Water | |
|-----------|------|------------------------------|--------------------|------------------------|--------------|
| | | | | Inches Per Sec. | Ft. Per Min. |
| 1/2" | 17 | 4.2' | 1.7 | 23 | 115 |
| 3/4" | 39 | 4.2' | 3.9 | 27 | 135 |
| 1" | 71 | 4.2' | 7.1 | 34 | 170 |
| 1-1/4" | 160 | 4.2' | 16.0 | 40 | 200 |
| 1-1/2" | 240 | 4.2' | 24.0 | *45 | 225 |
| 2" | 450 | 4.2' | 45.0 | *54 | 270 |
| 2-1/2" | 750 | 4.2' | 75.0 | *62 | 310 |
| 3" | 1400 | 4.2' | 140.0 | *72 | 360 |
| 4" | 2900 | 4.2' | 290.0 | *80 | 400 |

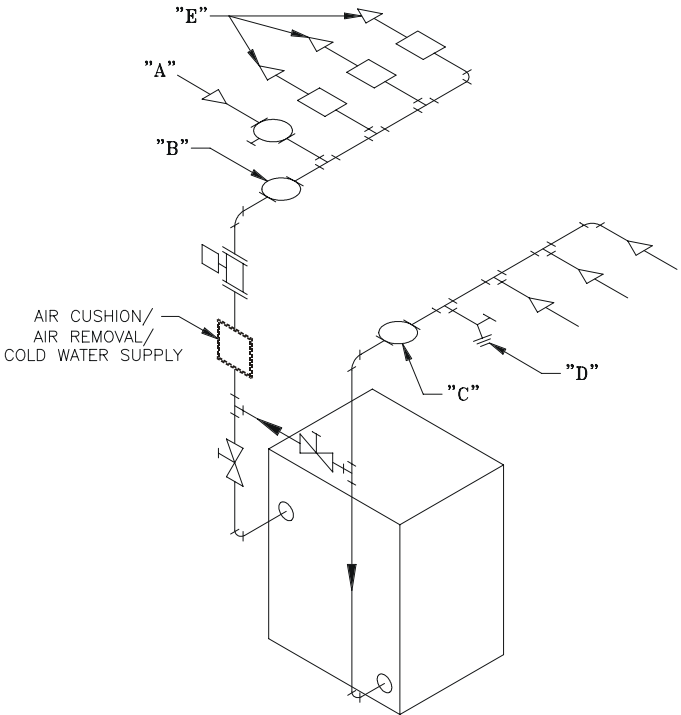
* - Maximum velocity flow

+ - In order for a pump to move the G.P.M. listed, the pump must overcome a friction head of 4.2 feet per 100' of pipe travers. (Total Equivalent Length)

Example: If one wants to carry 16 gpm in a 1-1/4" pipe through a pipe circuit of 300' (T.E.L.), the pump must overcome a friction head of 4.2' x 3 or 12.6 ft. In other words, the pump specification would be to pump 16 gpm against a 12.6 friction head.

CHAPTER 2 - HOT WATER PIPING - RESIDENTIAL

SYSTEM PURGE IN LIEU OF AIR VENTS



- (A) Separate 1/2" C.W.S. with Globe valve or by-pass P.R.V.
- (B) Ball valve: optional but preferred
- (C) Ball valve
- (D) Hose bib (for purging)
- (E) Zone valves or ball valves

CHAPTER 2 - HOT WATER PIPING - RESIDENTIAL

SYSTEM PURGE IN LIEU OF AIR VENTS

APPLICATION

Any series loop piping, especially where system high points may be concealed or venting is impractical. "System purge" is designed to remove air initially from the piping system, especially the high points.

GUIDELINES

After the initial removal of air it is important for the system to be able to handle air that will develop when the system water is heated. Installation of an air scoop or an in-line airtrol at the boiler supply will be necessary to either vent the air from the system or direct the air to a standard expansion tank.

A system purge is recommended only on Series Loop Installations. If other than non-ferrous baseboard is used, such as Base-Ray, the interconnecting pipe will purge but vents must be placed on each free standing assembly of Base-Ray.

PROCEDURE

1. Fill system with water. Isolate boiler by closing valves (B) and (C).
2. Begin purge by opening first zone or loop. Next, open valve (A) (cold water supply). Finally and immediately, open hose bib (D). Once water flows freely, close the first zone or loop and then do the same for the remaining zones or loops.

CHAPTER 2 - HOT WATER PIPING - RESIDENTIAL

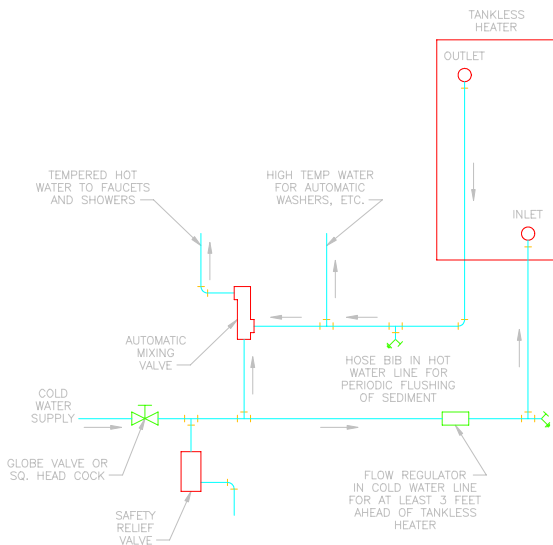
INDIRECT DOMESTIC HOT WATER PIPING

1. APPLICATION

- A. Tankless Heater: Piping below is essential for safe and reliable operation.
- B. Storage Heater: If the storage water temperature is maintained at or below 120°F, piping below is unnecessary.

If there is need to increase the storage, this can be accomplished by storing hotter water then tempering it. This will necessitate the piping below.

2. PIPING SCHEMATIC



- (1) Preferably the mixing valve should be set no higher than 120°F. Install mixing valve 15" below H.W. outlet. This will create a beneficial cooling leg.
- (2) Automatic Flow Regulator (tankless only) must match GPM rating of the tankless. If piped downstream of mixing valve, GPM flow will increase if heater water is hotter than 120°F.
- (3) Placement of hose bibs will permit periodic back flushing of heater (coil).

CHAPTER 2 - HOT WATER PIPING - RESIDENTIAL

3. INDIRECT DOMESTIC HOT WATER CONCERN

Scalding: Temperature - Minimum Time - Degree of Burn

| <u>Temp (°F)</u> | <u>1st Degree</u> | <u>2nd & 3rd Degree</u> |
|------------------|-------------------|-----------------------------|
| 111.2° | 5 hrs. | 7 hrs. |
| 116.6° | 35 min. | 45 min. |
| 118.4° | 10 min. | 14 min. |
| 122.0° | 1 min. | 5 min. |
| 131.0° | 5 sec. | 22 sec. |
| 140.0° | 2 sec. | 5 sec. |
| 149.0° | 1 sec. | 2 sec. |
| 158.0° | 1 sec. | 1 sec. |

INDIRECT SWIMMING POOL HEATING

APPLICATION

The use of a house heating boiler to indirectly heat a swimming pool is possible and even desirable. A factor of major significance would be comparable heat loads.

SIZING CONSIDERATIONS

Gallons of water to heat, temperature rise, and time allotted for temperature rise.

1. SIZING FORMULA (INITIAL RAISING OF WATER TEMPERATURE)

Gallons of Water = Pool (width x length x avg. depth) x 7.48
(gal. per cu. ft.)

Gallons of water x 8.34 x temp. rise ÷ hours to heat pool = BTUH

Example: Pool (40' x 20' x 5' avg.)

with initial pool water of 55°F to be raised to 75°F
allowing 48 hours to raise temperature

$40' \times 20' \times 5' \times 7.48 = 29,920$ gallons of water

$29,920 \times 8.34 \times 20 \div 48 = 103,972$ BTUH (I=B=R Net Ratings).

2. HEAT LOSS FROM POOL SURFACE* (MAINTAINING WATER TEMPERATURE)

| Temperature Difference °F | 10° | 15° | 20° | 25° | 30° |
|---------------------------|-----|-----|-----|-----|-----|
| BTUH/per Sq. Ft. | 105 | 158 | 210 | 263 | 368 |

Notes: • Assumed wind velocity: 3.5 mph

Wind velocity of 5 mph multiply BTUH by 1.25

Wind velocity of 10 mph multiply BTUH by 2.00

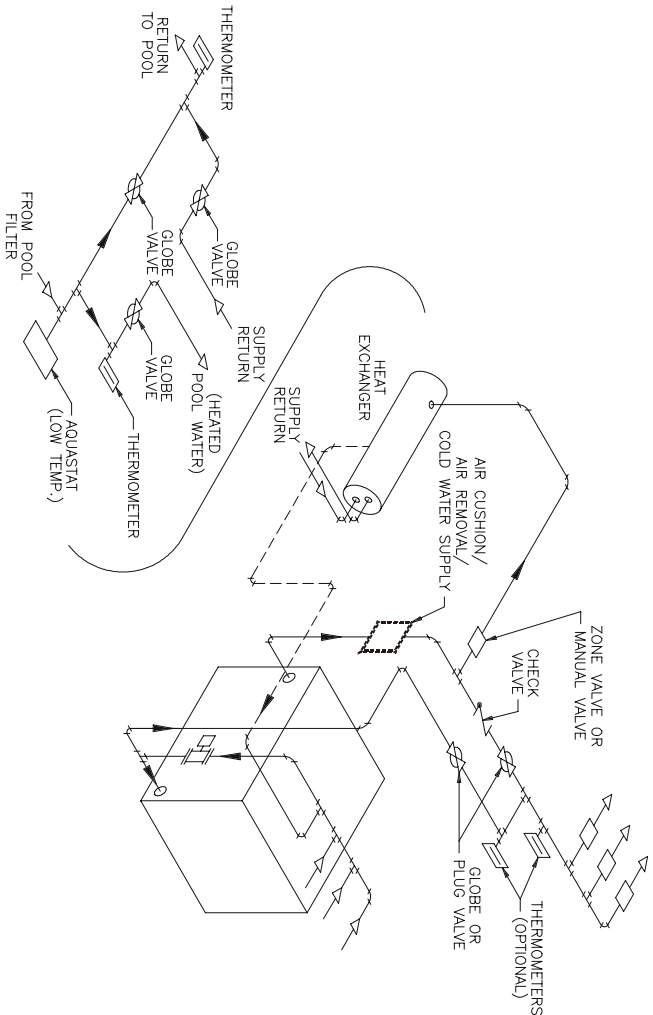
• Temperature Difference: Ambient air and desired water temp.

* Maintaining pool temperature when outside air is 20° to 30°F lower than pool water may require a larger boiler.

CHAPTER 2 - HOT WATER PIPING - RESIDENTIAL

PIPING: INDIRECT SWIMMING POOL HEATING

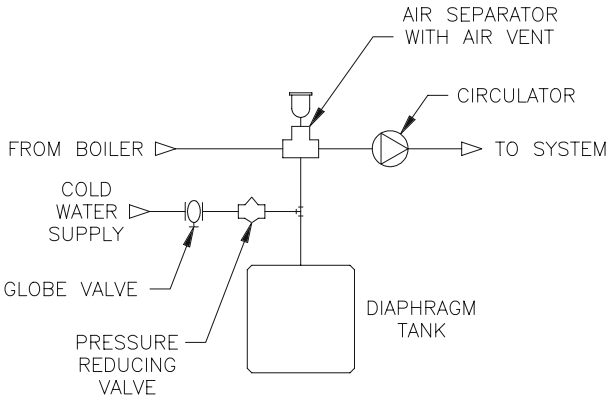
Note: By-pass enables one to regulate flow through heat exchanger and also provide a manual disconnect from heating system.



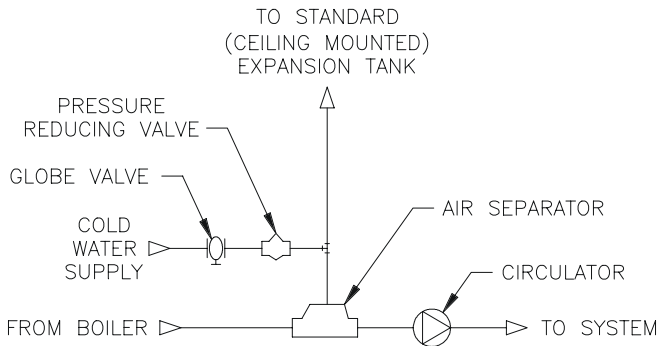
CHAPTER 2 - HOT WATER PIPING - RESIDENTIAL

PIPING: HOT WATER SYSTEM AIR CUSHION AND AIR REMOVAL OPTIONS

AIR REMOVAL SYSTEM

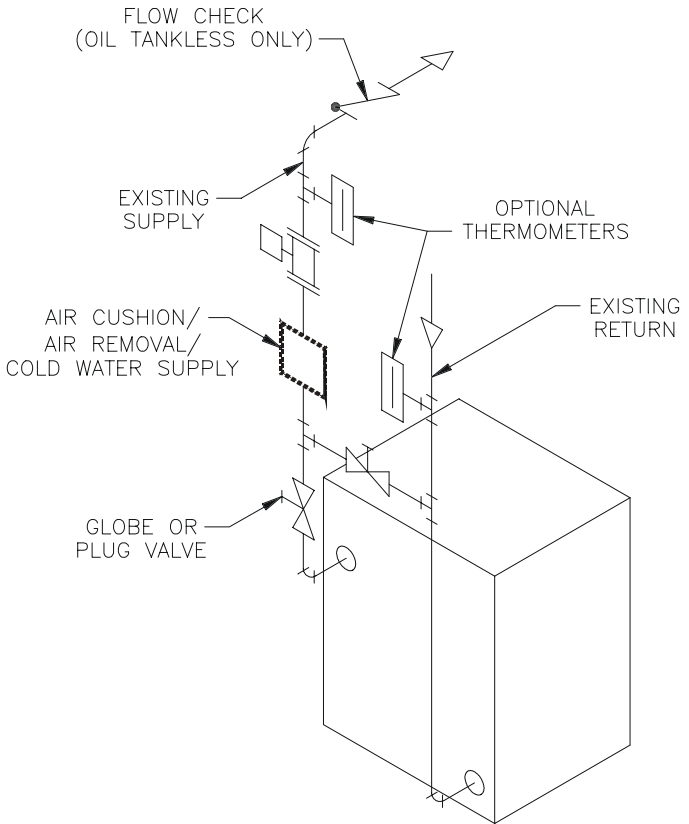


AIR CONTROL SYSTEM



CHAPTER 2 - HOT WATER PIPING - RESIDENTIAL

1. BOILER BY-PASS - ONE ZONE



CHAPTER 2 - HOT WATER PIPING - RESIDENTIAL

1. BOILER BY-PASS - ONE ZONE (CONT.)

PURPOSE FOR BY-PASS

GAS BOILER CONNECTED TO LARGE WATER CONTENT RADIATION

- Flue gas condensation will occur when the "burner" is on and the water in the boiler is less than 130°F. Over 50% of the heating season the radiators will satisfy the need for heat with less than 130°F. By-pass piping will permit the boiler to carry a higher temperature than the radiation. Regulation comes from the globe or plug valves. Thermometers, positioned above, will facilitate a proper by-pass. The supply water temperature need only overcome the temperature drop of the system. Even with water moving as slow as 2' to 1' per second in the oversized gravity piping, within a few minutes the thermometer mounted in the by-pass (return water) will begin to register temperature rise then serious balancing can start.

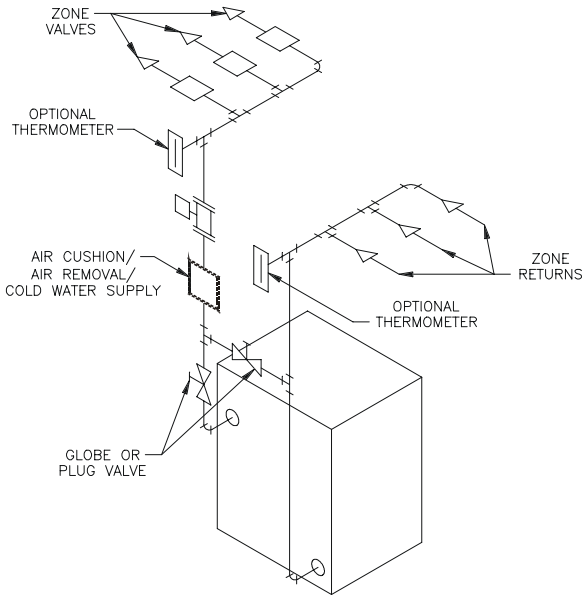
OIL BOILER WITH TANKLESS CONNECTED TO LARGE WATER CONTENT RADIATION

- Flue gas condensation can occur in oil boilers too. However, there is greater concern regarding system operation. Without a boiler by-pass, water returning from one radiator will drastically drop the boiler temperature and cause the circulator to stop. Heat leaves the boiler in "spurts" and all the "hot 'spurts'" end up in the same radiator and the heating system is quickly out of balance. By-pass piping properly set will dramatically minimize this problem.

CHAPTER 2 - HOT WATER PIPING - RESIDENTIAL

2. BOILER BY-PASS - MULTI-ZONE HEATING

PURPOSE OF BY-PASS (GAS AND OIL)



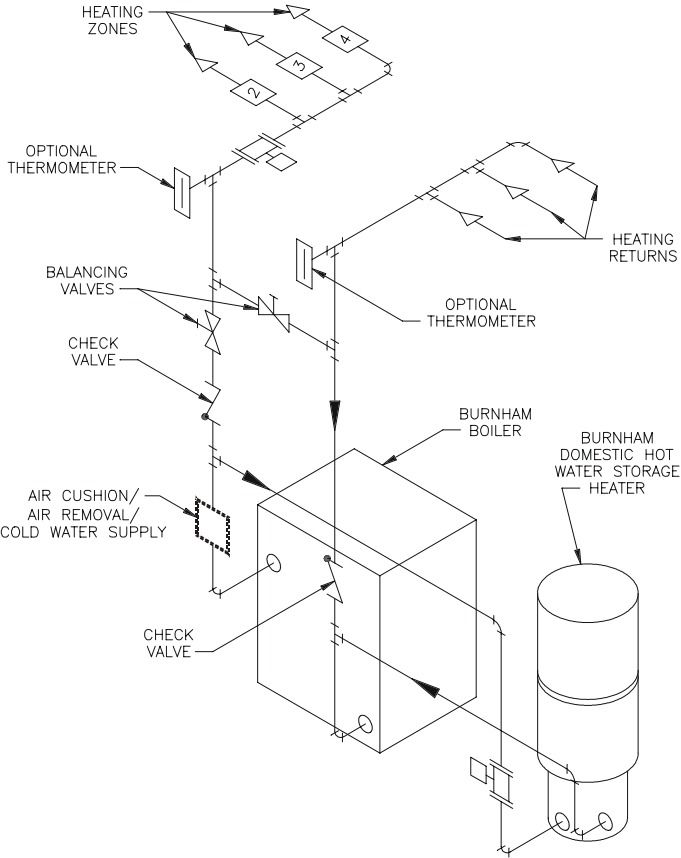
- In a multi-zone (cold start) system excess temperature develops. When one zone calls for heat it cannot dissipate the heat the boiler (sized for all zones) creates. When the zone is satisfied the hot water trapped in the boiler stays hot while the hot water trapped in the non-ferrous baseboard cools quickly. Before the next call for heat the temperature difference may be as high as 100°F. The system will be plagued with expansion noises unless boiler by-pass piping is installed. If, for example, the temperature drop in each of the zones is approximately 10°F then the balancing valves need only show on the thermometers a 15°F variance between supply and return piping, and expansion noises will be eliminated.

Beyond Boiler By-Pass

- Two-stage thermostats or an indoor-outdoor reset controller will minimize the temperature rise in the boiler but even with these controls, boiler by-pass piping will help.

CHAPTER 2 - HOT WATER PIPING - RESIDENTIAL

3. BOILER BY-PASS - MULTI-ZONE HEATING AND INDIRECT D.H.W. ZONE



3. BOILER BY-PASS - MULTI-ZONE HEATING AND INDIRECT D.H.W. ZONE (CONT.)

PURPOSE OF BY-PASS (GAS AND OIL)

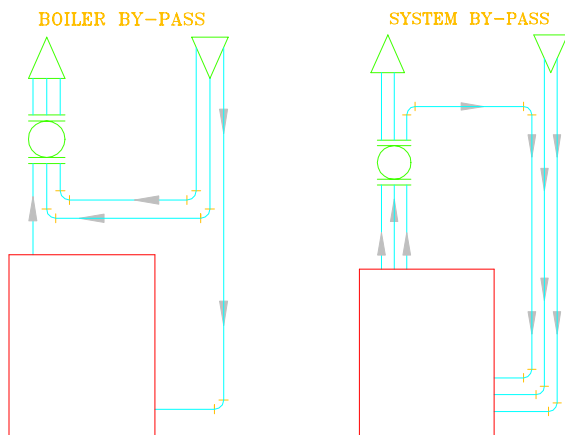
- Serves only the heating zones. Though, indirectly, the hotter boiler water it creates will help the indirect domestic hot water zone. Boiler by-pass piping, per se, is helpful to the heating zones yet detrimental to the DHW zone. However, with properly positioned check valve by-pass piping can serve both operations.

EXAMPLES

1. Heating zone (2, 3 or 4) calls for heat: Full flow of the circulator will move from the radiation to point (A). If the balancing valves (globe or plug) are adjusted to slightly overcome the temperature drop of the zone(s), typically 30% of the flow will move into the boiler at point (B) and 70% will by-pass the boiler at (C). At point (D) the full, but blended, flow will move to the radiation.
2. Indirect DHW zone (1) calls for heat: Full flow of the circulator will move through the boiler at (B). No water will move through the by-pass at (C) because of the check valve at point (E). In other words, all the heat in the boiler will be dedicated to satisfying the DHW needs.
3. Heating zone and indirect DHW zone call for heat simultaneously. Through the use of a special relay the DHW zone could be given priority. This means the heating zone is put on hold until the DHW zone is satisfied. One option would be to let the piping handle the situation. Remember, water flow takes the course of least restriction and there is considerably less restriction in the DHW zone piping.

4. BOILER BY-PASS VS SYSTEM BY-PASS

Several variations of by-pass piping exist for different reasons. Because the piping is near the boiler it is commonly categorized as “boiler by-pass” even though some may be a “system by-pass”. The difference is illustrated below.

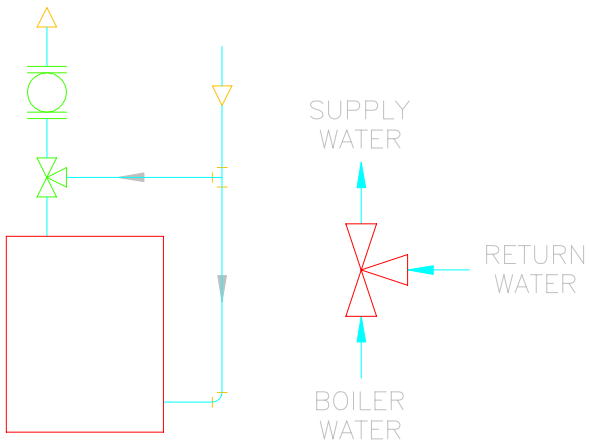


1. Drawing illustrates how water by-passes the boiler.
2. Total circulator's capacity is dedicated to the system

1. Drawing illustrates how water by-passes the system.
2. Some of the circulator's capacity is used to recirculate boiler water.

CHAPTER 2 - HOT WATER PIPING - RESIDENTIAL

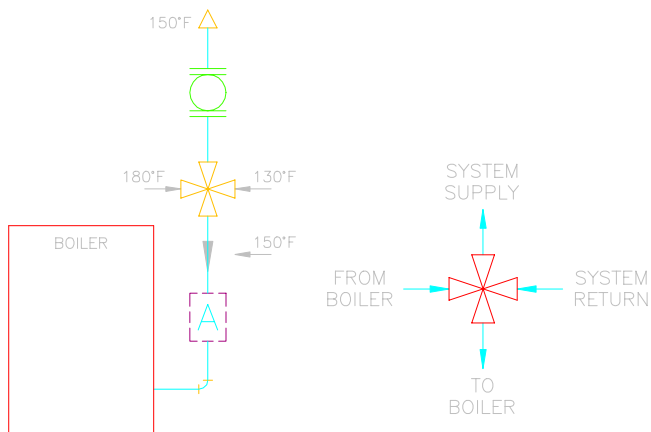
5. BOILER BY-PASS - THREE-WAY MIXING VALVES



NOTES:

- Restricting boiler water flow will increase return water flow and vice versa. Supply water flow will always be boiler water plus return water.
- Mixing may be accomplished manually or with an actuating motor.
- Three-way valve application is a boiler by-pass.

6. SYSTEM BY-PASS - FOUR-WAY MIXING VALVE



1. With application of four-way valve the preferred location of the circulator is on the "system supply" side.
2. Mixing may be accomplished manually or with an actuating motor.
3. Four-way valve application is a system by-pass.
4. Four-way valve application may require a second circulator, one dedicated to system flow and the other to boiler flow. The second circulator should be installed at point (A) pumping into the boiler
5. **Concern:** Should four-way valve be fully opened to the system, both circulators will be pumping in series.

CHAPTER 3-DOMESTIC HOT WATER

DOMESTIC HOT WATER SIZING REQUIREMENTS

A. RESIDENTIAL: Sizing residential water heaters vary with manufacturers. HUD and FHA have established minimum permissible water heater sizes as shown in the following table.

HUD--FHA Minimum Water Heater Capacities for One- and Two-Family Living Units

| Number of Baths | 1 to 1.5 | | | 2 to 2.5 | | | | 3 to 3.5 | | | |
|--|----------|------|------|----------|------|-------|------|----------|------|------|------|
| Number of Bedrooms | 1 | 2 | 3 | 2 | 3 | 4 | 5 | 3 | 4 | 5 | 6 |
| GAS (a) | | | | | | | | | | | |
| Storage, gal | 20 | 30 | 30 | 30 | 40 | 40 | 50 | 40 | 50 | 50 | 50 |
| 1000 Btu/h input | 27 | 36 | 36 | 36 | 36 | 38 | 47 | 38 | 38 | 47 | 50 |
| 1-h draw, gal | 43 | 60 | 60 | 60 | 70 | 72 | 90 | 72 | 82 | 90 | 92 |
| Recovery, gph | 23 | 30 | 30 | 30 | 30 | 32 | 40 | 32 | 32 | 40 | 42 |
| TANK TYPE INDIRECT (b,c) | | | | | | | | | | | |
| I-W-H rated draw, gal in 3-h, 100°F rise | | 40 | 40 | | 66 | 66 | 66 | 66 | 66 | 66 | 66 |
| Manufacturer-rated draw, gal in 3-h, 100°F (55.6°C) rise | | 49 | 49 | | 75 | 75d | 75 | 75 | 75 | 75 | 75 |
| Tank capacity, gal | | 66 | 66 | | 66 | 66d | 82 | 66 | 82 | 82 | 82 |
| TANKLESS TYPE INDIRECT (c,e) | | | | | | | | | | | |
| I-W-H rated, gpm 100°F rise | | 2.75 | 2.75 | | 3.25 | 3.25d | 3.75 | 3.25 | 3.75 | 3.75 | 3.75 |

- a. Storage capacity, input, and recovery requirements indicated in the table are typical and may vary with each individual manufacturer. Any combination of these requirements to produce the stated 1-h draw will be satisfactory.
- b. Boiler-connected water heater capacities [180°F boiler water, internal or external connection].
- c. Heater capacities and inputs are minimum allowable. A.G.A. recovery ratings for gas heaters, and IBR ratings for steam and hot water heaters.
- d. Also for 1 to 1.5 baths and 4 B.R. for indirect water heaters.
- e. Boiler-connected heater capacities [200°F boiler water, internal or external connection].

Reference: ASHRAE/Systems Handbook (Service Water Heating)

CHAPTER 3-DOMESTIC HOT WATER

B. COMMERCIAL: Commercial buildings have different domestic hot water needs. Building Type will be the major variable. The two charts that follow analyze the demand based either on Fixture or on Occupancy.

Note: Both charts presume the use of storage type heaters.

Chart #1

[Gallons of water per hour per fixture, calculated at a final temperature of 140°F]

| | Apt. House | Club | Gym | Hospital | Hotel | Ind. Plant | Office Bldg. | Priv. Res. | School | YMC |
|------------------------------|------------|-------|------|----------|-------|------------|--------------|------------|--------|--------|
| 1. Basins, private lavatory | 2 | 2 | 2 | 2 | 2 | 2 | 2 | 2 | 2 | 2 |
| 2. Basins, public lavatory | 4 | 6 | 8 | 6 | 8 | 12 | 6 | -- | 15 | 8 |
| 3. Bathtubs | 20 | 20 | 30 | 20 | 20 | -- | -- | 20 | -- | 30 |
| 4. Dishwashers | 15 | 50-10 | -- | 50-150 | 50-20 | 20-10 | -- | 15 | 20-100 | 20-100 |
| 5. Foot basins | 3 | 3 | 12 | 3 | 3 | 12 | -- | 3 | 3 | 12 |
| 6. Kitchen Sink | 10 | 20 | -- | 20 | 30 | 20 | 20 | 20 | 20 | 20 |
| 7. Laundry, stationary tubs | 20 | 28 | -- | 28 | 28 | -- | -- | 20 | -- | 28 |
| 8. Pantry sink | 5 | 10 | -- | 10 | 10 | -- | 10 | 5 | 10 | 10 |
| 9. Showers | 30 | 150 | 225 | 75 | 75 | 225 | 30 | 30 | 225 | 225 |
| 10. Slop sink | 20 | 20 | -- | 20 | 30 | 20 | 20 | 15 | 20 | 20 |
| 11. Hydrotherapeutic showers | | | | 400 | | | | | | |
| 12. Circular wash sinks | | | | 20 | 20 | 30 | 20 | | 30 | |
| 13. Semicircular wash sinks | | | | 10 | 10 | 15 | 10 | 15 | | |
| 14. DEMAND FACTOR | 0.30 | 0.30 | 0.40 | 0.25 | 0.25 | 0.40 | 0.30 | 0.30 | 0.40 | 0.40 |
| 15. STORAGE CAPACITY FACTOR | 1.25 | 0.90 | 1.00 | 0.60 | 0.80 | 1.00 | 2.00 | 0.70 | 1.00 | 1.00 |

Notes:

A. #1 through #13 - Possible Maximum Demand.

B. #14 (Demand Factor) - Probable Maximum Demand

C. #15 Ratio of Storage Tank Capacity to Probable Max. Demand Per Hr.

Example:

50 Unit Apartment Building

| | | |
|-----------------------|---|----------------|
| 50 lavatories x 2 | = | 100 GPH |
| 50 showers x 30 | = | 1500 GPH |
| 50 kitchen sinks x 10 | = | 500 GPH |
| 10 laundry tubs x 20 | = | <u>200 GPH</u> |

A) Possible Maximum Demand = 2300 GPH
 Demand Factor (#14) x .30

B) Probable Maximum Demand = 690 GPH
 Storage Capacity Factor (#15) 1.25

C) Storage Tank Size = 863 GAL.

CHAPTER 3-DOMESTIC HOT WATER

CHART #2

This chart may be used as a cross check to Chart #1. The Hot Water Demand listed represents the actual metering of 129 buildings. The number of each building type sampled is listed at the extreme left side of chart.

| Number | Type of Building | Maximum Hour | Maximum Day | Average Day |
|--------|---|------------------------|-------------------------|---------------------------|
| 8 | Men's dormitories | 3.8 gal/student | 22.0 gal/student | 13.1 gal/student |
| 8 | Women's dormitories | 5.0 gal/student | 26.5 gal/student | 12.3 gal/student |
| 15 | Motels: no. of units (a) | | | |
| | 20 or less, | 6.0 gal/unit | 35.0 gal/unit | 20.0 gal/unit |
| | 60 | 5.0 gal/unit | 25.0 gal/unit | 14.0 gal/unit |
| | 100 or more | 4.0 gal/unit | 15.0 gal/unit | 10.0 gal/unit |
| 13 | Nursing homes | 4.5 gal/bed | 30.0 gal/bed | 18.4 gal/bed |
| 6 | Office buildings | 0.4 gal/person | 2.0 gal/person | 1.0 gal/person |
| 25 | Food establishments: | | | |
| | Type A--full meal restaurants and cafeterias | 1.5 gal/max meals/h | 11.0 gal/max meals/h | 2.4 gal/avg meals/day* |
| | Type B--drive-ins, grilles, luncheonettes, sandwich and snack shops | 0.7 gal/max meals/h | 6.0 gal/max meals/h | 0.7 gal/avg meals/day* |
| 26 | Apartment houses: no. of apartments | | | |
| | 20 or less | 12.0 gal/apt. | 80.0 gal/apt. | 42.0 gal/apt. |
| | 50 | 10.0 gal/apt. | 73.0 gal/apt. | 40.0 gal/apt. |
| | 75 | 8.5 gal/apt. | 66.0 gal/apt. | 38.0 gal/apt. |
| | 100 | 7.0 gal/apt. | 60.0 gal/apt. | 37.0 gal/apt. |
| | 200 or more | 5.0 gal/apt. | 50.0 gal/apt. | 35.0 gal/apt. |
| 14 | Elementary schools | 0.6 gal/student | 1.5 gal/student | 0.6 gal/student* |
| 14 | Junior and senior high schools | 1.0 gal/student | 3.6 gal/student | 1.8 gal/student* |

*Per day of operation.

(a) Interpolate for intermediate values.

Reference: ASHRAE/Systems Handbook (Service Water Heating)

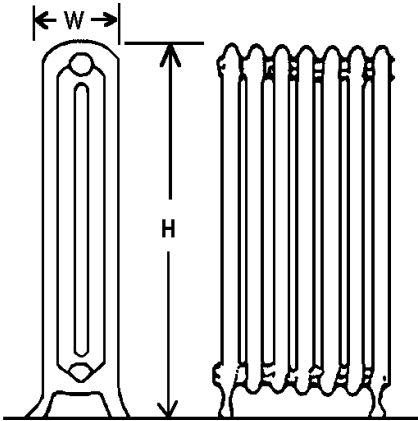
CH.4-INSTALLED RADIATION: DETERMINING HEAT LOAD

SIZING OBSOLETE RADIATION - CAST IRON RADIATORS

The output of a radiator is measured in square feet of radiation.
To determine the number of square feet of radiation in a radiator:

1. Measure the height of the radiator.
2. Count the number of columns in a section.
3. Count the number of sections.
4. Multiply the total number of sections by the number of square feet per section as shown in the following tables:

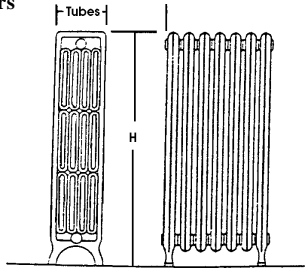
Column Type Radiators



| Sq. Ft. Radiation Per Section | | | | | |
|-------------------------------|------------------------|------------------------|----------------------|--------------------------|-----------------|
| Height Inches | 4-1/2" W One Column | 7-1/2" W Two Column | 9" W Three Column | 11-1/2" W Four Column | 13" W Window |
| 13 | — | — | — | — | 3 |
| 16 | — | — | — | — | 3-3/4 |
| 18 | — | — | 2-1/4 | 3 | 4-1/4 |
| 20 | 1-1/2 | 2 | — | — | 5 |
| 22 | — | — | 3 | 4 | — |
| 23 | 1-2/3 | 2-1/3 | — | — | — |
| 26 | 2 | 2-2/3 | 3-3/4 | 5 | — |
| 32 | 2-1/2 | 3-1/3 | 4-1/2 | 6-1/2 | — |
| 38 | 3 | 4 | 5 | 8 | — |
| 45 | — | 5 | 6 | 10 | — |

CH.4-INSTALLED RADIATION: DETERMINING HEAT LOAD

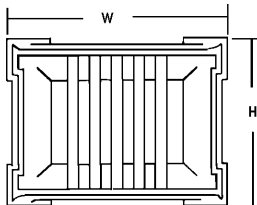
Tube Type Radiators



| Sq. Ft. Radiation Per Section | | | | | |
|-------------------------------|---------------|--------------|------------------|------------------------|--------------------|
| Height Inches | 5" Three Tube | 7" Four Tube | 8-3/4" Five Tube | 9-3/4" Window Six Tube | 12-1/2" Seven Tube |
| 14 | — | — | — | — | 2-1/2 |
| 17 | — | — | — | — | 3 |
| 20 | 1-3/4 | 2-1/4 | 2-2/3 | 3 | 3-2/3 |
| 23 | 2 | 2-1/2 | 3 | 3-1/2 | — |
| 26 | 2-1/3 | 2-3/4 | 3-1/2 | 4 | 4-3/4 |
| 32 | 3 | 3-1/2 | 4-1/3 | 5 | — |
| 38 | 3-1/2 | 4-1/4 | 5 | 6 | — |

Wall Type Radiators

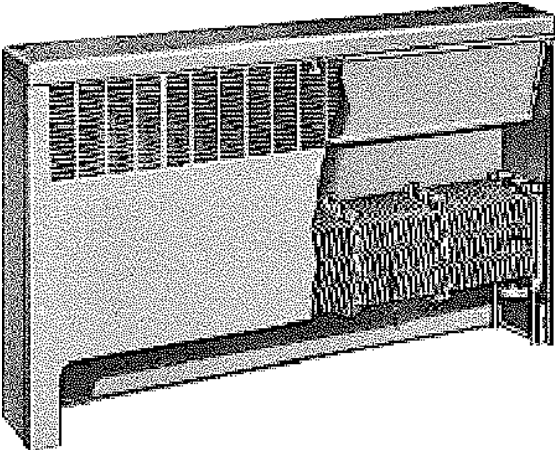
Wall radiators are measured by their height, length and thickness. The following table shows the number of square feet of heating surface per section.



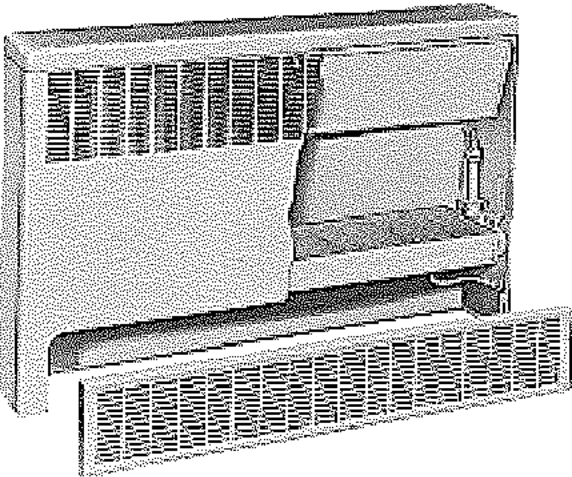
| Sq. Ft. Radiation Per Wall Radiator Section Heating | | | | |
|---|------------|---------------------|---------------|-------------------|
| Type of Section | Height In. | Length or Width In. | Thickness In. | Sq. Ft. Radiation |
| 5-A | 13-5/16 | 16-5/8 | 2-7/8 | 5 |
| 7-A | 13-5/16 | 21-7/8 | 2-7/8 | 7 |
| 7-B | 21-7/8 | 13-3/16 | 3-1/16 | 7 |
| 9-A | 18-5/16 | 29-1/16 | 2-7/8 | 9 |
| 9-B | 29-1/16 | 18-5/16 | 3-11/16 | 9 |

CONVECTORS

Cast Iron - Ratings: Page 43



Copper - Ratings: Page 44



CH.4-INSTALLED RADIATION: DETERMINING HEAT LOAD

SIZING: CONVECTORS - CAST IRON RATINGS - SQUARE FEET OF RADIATION

| FRONT OUTLET AND SLOPING OUTLET TYPE UNITS Cabinet Height - Floor Types | | 18 | 20 | 24 | 26 | 32 | 38 |
|--|-----------------------------|------|------|------|------|-------|------|
| Cabinet Depth Inches | Cabinet Length Inches | | | | | | |
| 4½" | 18½ | 8.4 | 9.1 | 10.5 | 11.0 | 11.8 | 12.3 |
| | 23½ | 10.9 | 11.8 | 13.5 | 14.2 | 15.2 | 15.9 |
| | 28½ | 13.3 | 14.4 | 16.5 | 17.4 | 18.6 | 19.4 |
| | 33½ | 15.8 | 17.1 | 19.7 | 20.6 | 22.1 | 23.0 |
| | 38½ | 18.2 | 19.7 | 22.7 | 23.8 | 25.5 | 26.5 |
| | 43½ | 20.6 | 22.3 | 25.7 | 26.9 | 28.9 | 30.1 |
| | 48½ | 23.1 | 25.0 | 28.7 | 30.1 | 32.3 | 33.6 |
| | 53½ | 25.5 | 27.6 | 31.8 | 33.3 | 35.7 | 37.2 |
| 58½ | 28.0 | 30.3 | 34.8 | 36.5 | 39.1 | 40.7 | |
| 63½ | 30.5 | 33.0 | 37.9 | 39.7 | 42.5 | 44.3 | |
| 6½" | 18½ | 12.3 | 13.5 | 15.4 | 16.2 | 17.5 | 18.2 |
| | 23½ | 15.9 | 17.4 | 19.9 | 20.9 | 22.6 | 23.5 |
| | 28½ | 19.5 | 21.3 | 24.4 | 25.6 | 27.7 | 28.8 |
| | 33½ | 23.1 | 25.2 | 28.9 | 30.4 | 32.9 | 34.1 |
| | 38½ | 26.7 | 29.2 | 33.4 | 35.1 | 38.0 | 39.4 |
| | 43½ | 30.3 | 33.1 | 37.9 | 39.8 | 43.1 | 44.7 |
| | 48½ | 33.9 | 37.0 | 42.4 | 44.5 | 48.1 | 50.0 |
| | 53½ | 37.5 | 40.9 | 46.8 | 49.2 | 53.3 | 55.3 |
| 58½ | 41.1 | 44.8 | 51.3 | 53.9 | 58.4 | 60.6 | |
| 63½ | 44.7 | 48.7 | 55.8 | 58.7 | 63.5 | 65.9 | |
| 8½" | 18½ | ---- | 17.1 | 19.4 | 20.4 | 22.5 | 23.7 |
| | 23½ | ---- | 22.2 | 25.0 | 26.4 | 29.1 | 30.6 |
| | 28½ | ---- | 27.2 | 30.7 | 32.4 | 35.7 | 37.5 |
| | 33½ | ---- | 32.2 | 36.4 | 38.4 | 42.3 | 44.5 |
| | 38½ | ---- | 37.2 | 42.1 | 44.3 | 48.9 | 51.4 |
| | 43½ | ---- | 42.3 | 47.8 | 50.3 | 55.5 | 58.4 |
| | 48½ | ---- | 47.3 | 53.5 | 56.3 | 62.0 | 65.3 |
| | 53½ | ---- | 52.3 | 59.2 | 62.3 | 68.6 | 72.3 |
| 58½ | ---- | 57.3 | 64.9 | 68.3 | 75.2 | 79.2 | |
| 63½ | ---- | 62.3 | 70.6 | 74.3 | 81.8 | 86.1 | |
| 10½" | 18½ | ---- | 20.6 | 23.4 | 24.6 | 27.3 | 28.8 |
| | 23½ | ---- | 26.7 | 30.3 | 31.8 | 35.3 | 37.2 |
| | 28½ | ---- | 32.8 | 37.2 | 39.1 | 43.3 | 45.7 |
| | 33½ | ---- | 38.9 | 44.2 | 46.3 | 51.4 | 54.2 |
| | 38½ | ---- | 45.0 | 51.1 | 53.6 | 59.5 | 62.7 |
| | 43½ | ---- | 51.1 | 58.0 | 60.8 | 67.5 | 71.2 |
| | 48½ | ---- | 57.2 | 64.9 | 68.1 | 75.6 | 79.6 |
| | 53½ | ---- | 63.3 | 71.8 | 75.4 | 83.6 | 88.1 |
| 58½ | ---- | 69.4 | 78.7 | 82.6 | 91.6 | 96.6 | |
| 63½ | ---- | 75.5 | 85.6 | 89.8 | 99.7 | 105.1 | |

CH.4-INSTALLED RADIATION: DETERMINING HEAT LOAD

SIZING: CONVECTORS - COPPER FRONT OUTLET TYPE UNITS RATINGS - SQUARE FEET OF RADIATION

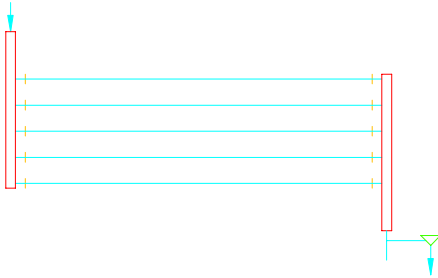
| FRONT OUTLET AND SLOPING OUTLET TYPE UNITS Cabinet Height - Floor Types | | 18 | 20 | 24 | 26 | 32 | 38 |
|--|-----------------------|------|------|------|------|------|-------|
| Cabinet Depth Inches | Cabinet Length Inches | | | | | | |
| 4-1/4" | 20½ | 10.4 | 11.3 | 13.1 | 13.4 | 14.0 | 14.6 |
| | 24½ | 12.8 | 13.9 | 16.1 | 16.4 | 17.2 | 18.0 |
| | 28½ | 15.2 | 16.5 | 19.2 | 19.5 | 20.4 | 21.3 |
| | 32½ | 17.6 | 19.1 | 22.2 | 22.5 | 23.6 | 24.6 |
| | 36½ | 19.7 | 21.7 | 25.2 | 25.6 | 26.8 | 28.0 |
| | 40½ | 22.3 | 24.3 | 28.2 | 28.6 | 30.0 | 31.3 |
| | 44½ | 24.7 | 26.9 | 31.2 | 31.7 | 33.2 | 34.7 |
| | 48½ | 27.1 | 29.5 | 34.2 | 34.7 | 36.4 | 38.0 |
| 6-1/4" | 56½ | 31.9 | 34.7 | 40.2 | 40.8 | 42.8 | 44.7 |
| | 64½ | 36.6 | 39.9 | 46.2 | 46.9 | 49.2 | 51.7 |
| | 20½ | 15.3 | 16.3 | 18.4 | 18.8 | 19.6 | 20.6 |
| | 24½ | 18.8 | 20.1 | 22.6 | 23.1 | 24.1 | 25.3 |
| | 28½ | 22.3 | 23.8 | 26.9 | 27.4 | 28.6 | 30.1 |
| | 32½ | 25.8 | 27.6 | 31.1 | 31.7 | 33.2 | 34.8 |
| | 36½ | 29.3 | 31.3 | 35.4 | 36.0 | 37.7 | 39.6 |
| | 40½ | 32.8 | 35.1 | 39.6 | 40.3 | 42.2 | 44.3 |
| 8-1/4" | 44½ | 36.3 | 38.9 | 43.8 | 44.6 | 46.7 | 49.0 |
| | 48½ | 39.8 | 42.6 | 48.1 | 48.9 | 51.2 | 53.8 |
| | 56½ | 46.8 | 50.1 | 56.6 | 57.5 | 60.3 | 63.3 |
| | 64½ | 53.9 | 57.7 | 65.0 | 66.2 | 69.3 | 72.7 |
| | 20½ | 18.7 | 20.0 | 22.5 | 23.0 | 24.4 | 25.8 |
| | 24½ | 22.9 | 24.5 | 27.7 | 28.2 | 30.0 | 31.7 |
| | 28½ | 27.2 | 29.1 | 32.8 | 33.5 | 35.6 | 37.7 |
| | 32½ | 31.5 | 33.7 | 38.0 | 38.8 | 41.2 | 43.6 |
| 10-1/4" | 36½ | 35.8 | 38.3 | 43.2 | 44.1 | 46.8 | 49.5 |
| | 40½ | 40.1 | 42.9 | 48.3 | 49.4 | 52.4 | 55.5 |
| | 44½ | 44.3 | 47.4 | 53.5 | 54.6 | 58.0 | 61.4 |
| | 48½ | 48.6 | 52.0 | 58.7 | 59.9 | 63.6 | 67.3 |
| | 56½ | 57.2 | 61.2 | 69.0 | 70.5 | 74.8 | 79.2 |
| | 64½ | 65.7 | 70.3 | 79.4 | 81.0 | 86.0 | 91.0 |
| | 20½ | 20.4 | 22.0 | 25.0 | 25.7 | 27.4 | 29.3 |
| | 24½ | 25.2 | 27.1 | 30.9 | 31.7 | 33.8 | 36.2 |
| 28½ | 30.0 | 32.3 | 36.8 | 37.7 | 40.2 | 43.1 | |
| 10-1/4" | 32½ | 34.8 | 37.4 | 42.6 | 43.7 | 46.6 | 50.0 |
| | 36½ | 39.6 | 42.6 | 48.5 | 49.8 | 53.1 | 56.9 |
| | 40½ | 44.4 | 47.7 | 54.4 | 55.8 | 59.5 | 63.7 |
| | 44½ | 49.2 | 52.9 | 60.3 | 61.8 | 65.9 | 70.6 |
| | 48½ | 53.9 | 58.0 | 66.1 | 67.8 | 72.3 | 77.5 |
| | 56½ | 63.5 | 68.3 | 77.9 | 79.9 | 85.2 | 91.2 |
| | 64½ | 73.1 | 78.6 | 89.6 | 91.9 | 98.0 | 105.0 |

CH.4-INSTALLED RADIATION: DETERMINING HEAT LOAD

STEEL PIPE COILS - Sq Ft of Radiating Surface Per Linear Ft of Coil

| Nominal Pipe Size (Inches) | Number of Rows of Pipe In Coil | | | | | | | |
|----------------------------|--------------------------------|------|------|------|------|------|------|------|
| | 1 | 2 | 3 | 4 | 5 | 6 | 8 | 10 |
| 1 | .57 | 1.09 | 1.53 | 1.90 | 2.20 | 2.44 | 2.83 | 3.14 |
| 1-1/4 | .71 | 1.36 | 1.91 | 2.37 | 2.74 | 3.04 | 3.52 | 3.90 |
| 1-1/2 | .81 | 1.55 | 2.18 | 2.70 | 3.12 | 3.46 | 4.02 | 4.45 |
| 2 | .95 | 1.81 | 2.55 | 3.17 | 3.66 | 4.05 | 4.71 | 5.22 |

- Notes:
1. Based on 70°F room temperature.
 2. Pipes are positioned level, on vertical wall.
 3. For coils positioned along floor or ceiling.
Multiply chart value for 1 row of pipe x no. of rows of pipe.



CORRECTION FACTORS IF ROOM TEMPERATURE IS OTHER THAN 70° DIVIDE SQ. FT. OF RADIATION BY

| Rm. Temp | 80° | 75° | 70° | 65° | 60° | 55° | 50° |
|----------|------|------|------|------|------|------|------|
| Divisor | 1.10 | 1.05 | 1.00 | 0.96 | 0.92 | 0.88 | 0.85 |

| Heat Emissions for Cast Iron Radiators | | | |
|--|---------------------------------------|-------------------------------------|--------------------------------------|
| Design or Average Water Temperature | Heat Emissions Rates Btuh per sq. ft. | Design or Average Water Temperature | Heat Emission Rates Btuh per sq. ft. |
| 110° | 30 | 180° | 170 |
| 120° | 50 | 185° | 180 |
| 130° | 70 | 190° | 190 |
| 140° | 90 | 195° | 200 |
| 150° | 110 | 200° | 210 |
| 160° | 130 | 205° | 220 |
| 170° | 150 | 210° | 230 |
| 175° | 160 | 215° | 240 |

CH.4-INSTALLED RADIATION: DETERMINING HEAT LOAD

HEAT LOSSES FROM BARE STEEL PIPE

Based On 70° Surrounding Air

| Diameter of Pipe, Inches | Temperature of Pipe, Deg. F | | | | | |
|--------------------------|---|-----|-----|------|------|------|
| | 100 | 120 | 150 | 180 | 210 | 240 |
| | Heat Loss per Lineal Foot of Pipe, BTU per Hour | | | | | |
| 1/2 | 13 | 22 | 40 | 60 | 82 | 106 |
| 3/4 | 15 | 27 | 50 | 74 | 100 | 131 |
| 1 | 19 | 34 | 61 | 90 | 123 | 160 |
| 1-1/4 | 23 | 42 | 75 | 111 | 152 | 198 |
| 1-1/2 | 27 | 48 | 85 | 126 | 173 | 224 |
| 2 | 33 | 59 | 104 | 154 | 212 | 275 |
| 2-1/2 | 39 | 70 | 123 | 184 | 252 | 327 |
| 3 | 46 | 84 | 148 | 221 | 303 | 393 |
| 3-1/2 | 52 | 95 | 168 | 250 | 342 | 444 |
| 4 | 59 | 106 | 187 | 278 | 381 | 496 |
| 5 | 71 | 129 | 227 | 339 | 464 | 603 |
| 6 | 84 | 151 | 267 | 398 | 546 | 709 |
| 8 | 107 | 194 | 341 | 509 | 697 | 906 |
| 10 | 132 | 238 | 420 | 626 | 857 | 1114 |
| 12 | 154 | 279 | 491 | 732 | 1003 | 1305 |
| 14 | 181 | 326 | 575 | 856 | 1173 | 1527 |
| 16 | 203 | 366 | 644 | 960 | 1314 | 1711 |
| 18 | 214 | 385 | 678 | 1011 | 1385 | 1802 |
| 20 | 236 | 426 | 748 | 1115 | 1529 | 1990 |

HEAT LOSSES FROM BARE TARNISHED COPPER TUBE

Based On 70° Surrounding Air

| Diameter of Pipe, Inches | Temperature of Pipe, Deg. F | | | | | |
|--------------------------|---|-----|-----|-----|-----|-----|
| | 100 | 120 | 150 | 180 | 210 | 240 |
| | Heat Loss per Lineal Foot of Pipe, BTU per Hour | | | | | |
| 1/4 | 4 | 8 | 14 | 21 | 29 | 37 |
| 3/8 | 6 | 10 | 18 | 28 | 37 | 48 |
| 1/2 | 7 | 13 | 22 | 33 | 45 | 59 |
| 5/8 | 8 | 15 | 26 | 39 | 53 | 68 |
| 3/4 | 9 | 17 | 30 | 45 | 61 | 79 |
| 1 | 11 | 21 | 37 | 55 | 75 | 97 |
| 1-1/4 | 14 | 25 | 45 | 66 | 90 | 117 |
| 1-1/2 | 16 | 29 | 52 | 77 | 105 | 135 |
| 2 | 20 | 37 | 66 | 97 | 132 | 171 |
| 2-1/2 | 24 | 44 | 78 | 117 | 160 | 206 |
| 3 | 28 | 51 | 92 | 136 | 186 | 240 |
| 3-1/2 | 32 | 59 | 104 | 156 | 212 | 274 |
| 4 | 36 | 66 | 118 | 174 | 238 | 307 |
| 5 | 43 | 80 | 142 | 212 | 288 | 373 |
| 6 | 51 | 93 | 166 | 246 | 336 | 432 |
| 8 | 66 | 120 | 215 | 317 | 435 | 562 |
| 10 | 80 | 146 | 260 | 387 | 527 | 681 |
| 12 | 94 | 172 | 304 | 447 | 621 | 802 |

CH.4-INSTALLED RADIATION: DETERMINING HEAT LOAD

| HEAT LOSSES FROM COVERED PIPE 85 PERCENT MAGNESIA TYPE BTU PER LINEAR FOOT PER HOUR PER °F TEMPERATURE DIFFERENCE (SURROUNDING AIR ASSUMED 75 °F) | | | | | |
|--|-------------------------------------|--------------------------------|------|------|------|
| Pipe Size | Insulation, Thickness, Inches | Max. Temp. of Pipe Surface °F. | | | |
| | | 125 | 175 | 225 | 275 |
| ½ | 1 | .145 | .150 | .157 | .160 |
| ¾ | 1 | .165 | .172 | .177 | .180 |
| 1 | 1 | .190 | .195 | .200 | .203 |
| | 1½ | .160 | .165 | .167 | .170 |
| 1¼ | 1 | .220 | .225 | .232 | .237 |
| | 1½ | .182 | .187 | .193 | .197 |
| 1½ | 1 | .240 | .247 | .255 | .260 |
| | 1½ | .200 | .205 | .210 | .215 |
| 2 | 1 | .282 | .290 | .297 | .303 |
| | 1½ | .230 | .235 | .240 | .243 |
| | 2 | .197 | .200 | .205 | .210 |
| 2½ | 1 | .322 | .330 | .340 | .345 |
| | 1½ | .260 | .265 | .270 | .275 |
| | 2 | .220 | .225 | .230 | .237 |
| 3 | 1 | .375 | .385 | .395 | .405 |
| | 1½ | .300 | .305 | .312 | .320 |
| | 2 | .253 | .257 | .263 | .270 |
| 3½ | 1 | .419 | .430 | .440 | .450 |
| | 1½ | .332 | .339 | .345 | .352 |
| | 2 | .280 | .285 | .290 | .295 |
| 4 | 1 | .460 | .470 | .480 | .492 |
| | 1½ | .362 | .370 | .379 | .385 |
| | 2 | .303 | .308 | .315 | .320 |
| 5 | 1 | .545 | .560 | .572 | .585 |
| | 1½ | .423 | .435 | .442 | .450 |
| | 2 | .355 | .360 | .367 | .375 |
| 6 | 1 | .630 | .645 | .662 | .680 |
| | 1½ | .487 | .500 | .510 | .520 |
| | 2 | .405 | .415 | .420 | .430 |

CHAPTER 5 - WATER CONTENT EXISTING RADIATION/PIPING

A. RADIATION

| Based on Sq. Ft. Rating | Water Content/Gal. | Weight/Lbs. |
|--------------------------------|--------------------|-------------|
| Slenderized Radiators | 0.066 | 4.5 |
| Large Tube Radiators | 0.103 | 5.25 |
| Column Radiators | 0.188 | 7.0 |
| Convectors (Non-Ferrous) | 0.004 | 1.5 |
| Convectors (Cast Iron) | 0.040 | 3.4 |
| Radiant Radiators | 0.066 | 5.0 |
| Base-Ray (Cast Iron Baseboard) | 0.066 | 4.4 |

B. STEEL AND WROUGHT IRON PIPE (STD. WGT.)

| Nominal Size: Inches | Based On Lineal Foot | |
|-------------------------|----------------------|-------------|
| | Water Content/Gal. | Weight/Lbs. |
| 1/2" | .016 | .132 |
| 3/4" | .028 | .231 |
| 1" | .045 | .375 |
| 1-1/4" | .078 | .648 |
| 1-1/2" | .106 | .883 |
| 2" | .174 | 1.455 |
| 2-1/2" | .249 | 2.076 |
| 3" | .384 | 3.205 |
| 4" | .661 | 5.519 |
| 5" | 1.039 | 8.662 |
| 6" | 1.501 | 12.510 |

C. COPPER TUBING (TYPE L)

| Nominal Size: Inches | Based on Lineal Foot | |
|-------------------------|----------------------|-------------|
| | Water Content/Gal. | Weight/Lbs. |
| 3/8" | .007 | .063 |
| 1/2" | .012 | .101 |
| 5/8" | .018 | .151 |
| 3/4" | .025 | .210 |
| 1" | .043 | .357 |
| 1-1/4" | .065 | .544 |
| 1-1/2" | .092 | .770 |
| 2" | .161 | 1.340 |

D. WATER CONVERSION FACTORS

Lbs. of Water x 0.12 = Gallons

Gallons of Water x 8.34 = Lbs.

CHAPTER 6-HEAT LOSS CALCULATION

SHORT FORM HEAT LOSS SURVEY

Application: Excellent when determining heat loss of a building as a whole. Precise method of sizing replacement hot water boilers.

*Heating Multipliers (H.M.) BTU/Hr. Based on 60°F Temperature Difference (T.D.)

| <u>WALL</u> | <u>*H.M.</u> | <u>CEILING</u> | <u>*H.M.</u> | <u>FLOOR</u> | <u>*H.M.</u> |
|---------------------|--------------|----------------|---------------------|---------------|--------------|
| No Insulation | 15 | 3 Inches | 5 | No Insulation | 4 |
| 2 Inches | 6 | 6 Inches | 4 | Overhang — 3" | 5 |
| 3 Inches | 4 | 10 Inches | 2 | Overhang — 6" | 3 |
| <u>WINDOWS/DOOR</u> | | <u>*H.M.</u> | <u>INFILTRATION</u> | | <u>*H.M.</u> |
| Storm | | 34 | 1-1/2 Air Change | | 1.61 |
| Insulated | | 41 | 1 Air Change | | 1.07 |
| Single | | 67 | 3/4 Air Change | | .81 |

PROCEDURE:

1. Measure the length (L) and width (W) of the outside walls of the house and record. Calculate gross wall area by multiplying height of the walls by total length of outside walls. (2L + 2W).
 2. Measure the window and door area and record.
 3. Record Net Wall Area = (gross wall area minus door and window area) select proper H.M.
 4. Measure and record the ceiling area and select H.M.
 5. Measure and record floor area and select H.M. (H.M. of 4 used unheated basement).
 6. Multiply Floor area by ceiling height to obtain volume of home and select proper air change factor: 1.61 for Loose House - 1.07 for Average House - .81 for Tight House.
- H.M. Floor over Basement T.D. 20°

WORKSHEET:

| LENGTH _____ | WIDTH _____ | HEIGHT _____ | |
|------------------------------|----------------|--------------|--------------|
| AREA (ft ²) | H.M. (BTU/Hr.) | | BTU/Hr. LOSS |
| GROSS _____ | X _____ | | = _____ |
| WINDOWS & DOORS _____ | X _____ | | = _____ |
| NET WALL _____ | X _____ | | = _____ |
| CEILING _____ | X _____ | | = _____ |
| FLOOR _____ | X _____ | | = _____ |
| INFILTRATION (CU. FT.) _____ | X _____ | | = _____ |
| (HEIGHT) X (FLOOR AREA) | | | |
| TOTAL HEAT LOSS | | | = _____ |
| TEMP. DIFFERENCE CORRECTION | | | = _____ |

*To Increase Temp. Difference to 70°F, Multiply Total Heat Loss 1.18
 80°F, Multiply Total Heat Loss 1.34
 90°F, Multiply Total Heat Loss 1.50
 100°F, Multiply Total Heat Loss 1.66

CHAPTER 6-HEAT LOSS CALCULATION

GREENHOUSE HEAT LOSS CALCULATION

The method described in this manual based on the method prepared and adopted by N.G.M.A. (National Greenhouse Manufacturer's Association).

The recommended equation for calculating heat loss is:

$[A1 + (A2 \times R)] \times TD \times G \times W \times C = \text{BTU/Hr. Heat Loss, where:}$

- A1 - Square feet of exposed glass area;
- A2 - Square feet to exposed wall area (other than glass);
- R - Resistance of greenhouse wall base transposed into a "glass" factor;
- T - Highest temperature to be maintained is greenhouse minus outside design temperature;
- G - Coefficient of transmission of glass is BTU/Hr./Ft.²/°F. (Table 2).
- W - Wind factor (Table 3);
- C - Construction factor (Table 4).

Example (See Figure 1):

Given: Glass area = 4040 ft.² Find: "R" for 8" brick wall = 0.43;
wall area = 810 ft.² TD x "G" (Table 5) = 79;
design temperature: "W" for 15 mph wind = 1.00;
inside = 60°F "C" for all metal, glass = 1.08
outside = 50°F
brick wall = 8" thick
wind = 15 mph or less

Calculate:

$$\begin{aligned}\text{Heat Loss} &= [A1 + (A2 \times R)] \times TD \times G \times W \times C \\ &= [404 + (810 \times 0.43)] \times 79 \times 1.00 \times 1.08 \\ &= 4388 \times 85.3 \\ &= 374,000 \text{ BTU/Hr.}\end{aligned}$$

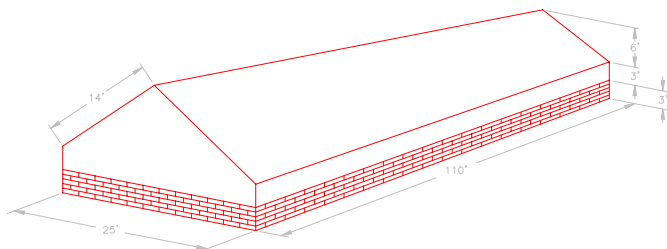


Figure 1
Typical Greenhouse

CHAPTER 6 - HEAT LOSS CALCULATION

TABLE 1 - "R" FACTOR

| | | |
|--|--|------|
| These are factors for "curtain" wall materials normally used on greenhouses: | | |
| 3/8 in. Corrugated asbestos cement board wall | | 0.94 |
| 4 in. Poured concrete wall | | 0.76 |
| 6 in. Poured concrete wall | | 0.67 |
| 8 in. Poured concrete wall | | 0.60 |
| 4 in. Concrete block wall | | 0.58 |
| 8 in. Concrete block wall | | 0.46 |
| 8 in. Brick Wall | | 0.43 |

**TABLE 2
COEFFICIENT OF
TRANSMISSION OF GLASS
("G" FACTOR)**

| Temp. Diff. Between Inside & Outside Design Temp. | Coefficient of Transmission |
|--|-----------------------------------|
| 50° | 1.09 |
| 55° | 1.10 |
| 60° | 1.11 |
| 65° | 1.12 |
| 70° | 1.13 |
| 75° | 1.14 |
| 80° | 1.15 |
| 85° | 1.16 |
| 90° | 1.17 |
| 95° | 1.18 |
| 100° | 1.19 |
| 105° | 1.20 |

**TABLE 3
WIND "W" FACTOR (2)**

| If wind velocity exceeds 15 mi. per hr. average during the heating season, the heat loss calculation should be increased by 4% for each 5 mi. per hr. over 15 as below: | | |
|---|---------------|--------------------------------|
| Wind Velocity In Miles Per Hour | "W" Factor | Alternate "W" Factor (3) |
| 15 m.p.h. or less | 1.00 | 1.10 |
| 20 m.p.h. | 1.04 | 1.14 |
| 25 m.p.h. | 1.08 | 1.18 |
| 30 m.p.h. | 1.12 | 1.22 |
| 35 m.p.h. | 1.16 | 1.26 |

TABLE 4 - CONSTRUCTION "C" FACTOR (4)

| | |
|---|------|
| The condition and type of a greenhouse directly affect the heat loss. Consult this table for the factor which best describes the greenhouse on which you are calculating the heat loss. | |
| All metal (good tight glass house - 20 or 24 in. glass spacing) | 1.08 |
| Wood & steel (good tight glass house - 16 or 20 in. glass spacing) (Metal gutters, vents, headers, etc.) | 1.05 |
| Wood houses (glass houses with wood bars, gutters, vents, etc.- up to and including 20 in. glass spacing) | |
| Good tight houses | 1.00 |
| Fairly tight houses | 1.13 |
| Loose houses | 1.25 |
| Fiberglass covered wood houses | 0.95 |
| Fiberglass covered metal houses | 1.00 |
| Plastic covered (5) metal houses (single thickness) | 1.00 |
| Plastic covered (6) metal houses (double thickness) | 0.70 |

CHAPTER 6 - HEAT LOSS CALCULATION

GREENHOUSE HEAT LOSS CALCULATION

TABLE 5 - HEAT REQUIREMENTS FOR GREENHOUSES

BTU/Hr. for each equivalent square foot of glass [C = 1.00 (7); W = 1.00 (8)]

| Inside Temp °F | Outside design Temperature - °F | | | | | | |
|----------------|---------------------------------|-----|-----|----|----|----|----|
| | -30 | -20 | -10 | 0 | 10 | 20 | 30 |
| 40 | 79 | 67 | 54 | 43 | 33 | 22 | 11 |
| 45 | 86 | 73 | 61 | 49 | 38 | 27 | 16 |
| 50 | 92 | 79 | 67 | 54 | 43 | 33 | 22 |
| 55 | 99 | 86 | 73 | 61 | 49 | 38 | 27 |
| 60 | 105 | 92 | 79 | 67 | 54 | 43 | 33 |
| 65 | 112 | 99 | 86 | 73 | 61 | 49 | 38 |
| 70 | 119 | 105 | 92 | 79 | 67 | 54 | 43 |
| 75 | 126 | 112 | 99 | 86 | 73 | 61 | 49 |

TABLE 6 - CUSTOMARY GREENHOUSE TEMPERATURES

| Type of Plant or Building Use | Temperature Range, °F |
|---|-----------------------|
| Carnation | 45-55 |
| Chrysanthemums, cut flowers and potted plants | 60-65 |
| Cool | 45-50 |
| Cucumber | 65-60 |
| Fern | 60-65 |
| Forcing | 60-65 |
| General purpose | 55-60 |
| Lettuce | 40-45 |
| Orchid, warm | 65-70 |
| Orchid, cool | 50-55 |
| Palm, warm | 60-65 |
| Palm, cool | 50-55 |
| Poinsettias | 60-70 |
| Propagating | 55-60 |
| Rose | 55-60 |
| Sweet pea | 45-50 |
| Tomato | 65-70 |
| Tropical | 65-70 |
| Violet | 40-45 |

NOTES

2. The effect of wind on heat loss cannot be overemphasized. If increased wind velocity is not compensated for, there is likely to be a 1/2° temperature drop for every mile per hour over 15 mph. These figures are based on observations by greenhouse heating design engineers over many years of observation.
3. Lapped glass on greenhouses normally freezes shut when the outdoor temperature reaches 15° above zero. If high design temperatures are used (70° or over), if unit heaters or turbulators are located so that they blow air against the glass, if 40% or more of the greenhouse radiation will be overhead, or if the "greenhouse" will be used for other than "growing" purposes, use alternate "W" factors above.
4. These figures are based on observations by greenhouse heating design engineers.
5. With no air leaks (holes, tears, etc.)
6. 1/4" minimum, 1/2" maximum distance between layers with no air leaks (holes, tears, etc.)

CH.7-FUEL CONSUMPTION AND ENERGY COST

There are several ways to calculate the operational cost of a specific boiler in a particular location. The accuracy of each will depend on the attention one gives the many variables.

Two popular methods are the “bin” and “degree-day”. Though the “bin” method has the potential to be more accurate, it is more complicated when necessary weather data is not readily available. Discussion will be limited to the “Degree-Day” method.

The “Degree-Day” method is not without controversy. Historically, the reference temperature is 65°. This assumes that a building with average insulation and a thermostat set at 70° will cause a boiler to begin operation when outdoor temperature falls below 65°. Overlooked variables could be sunshine, wind, heavier insulation, different thermostat setting.

WHAT IS A DEGREE DAY?

When the outside temperature falls below 65°F, heat will be required to maintain the temperature within the building. The average outside temperature is estimated by adding the high and low temperature for a given day and dividing by two. (Example: A high of 40°F and a low of 20°F would be equivalent to a 30°F average temperature.) “Degree Days” are defined as the difference between the average temperature and the 65°F reference temperature. Therefore, 65°F minus 30°F equals 35 “Degree Days”.

$$\text{Degree Days} = 65 - (\text{high} + \text{low}) \div 2$$

FUEL CONSUMPTION

In order to determine fuel usage the D.D. is used in the following equation:

$$F = \frac{HL \times 24 \times DD}{E \times P \times T.D.}$$

| | | |
|------|---|---|
| HL | = | Heating Load (Btuh) |
| DD | = | Degree Day |
| 24 | = | Hours in a day |
| E | = | *Boiler Efficiency (AFUE) |
| P | = | Heating value of fuel (Btu) |
| T.D. | = | Design temperature difference (inside-outside) |
| F | = | Annual fuel consumption |

*Example: 80% = .8

CH.7-FUEL CONSUMPTION AND ENERGY COST

HEATING DEGREE DAYS (SAMPLING) USA/CANADA

| STATE/PROVINCE/CITY | D.D. | STATE/PROVINCE/CITY | D.D. |
|-----------------------------|-------|----------------------|------|
| Alaska | | Indianapolis | 5630 |
| Anchorage | 10860 | South Bend | 6460 |
| Barrow | 20265 | | |
| Fairbanks | 14290 | Iowa | |
| | | Des Moines | 6610 |
| Colorado | | Dubuque | 7380 |
| Alamosa | 8529 | | |
| Colorado Springs | 6410 | Kansas | |
| Denver | 6150 | Topeka | 5210 |
| | | Wichita | 4640 |
| Connecticut | | | |
| Bridgeport | 5617 | Kentucky | |
| Hartford | 6170 | Lexington | 4760 |
| Waterbury | 6672 | Louisville | 4610 |
| | | | |
| Delaware | | Maine | |
| Wilmington | 4930 | Bangor | 8220 |
| | | Portland | 7570 |
| District of Columbia | | | |
| Washington | 4240 | Maryland | |
| | | Baltimore | 4680 |
| Georgia | | Hagerstown | 5130 |
| Atlanta | 2990 | | |
| | | Massachusetts | |
| Idaho | | Boston | 5630 |
| Boise | 5830 | Lowell | 6060 |
| Pocatello | 7030 | Pittsfield | 7580 |
| | | Worcester | 6970 |
| Illinois | | | |
| Champaign/Urbana | 5800 | Michigan | |
| Chicago, O'Hare | 6640 | Battle Creek | 6580 |
| Rockford | 6840 | Detroit | 6290 |
| Springfield | 5530 | Flint | 7200 |
| | | Marquette | 8390 |
| Indiana | | Traverse City | 7700 |
| Fort Wayne | 6220 | | |

CH.7-FUEL CONSUMPTION AND ENERGY COST

HEATING DEGREE DAYS (SAMPLING) USA/CANADA

| STATE/PROVINCE/CITY | D.D. | STATE/PROVINCE/CITY | D.D. |
|----------------------|-------|-----------------------|------|
| Minnesota | | New York | |
| Duluth | 9890 | Albany | 6900 |
| International Falls | 10600 | Binghamton | 7340 |
| Minneapolis/St. Paul | 8250 | Buffalo | 6960 |
| | | NYC-Central Park | 4880 |
| | | Poughkeepsie | 5820 |
| Missouri | | Syracuse | 6720 |
| Kansas City | 4750 | | |
| St. Louis | 4900 | North Carolina | |
| | | Ashville | 4130 |
| Montana | | | |
| Billings | 7150 | North Dakota | |
| Butte | 9730 | Bismark | 8960 |
| Helena | 8190 | Fargo | 9250 |
| | | | |
| Nebraska | | Ohio | |
| Lincoln | 6050 | Cleveland | 6200 |
| Omaha | 6290 | Columbus | 5670 |
| | | | |
| Nevada | | Pennsylvania | |
| Reno | 6150 | Harrisburg | 5280 |
| | | Johnstown | 7804 |
| New Hampshire | | Philadelphia | 5180 |
| Berlin | 8270 | Pittsburgh | 5950 |
| Concord | 7360 | Scranton/Wilkes-Barre | 6160 |
| Manchester | 7100 | | |
| | | Rhode Island | |
| New Jersey | | Providence | 5950 |
| Atlantic City | 4810 | | |
| Newark | 4900 | South Dakota | |
| Trenton | 4980 | Aberdeen | 8620 |
| | | Pierre | 7550 |
| | | | |
| New Mexico | | Utah | |
| Albuquerque | 4250 | Logan | 6750 |
| Santa Fe | 6120 | Salt Lake City | 5990 |

CH.7-FUEL CONSUMPTION AND ENERGY COST

HEATING DEGREE DAYS (SAMPLING) USA/CANADA

| STATE/PROVINCE/CITY | D.D. | STATE/PROVINCE/CITY | D.D. |
|-------------------------|-------|-----------------------------|-------|
| Vermont | | Manitoba | |
| Burlington | 8030 | Winnipeg | 10679 |
| Rutland | 7440 | | |
| Virginia | | New Brunswick | |
| Lynchburg | 4166 | Saint John | 8453 |
| Richmond | 3910 | | |
| Roanoke | 4150 | Newfoundland | |
| Winchester | 4780 | Gander | 9254 |
| | | St. John's | 8991 |
| Washington | | Northwest Terr. | |
| Seattle-Tacoma | 5190 | Resolute | 22673 |
| Spokane | 6770 | Yellowknife | 15634 |
| West Virginia | | Nova Scotia | |
| Charleston | 4510 | Halifax | 7361 |
| Morgantown | 5100 | Sydney | 8049 |
| Wheeling | 5220 | | |
| Wisconsin | | Ontario | |
| Green Bay | 8100 | London | 7349 |
| Milwaukee | 7470 | Thunder Bay | 10405 |
| Wausau | 8490 | Toronto | 6827 |
| | | Windsor | 6579 |
| Wyoming | | Prince Edward Island | |
| Casper | 7510 | Charlottetown | 8486 |
| Cheyenne | 7370 | | |
| Alberta | | Quebec | |
| Calgary | 9703 | Montreal | 8213 |
| Edmonton | 10268 | Quebec | 8937 |
| British Columbia | | Saskatchewan | |
| Kamloops | 6799 | Regina | 10806 |
| Vancouver | 5515 | Saskatoon | 10856 |
| Victoria | 5579 | Yukon Territory | |
| | | Whitehorse | 12475 |

CH.7-FUEL CONSUMPTION AND ENERGY COST

HOME HEATING COST COMPARISON BETWEEN BOILER AND FURNACE Fuel Consumption for Heating Systems Using Gas and Oil*

OIL HEAT-Gallons burned per year at the indicated heat-loss (approx.)

| <u>Heat Loss</u> <u>(Btuh)**</u> | <u>Hydronics</u> <u>(gallons)</u> | <u>Warm Air</u> <u>(gallons)</u> | <u>Hydronics</u> <u>Savings Per Year</u> <u>(gallons)</u> |
|-------------------------------------|--------------------------------------|-------------------------------------|---|
| 20,000 | 490 | 581 | 91 |
| 30,000 | 734 | 871 | 137 |
| 40,000 | 979 | 1,161 | 182 |
| 50,000 | 1,224 | 1,452 | 228 |
| 60,000 | 1,469 | 1,742 | 273 |
| 70,000 | 1,714 | 2,032 | 319 |
| 80,000 | 1,958 | 2,323 | 364 |
| 90,000 | 2,203 | 2,613 | 410 |
| 100,000 | 2,448 | 2,903 | 455 |
| 110,000 | 2,693 | 3,194 | 501 |
| 120,000 | 2,938 | 3,484 | 546 |

GAS HEAT-Therms burned per year at the indicated heat-loss (approx.)

| <u>Heat Loss</u> <u>(Btuh)**</u> | <u>Hydronic</u> <u>(therms)</u> | <u>Warm Air</u> <u>(therms)</u> | <u>Hydronic</u> <u>Savings Per Year</u> <u>(therms)</u> |
|-------------------------------------|------------------------------------|------------------------------------|---|
| 20,000 | 676 | 801 | 126 |
| 30,000 | 1,014 | 1,202 | 188 |
| 40,000 | 1,351 | 1,603 | 251 |
| 50,000 | 1,689 | 2,003 | 314 |
| 60,000 | 2,027 | 2,404 | 377 |
| 70,000 | 2,365 | 2,804 | 440 |
| 80,000 | 2,703 | 3,205 | 502 |
| 90,000 | 3,041 | 3,606 | 565 |
| 100,000 | 3,378 | 4,006 | 628 |
| 110,000 | 3,716 | 4,407 | 691 |
| 120,000 | 4,054 | 4,808 | 754 |

*Hydronics = Circulating hot water with boiler and baseboard.

Warm Air = Blower system with furnace, ducts and registers.

**Calculated heat loss per hour in BTU's.

CH.7-FUEL CONSUMPTION AND ENERGY COST

NOTE: The average fuel saving is 18.6% for Hydronic systems over Warm air systems.

Hydronic system efficiency - DOE Minimum seasonal efficiency (AFUE) 80%¹ - Piping Losses 7½%² = 74% System Efficiency

Warm air system efficiency - DOE Minimum seasonal efficiency (AFUE)-78%³ - Duct losses 20%⁴ = 62.4% System Efficiency

To determine the actual dollar savings, multiply the cost of a gallon of oil or a therm of gas, for your area, times gallons or therms saved.

1. Department of Energy Minimum Seasonal Efficiency (AFUE) for boilers as of January 1, 1992
2. "Summary Cooperative Research of Hydronic Heating and Cooling," University of Illinois, 1968
3. Department of Energy Minimum Seasonal Efficiency (AFUE) for furnaces, as of January 1, 1992
4. "Air Conditioning, Heating and Refrigeration News," February 19, 1973

Example #1: Using Oil

For a hydronically-heated oil-fueled home, with a heat loss of 40,000 Btuh: 40,000 heat loss house divided by 138,000 Btu/gallon for oil divided by .74 (system efficiency) x 2500 hours = 979 gallons used.

For a warm air-heated oil fueled home, with a heat loss of 40,000 Btuh: 40,000 heat loss house divided by 138,000 Btu/gallon for oil divided by .624 (system efficiency) x 2500 hours = 1,161 gallons used.

Example #2: Using Gas

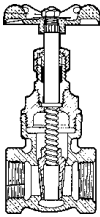
For a hydronically-heated gas-fueled home, with a heat loss of 40,000 Btuh: 40,000 heat loss house divided by 100,000 Btu/therm for gas divided by .74 (system efficiency) x 2500 hours = 1,351 therms used.

For a warm air-heated gas-fueled home, with a heat loss of 40,000 Btuh: 40,000 heat loss house divide by 100,000 Btu/therm for gas divided by .624 (system efficiency) x 2500 hours = 1,603 therms used.

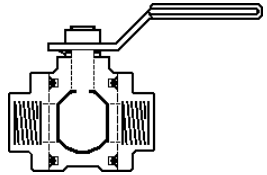
NOTE: For the Northeast and Upper Midwest there are approximately 2,500 heat loss hours per year. (From GAMA "Consumers' Directory of Certified Efficiency Ratings," April, 1992)

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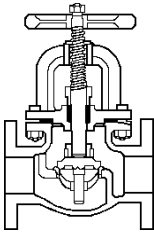
CHAPTER 8-VALVES: TYPES AND APPLICATIONS



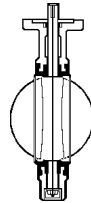
Gate Valve Has a wedge-shaped disc that is raised to open and lowered to close the valve. It is used either fully open or totally closed and is not designed for throttling.



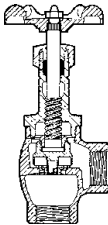
Ball Valve Its operating device consists of a ball with a hole through its center. The valve operator rotates 90 degrees from fully open to the fully closed position. Its major application is isolation but can be used to regulate flow.



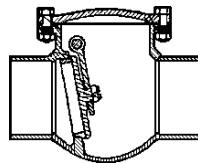
Globe Valve Uses a round disc or tapered plug-type disc that seats against a port to close the valve. Globe valves are used where throttling and/or when frequent operation is required.



Butterfly Valve Has a wafer-shaped body with a thin, rotating disc closing device. Like the ball valve, the butterfly operates with a one-quarter turn to go full open to closed. The disc is always in the flow but the disc is thin and offers little flow restriction.

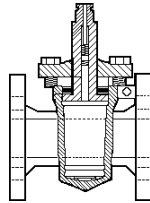
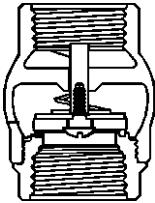


Angle Valve Operates the same as globe valve. Has less resistance to flow than its globe counterpart and by design acts as a 90° elbow, eliminating the need for a fitting.



Swing Check Valve Has a hinged disc that swings on a hinge pin. When the flow reverses, the pressure pushes the disc against a seat preventing back flow.

CHAPTER 8 - VALVES: TYPES AND APPLICATIONS



Lift Check Valve Has a guided disc that rises from its seat by upward flow pressure. Flow reversal pushes the disc down against its seat, stopping back flow. Though the lift check has greater flow resistance than the swing check, it is more suitable for rapid operation cycles.

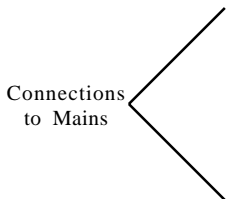
Plug Valve Uses a tapered cylindrical plug that fits a body seat of corresponding shape. A one-quarter turn (90°) operates the valve from open to close. Valve plugs may be lubricated or non-lubricated.

APPLICATION

| Valve Type | Principle Service Function | Full Open-Flow Resistance | Throttling Application | Closure Speed | Allowable Frequency of Operation | Positive Shut-off |
|-------------|----------------------------|---------------------------|------------------------|---------------|----------------------------------|-------------------|
| Gate | Isolation | Low | Unacceptable | Slow | Low | Poor |
| Globe | Flow regulation | High | Good | Moderate | Moderate | Good |
| Angle | Flow regulation | Medium-high | Good | Moderate | Moderate | Good |
| Ball | Isolation | Low | Fair | Fast | High | Good |
| Butterfly | Isolation | Low | Fair to good | Fast | High | Good |
| Swing check | Flow-reversal control | Low | N/A | Fast | Low | Poor |
| Lift check | Flow-reversal control | High | N/A | Moderate | Moderate | Fair |
| Plug | Flow regulation/ isolation | Low | Good to excellent | Fast | Moderate | Good |

CH.9-STANDARD DRAWINGS USED IN HEATING

CONNECTIONS



from top



from side



from bottom



Drop in Main



Rise in Main



FITTINGS

Pipe Flange



Union



Strainer



MISCELLANEOUS

Pump



Hot Water In-Line



Vacuum



Condensation

Tank



Tank - Designate Type

Thermostat



CH.9-STANDARD DRAWINGS USED IN HEATING

PIPING

Anchor



Condensate Discharge



Dry Return



Eccentric Reducer



Expansion Joint



First Floor Radiation



Hanger or Support



High Pressure Steam



Low Pressure Steam



Medium Pressure Steam



Reducer



Riser and Designation Number



Vacuum Vent Line



Wet Return



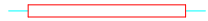
CH.9-STANDARD DRAWINGS USED IN HEATING

RADIATION

Fin Tube Radiation

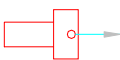


Heat Transfer Surface



Unit Heaters

Propeller Fan Type

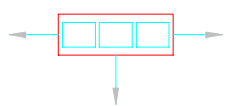


Horizontal



Vertical

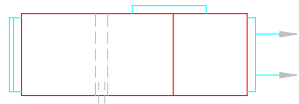
Centrifugal Fan Type



Unit Ventilator



Unit Ventilator - Auditorium Type



TRAPS

Float Trap



Alternates



Float and
Thermostatic
Trap



Thermostatic
Trap



CH.9-STANDARD DRAWINGS USED IN HEATING

VALVES

Alternates

Angle



Ball



Butterfly Valve



Check Valve



Three Way



Diaphragm Valve



Four Way



Gate Valve



Globe Valve



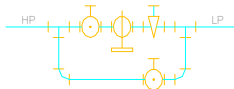
Plug Valve



Motor Operated Valve



Pressure Reducing Valve



Relief Valve



CHAPTER 10-VENTING FLUE PRODUCTS

A. ACCEPTABLE VENTING MATERIAL

| <u>Type</u> | <u>Temperature</u> | <u>Application</u> |
|--|--|--|
| AL29-4C® (Proprietary stainless steel pipe) | 100°F to 480°F | Pressure vented gas equipment |
| Single-wall metal pipe on application (1) | Refer to codes and to manufacturer's recommendations | Aluminum, galvanized steel or stainless steel; depending the |
| B and BW vent | To 550°F | Gas equipment |
| L vent | To 1000°F | Oil equipment (1) |
| Factory-built chimney | 500°F to 2,200°F | Oil or gas-fired equipment (1) |
| Masonry chimney (tile liner and air space) | 360°F to 1,800°F | Gas or oil equipment (metal liner may be required.) (2) |

- (1) Stainless-steel vents can resist the heat that is associated with oil-fired vent gases, but when the inner wall temperature is below 250°F, it loses its ability to resist acidic damage.
- (2) Masonry chimneys also are susceptible to acidic damage.

CHAPTER 10- VENTING FLUE PRODUCTS

B. METAL VENTS

| Type | Description |
|------|--|
| “B” | Double-wall round gas vent pipe with a relatively low permissible temperature of vent gases. |
| “BW” | Double-wall oval gas vent pipe designed to fit within a conventional stud wall. Same application as Type “B”. |
| “L” | Double-wall round stainless steel insulated. Type “L” is rated for higher temperature vent gas than the Type “B”. Type “L” may be used in lieu of “B” but not vice versa. |
| “C” | Single-wall galvanized |

IMPORTANT:

**VENTING GUIDE FOR ALL BURNHAM RESIDENTIAL GAS
BOILERS -- SEE PAGE 109**

CHAPTER 10-VENTING FLUE PRODUCTS

C. VENTING CATEGORIES: GAS-FIRED EQUIPMENT

| Operating characteristics | Category I | Category II | Category III | Category IV |
|--|---|----------------------|----------------------|----------------------|
| Pressure in the vent | Negative | Negative | Positive | Positive |
| Temperature of vent gas (4) | Above 275°F | Below 275°F | Above 275°F | Below 275°F |
| Annual efficiency | Below 84% | Above 84% | Below 84% | Above 84% |
| Condensation | Not acceptable (1) | Possible (in vent) | Possible (3) | In heat exchanger |
| Design requirements | | | | |
| Gas (air) tight vent | No | No | Yes | Yes |
| Corrosion-resistant vent (water tight) | No (1) | Yes | Possible (3) | Yes |
| Vent into masonry chimney | Permitted (1) and (2) | No | No | No |
| Combined venting | Permitted | No | No | No |
| Condensate drain | No | Ask manufacturer | Possible (3) | Yes (At equipment) |
| Source of information | N.F.G.C. Fuel gas code, heating equipment and vent system manufacturers | Manufact. literature | Manufact. literature | Manufact. literature |

NOTE 1 Usually, there is no problem when high vent gas temperature equipment is vented into double-wall vent or into a lined masonry chimney; but condensation could occur if mid-efficiency (80% to 84%) mechanical draft equipment is vented into a vent that has highly conductive walls, cold walls, or massive walls. In this case, design a vent system that minimizes the wall losses (use double-wall pipe for the whole run and avoid long runs through cold spaces).

NOTE 2 Install either a rigid or flexible metal liner inside of the masonry chimney and use a double-wall connector when venting mid-efficiency (80% to 84%), mechanical draft equipment.

NOTE 3 Condensation in the vent is possible with some types mid-efficiency (80% to 84%), direct-vent equipment, depend-

ing on the ambient temperature and the conductivity of the vent walls. In this case, design a vent system that minimizes the wall losses (use insulated pipe and avoid long runs through cold spaces). A corrosion-resistant flue and a drain may be required if condensation cannot be prevented — refer to the manufacturer's recommendations.

NOTE 4 The dewpoint of the vent gas depends on the fuel (natural or LP gas), the amount of excess air and the amount of dilution air. The limiting case occurs when the dewpoint of the vent gas is at a maximum, which is about 135°F. This maximum is produced when natural gas is burned with no excess air or dilution air. Therefore 275°F = 135°F dewpoint + 140°F rise.

CHAPTER 11-BASIC ELECTRICITY FOR HEATING

A. UNITS OF ELECTRICAL MEASUREMENT

- **VOLTAGE:** In order for current to flow in any conductor there must be an excess of electrons at one point and a deficiency of electrons at another point. The amount of difference between the two points is called "potential" or "voltage".
- **AMPERES:** current flow
- **OHMS:** resistance or "voltage drop" through a conductor
- **WATTS:** amount of power (volts multiplied by amperes)

$$\text{Watts} = \text{Amps} \times \text{volts}$$

$$\text{Volts} = \text{watts} \div \text{amps}$$

$$\text{Amps} = \text{watts} \div \text{volts}$$

Note: For convenience, **magnitude** of any of the above is expressed with the following prefixes:

$$\text{KILO} = 1,000 \text{ (one kilowatt} = 1,000 \text{ watts)}$$

$$\text{MEG(A)} = 1,000,000 \text{ (one megohm} = 1,000,000 \text{ ohms)}$$

$$\text{MILLI} = 1,000 \text{ divisor (one millivolt} = 1/1,000\text{th of a volt)}$$

$$\text{MICRO} = 1,000,000 \text{ divisor (one microampere} = 1/1,000,000\text{th of an ampere)}$$

B. DIRECT CURRENT AND ALTERNATING CURRENT

- Direct Current (D.C.) is a continuous current that always flows in the same direction.
- Alternating Current (A.C.) is a current that periodically "cycles", or reverses its direction of flow. The number of complete cycles that occur in one second is the frequency of the current.
- Commercially produced power, and the utilization equipment for use on commercial power, is almost universally A.C.

C. MOTORS (TYPES)

- Universal: Essentially a D.C. motor modified to run on A.C. Primarily used in small household appliances (mixers, shavers, small power tools). Single phase only.

- Capacitor Start/Induction Run: Has good torque characteristics and is used in heavy duty applications such as pumps, blowers, and large power burners.
- Shaded Pole: The simplest of all A.C. motors, it operates only on single phase current. Is available in sizes up to and including 1/3 H.P. It has no brushes or commutator. Has very low starting torque and therefore should not be used on equipment that is hard to start in motion. Some typical applications are small fans, heater blowers, etc.
- Split Phase: Has starting windings and a centrifugal switch. Its simplicity, good torque, and rugged construction give it a wide range of fractional horsepower applications.

D. TRANSFORMERS

- A transformer transfers power from one circuit to another circuit of different voltage, without an electrical connection between the two circuits.
- Input of a transformer must be A.C. current, and the output will also be A.C. current.
- A step-up transformer increases voltage but the available amperage decreases. A step-down transformer decreases voltage but the available amperage increases.
- The safe load rating of a transformer is expressed in VA. The actual power drawn depends on the connected load.

E. RELAYS

An electromagnetic switching device that can perform switching action in two or more circuits that have no electrical connection.

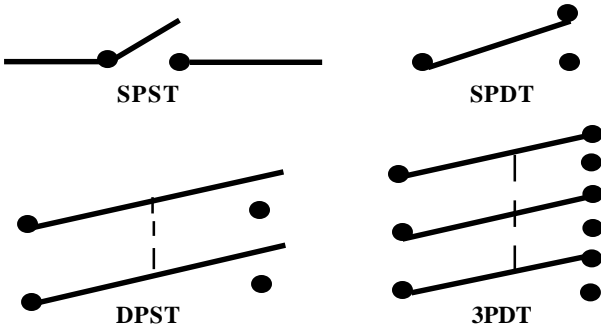
Relay Terminology

Relays are referred to as "single-pole-single throw", "single pole-double throw", "double pole-single throw", etc.

The poles are stationary and are usually used for power input. The arms extending from the poles of each switch or relay are mechanically linked together but are independent of each other electrically.

Relay Terminology (con't)

The arms extending from each pole include a moveable contact. the number of throws is the number of positions in which the moveable contact, connected to each pole, can complete a circuit. See examples below:



The dotted lines indicate that the arms are mechanically linked and will move together.

- Normally open, Normally closed: refers to the position of the switch contacts when the magnetic coil is not energized.

Relay Types

1. Time delay relay: incorporates a means for delaying the opening or closing of the load contacts for a specific period of time after the control circuit is energized or de-energized.
2. Transfer relay: incorporates overlapping load contacts that permit the load to be transferred from one circuit to another when the coil is energized.
3. Transformer relay: incorporates a step-down transformer to power the low voltage control circuit.
4. Alarm silencing relay: incorporates contacts arranged to silence an audible alarm when a push button is depressed, without re-establishing the main load circuit to the controlled device.
5. Contactor: a heavy duty relay for control of motors or high amperage loads. May incorporate auxiliary holding contacts for interlocking with safety limits or other devices.

CHAPTER 11 - BASIC ELECTRICITY FOR HEATING

6. Balancing relay: used in bridge circuits to equalize resistance values.
7. Overload relay: opens the load circuit if current drawn exceeds a selected value (“magnetic overload relay”) or in response to heat generated by excessive current flow (“thermal overload relay”).

F. SWITCHES

- Switches are used to open (break) or complete (make) a circuit.
- Switches may be actuated manually, by heat, mechanically, or electromagnetically.
- Switches are identified by the number of poles provided, and whether they are single throw or double throw.

G. OVERLOAD PROTECTION

- Fuses and circuit breakers are used in the power circuit to protect against damage from a short circuit (circuit failure that permits the current to take a shortcut between the terminals of a power source).
- Motors have their own overload protection in addition to power circuit overload protection.

H. ELECTRIC SERVICE

- Single Phase Two Wire: residential service to older buildings, seldom used in new construction. Provides power at 115 volts.
- Single Phase Three Line: residential and light commercial service universally used in new construction. Provides power at 115/230 volts.
- Three Phase Three Wire: commercial/industrial service. Provides power at a number of nominal supply voltages (240, 480 or 600 volts; sometimes 220, 440 or 550 are available).
- Three Phase Four Wire: commercial/industrial service. Provides power at nominal system voltages of 208, 416 or 480 volts.

CAUTION: The above description is oversimplified but electrical characteristics of heating equipment must be compatible with the available power supply.

CHAPTER 12- HEATING INDUSTRY ACRONYMS

BUILDING CODES & STANDARDS

| | |
|---------------|--|
| ANSI | American National Standards Institute 11 West 42nd St., New York, NY 10036 |
| ASHRAE | American Society of Heating, Refrigeration and Air Conditioning Engineers, Inc. 1791 Tullie Circle NE, Atlanta, GA 30329 |
| BOCA | Building Officials & Code Administrators International, Inc. 4051 W. Flossmoor Rd., Country Club Hills, IL 60477 |
| CABO | Council of American Building Officials 5203 Leesburg Pike, Suite 708, Falls Church, VA 22041 |
| IAPMO | International Association of Plumbing & Mechanical Officials 20001 South Walnut Dr., Walnut, CA 91789 |
| ICBO | International Conference of Building Officials 5360 South Workman Mill Rd., Whittier, CA 90601 |
| NBS | National Bureau of Standards Washington, DC 20234 |
| NCSBCS | National Conference of States on Building Codes and Standards, Inc. 481 Carlisle Drive, Herndon, VA 22070 |
| NFPA | National Fire Protection Association Batterymarch Park, P.O. Box 9101, Quincy, MA 02269 |
| SBCCI | Southern Building Code Congress International, Inc. 900 Montclair Rd., Birmingham, AL 35213 |

PROFESSIONAL ASSOCIATIONS & SOCIETIES

| | |
|-------------|--|
| ABMA | American Boiler Manufacturers Association 950 N. Glebe Rd., Suite 160, Arlington, VA 22203 |
| ACCA | Air Conditioning Contractors of America 1513 16th Street NW, Washington, DC 20036 (formerly National Environmental Systems Contractors Association) |
| ARI | Air-Conditioning and Refrigeration Institute 1501 Wilson Blvd., Suite 600, Arlington, VA 22209 |
| ASGE | American Society of Gas Engineers P.O. Box 31756, Independence, OH 44131 |

CHAPTER 12-HEATING INDUSTRY ACRONYMS

| | |
|---------------|---|
| BHC | Better Heating-Cooling Council P.O. Box 218, Berkeley Heights, NJ 07922 |
| GAMA | Gas Appliance Manufacturers Association, Inc. 1901 N. Moore St., Suite 1100, Arlington, VA 22209 |
| GRI | Gas Research Institute 8600 W Bryn Mawr Ave., Chicago, IL 60631 |
| HYDI | Hydronics Institute 35 Russo Place, Berkeley Heights, NJ 07922 |
| MCAA | Mechanical Contractors Association of America, Inc. 1385 Piccard Drive, Rockville, MD 20850 |
| NAPHCC | National Association of Plumbing-Heating-Cooling Contractors 180 S Washington St., P.O. Box 6808, Falls Church, VA 22040 |
| RSES | Refrigeration Service Engineers Society 1666 Rand Rd., Des Plaines, IL 60016 |
| SMACNA | Sheet Metal and Air Conditioning Contractors' National Assoc. 8224 Old Courthouse Rd., Vienna, VA 22180 |

APPROVAL AGENCIES

| | |
|-----------------|--|
| ETL/ETLc | Intertech Testing Services 18 Boulden Circle, Suite 34, New Castle, DE 19720 |
| AGA | American Gas Association 1515 Wilson Blvd., Arlington, VA 22209 |
| CGA | Canadian Gas Association 55 Scarsdale Rd., Toronto, Ontario M3B 2R3 |
| IAS | International Approval Services Joint Venture of AGA and CGA (effective 1/1/96) Offices in Cleveland, OH, Los Angeles, CA, and Toronto, Ontario |
| ASME | American Society of Mechanical Engineers 345 East 47th St., New York, NY 10017 |
| CSA | Canadian Standards Association 178 Rexdale Blvd., Rexdale, ONT M9W 1R3, Canada |
| IBR | Institute of Boiler and Radiator Manufacturers superseded by Hydronics Institute |
| UL | Underwriters' Laboratories, Inc. 333 Pfingsten Rd., Northbrook, IL 60062 |

CHAPTER 13-THE METRIC SYSTEM

The metric system, a European system of measurement, was developed by the French during the latter part of the 18th century to standardize measurement units. By the end of the 19th century it was being used as a commercial unit of measure by most European nations as well as Central and South America.

The system is not new to the United States. As early as 1790 Thomas Jefferson had suggested its use. In 1866 an Act of Congress made the metric measure legal but not mandatory. Since World War II many other nations have changed to metric: Russia, The Peoples Republic of China, India, Japan, The United Kingdom, Germany, Mexico and Canada.

1. THE METRIC UNITS:

LENGTH:

The base unit of metric length is the meter -- this unit is 39.37 inches and is divided into one hundred centimeters. The meter is one thousandth part of the kilometer which is the unit that will replace the mile.

Symbols:

| | |
|------------|---------------|
| Millimeter | mm |
| Centimeter | cm |
| Decimeter | dm |
| meter | m |
| Kilometer | km = 1,000 m. |

AREA:

The base unit of metric area is the square meter -- this unit is one ten thousandth part of a hectare. The hectare is approximately 2½ acres in area and will probably replace the acre as a basis for land surveys.

Symbols:

| | |
|-------------------|---------------------------|
| Square centimeter | cm ² |
| Square decimeter | dm ² |
| Square meter | m ² |
| are | a |
| hectare | ha |
| Square kilometer | km ² = 100 ha. |

CHAPTER 13 - THE METRIC SYSTEM

VOLUME:

The base unit of metric volume is the liter -- this unit is 1.8 fl. ounces larger than the U.S. Quart. Drug dispensers prefer to work with cubic centimeters or milliliters as the unit is also known. One teaspoon is the equivalent of 5 milliliters. For measuring larger quantities, e.g. readymix concrete, cubic meters is the unit used.

Symbols:

| | |
|------------------|--|
| Cubic centimeter | cm ³ or cc. as it is commonly referred to milliliter ml |
| cubic decimeter | dm ³ |
| liter | l |
| cubic meter | m ³ = 1,000 l |

WEIGHT/MASS:

The base unit of metric weight is the gram -- this unit is approximately one thirtieth of an avoirdupois ounce and is mostly used in pharmaceutical and scientific work. The more convenient unit is the kilogram, weighing approximately 2.2 pounds.

Symbols:

| | |
|------------|---------------|
| milligram | mg |
| gram | g |
| Kilogram | kg |
| Metric ton | t = 1,000 kg. |


TEMPERATURE:

Most countries using the Metric System use the Celsius (formerly centigrade) thermometer where water freezes at 0°C and boils at 100°C. Comfortable room temperature would be about 22°C and body temperature 37°C.

Temperature conversion from Fahrenheit to Celsius (Centigrade):

The simple formula is: Fahrenheit to Celsius = $(F^{\circ} - 32^{\circ}) \div 1.8$
Celsius to Fahrenheit = $C^{\circ} \times 1.8 + 32$

CHAPTER 13-THE METRIC SYSTEM

| | °Fahrenheit | | °Celsius |
|------------------------|-------------------------|---|----------|
| Boiling Point of Water | 212.0 |  | 100 |
| | 167.0 | | 75 |
| | 122.0 | | 50 |
| | 104.0 | | 40 |
| | 98.6 | | 37 |
| Body Temperature | 95.0 | | 35 |
| | 85.0 | | 30 |
| | 77.0 | | 25 |
| | 72.0 | | 22.2 |
| | 68.0 | | 20 |
| Comfortable Room Temp. | 59.0 | 10 | |
| | 50.0 | 10 | |
| | 41.0 | 5 | |
| | 39.2 | 4 | |
| | 37.4 | 3 | |
| | 35.6 | 2 | |
| | 33.8 | 1 | |
| | 32 F | 0 C | |
| | Freezing Point of Water | | |

2. TABLES OF METRIC WEIGHTS AND MEASURES:

LINEAR MEASURE (LENGTH):

| | |
|---------------------|---------------------|
| 10 millimeters (mm) | = 1 Centimeter (cm) |
| 10 centimeters | = 1 Decimeter (dm) |
| | = 100 millimeters |
| 10 decimeters | = 1 Meter (m) |
| | = 1,000 millimeters |
| 10 meters | = 1 Dekameter |
| 10 dekameters | = 1 Hectometer (hm) |
| | = 100 meters |
| 10 hectometers | = 1 Kilometer (km) |
| | = 1,000 meters |

SQUARE MEASURE (AREA):

| | |
|---------------------------|---------------------------------------|
| 100 Square Millimeters | = 1 Sq. Centimeter (cm ²) |
| 10,000 Square Centimeters | = 1 Square Meter (m ²) |
| 100 Square meters | = 1 Are (a) |
| 100 Ares | = 1 Hectare (ha) |

CHAPTER 13 - THE METRIC SYSTEM

100 Hectares = 1 Square kilometer (km²)
= 1,000,000 square meters

VOLUME MEASURE:

1 milliliter = 1 cubic centimeter
(cm³ or cc)
1 liter = 0.001 cubic meter
10 milliliters (ml) = 1 Centiliter (cl)
10 centiliters = 1 Deciliter (dl)
= 100 milliliters
10 deciliters = 1 Liter (l)
= 1,000 milliliters
10 liters = 1 dekaliter
10 dekaliters = 1 Hectoliter (hl)
= 100 liters
10 hectoliters = 1 kiloliter
= 1,000 liters

CUBIC MEASURE (CAPACITY):

1,000 cubic millimeters = 1 cubic centimeter
(cm³ or cc)
1,000 cubic centimeters = 1 cubic decimeter
= 1,000,00 cubic millimeters
= 1 liter
1,000 cubic decimeters = 1 cubic meter

WEIGHT MEASURE:

10 milligrams (mg) = 1 centigram (cg)
10 centigrams = 1 decigram (dg)
10 decigrams = 1 gram (g)
10 grams = 1 dekagram
10 dekagrams = 1 hectogram (hg)
10 hectograms = 1 kilogram (kg)
1,000 kilograms = 1 Metric Ton

CHAPTER 13-THE METRIC SYSTEM

3. MULTIPLES AND SUBMULTIPLES:

| Prefix | Symbol | Equivalent | Factor |
|--------|--------|--------------------|-------------------|
| atto- | a | quintillionth part | $\times 10^{-18}$ |
| femto- | f | quadrillionth part | $\times 10^{-15}$ |
| pico- | p | trillionth part | $\times 10^{-12}$ |
| nano- | n | billionth part | $\times 10^{-9}$ |
| micro- | μ | millionth part | $\times 10^{-6}$ |
| milli- | m | thousandth part | $\times 10^{-3}$ |
| centi- | c | hundredth part | $\times 10^{-2}$ |
| deci- | d | tenth part | $\times 10^{-1}$ |
| deka | da | tenfold | $\times 10$ |
| hecto- | h | hundredfold | $\times 10^2$ |
| kilo- | k | thousandfold | $\times 10^3$ |
| mega- | M | millionfold | $\times 10^6$ |
| giga- | G | billionfold | $\times 10^9$ |
| tera- | T | trillionfold | $\times 10^{12}$ |

The small figure in the factor determines the position of the decimal point in multiplying out a conversion.

EXAMPLE:

$$3.0 \times 10^2 = 3.00 \times 10 \times 10$$

move decimal point 2 places to the right

$$= 300.00$$

$$6.37 \times 10^3 = 6.37 \times 10 \times 10 \times 10$$

move decimal point 3 places to the right

$$= 6370.00$$

$$6.37 \times 10^{-3} = .00637$$

move decimal point 3 places to the left

4. TABLE OF EQUIVALENTS:

LENGTH/DISTANCE:

| | |
|---------------------|--------------------|
| 1 chain (surveyors) | = 66 feet |
| | = 20.1168 meters |
| 1 fathom | = 6 feet |
| | = 1.8288 meters |
| 1 foot | = 0.3048 meter |
| 1 inch | = 2.54 centimeters |
| | = 25.4 millimeters |

CHAPTER 13 - THE METRIC SYSTEM

| | |
|-------------------|-----------------------|
| 1 mile (statute) | = 1.609 kilometers |
| 1 mile (nautical) | = 1.151 statute miles |
| | = 1.852 kilometers |
| 1 yard | = 0.9144 meter |
| 1 centimeter | = 0.3937 inch |
| 1 decimeter | = 3.937 inches |
| 1 dekameter | = 32.808 feet |
| 1 kilometer | = 0.621 mile |
| 1 meter | = 1.094 yards |
| | = 39.37 inches |
| 1 millimeter | = 0.03937 inch |

AREA/SURFACE:

| | |
|---------------------|------------------------------|
| 1 acre | = 4,840 square yards |
| | = 0.405 hectare |
| 1 square foot | = 929.030 square centimeters |
| 1 square inch | = 6.4516 square centimeters |
| 1 square mile | = 258.999 hectares |
| 1 square yard | = 0.836 square meter |
| 1 are | = 119.599 square yards |
| | = 0.025 acre |
| 1 hectare | = 2.471 acres |
| 1 square centimeter | = 0.155 square inch |
| 1 square decimeter | = 15.500 square inches |
| 1 square kilometer | = 247.105 acres |
| | = 0.386 square mile |
| 1 square meter | = 10.764 square feet |
| | = 1.196 square yards |
| 1 square millimeter | = 0.002 square inch |

CAPACITY/VOLUME:

| | |
|---------------|----------------------------|
| 1 cubic foot | = 28.316 cubic decimeters |
| 1 cubic inch | = 16.387 cubic centimeters |
| 1 cubic yard | = 0.765 cubic meter |
| 1 U.S. gallon | = 128 fluid ounces |
| | = 231 cubic inches |
| | = 3.785 liters |

CHAPTER 13-THE METRIC SYSTEM

| | |
|--------------------|---------------------------------------|
| 1 gill | = 4 fluid ounces = 0.118 liter |
| 1 U.S. fluid ounce | = 29.574 milliliters |
| 1 U.S. fluid quart | = 57.75 cubic inches = 0.946 liter |
| 1 cubic centimeter | = 0.061 cubic inch |
| 1 cubic decimeter | = 61.024 cubic inches |
| 1 cubic meter | = 1.308 cubic yards |
| 1 cubic dekaliter | = 2.642 gallons |
| 1 hectoliter | = 26.418 gallons |
| 1 liter | = 61.024 cubic inches |
| 1 milliliter | = 0.061 cubic inch |

WEIGHT/MASS:

| | |
|--------------|--------------------|
| 1 carat | = 200 milligrams |
| 1 gram | = 0.035 ounce |
| 1 kilogram | = 2.205 pounds |
| 1 Metric Ton | = 2,204.623 pounds |
| 1 ounce | = 28.35 grams |
| 1 pound | = 0.453592 kg |

5. UNIT CONVERSIONS

| To Convert | Into | Multiply by |
|---------------------------|----------------|------------------------|
| Acre | Hectare | 0.4047 |
| Astronomical Unit | Kilometers | 1.495×10^8 |
| Bolt (U.S. cloth measure) | Meters | 36.576 |
| B.T.U. | Kilowatt Hours | 2.928×10^{-4} |
| B.T.U./Hr | Watts | 0.2931 |
| Centimeters | Kilometers | 1×10^{-5} |
| Centimeters | Meters | 1×10^{-2} |
| Centimeters | Millimeters | 10.00 |
| Centimeters | Feet | 3.281×10^{-2} |

CHAPTER 13 - THE METRIC SYSTEM

| | | |
|-------------------|---------------------|------------------------|
| Centimeters | Inches | 0.3937 |
| Cubic Centimeters | Cubic Inches | 0.06102 |
| Cubic Feet | Cubic Meters | 0.02832 |
| Feet | Centimeters | 30.48 |
| Feet | Meters | 0.3048 |
| Feet/Min | Centimeters/Sec | 0.5080 |
| Foot Pounds | Kilowatt Hours | 3.766×10^{-7} |
| Gallons | Liters | 3.785 |
| Grams | Ounces | 3.527×10^{-2} |
| Grams | Pounds | 2.205×10^{-3} |
| Hectares | Acres | 2.471 |
| Hectares | Square Feet | 1.076×10^5 |
| Horsepower | Kilowatts | 0.7457 |
| Horsepower | Watts | 745.7 |
| Inches | Centimeters | 2.54 |
| Kilograms | Pounds | 2.205 |
| Kilometers | Feet | 3,281.00 |
| Kilometers | Miles | 0.6214 |
| Kilometers/hour | Knots | 0.5396 |
| Kilowatts | Horsepower | 1.341 |
| Kilowatt Hours | B.T.U. | 3,413.00 |
| Liters | Cubic Inches | 61.02 |
| Liters | U.S. Liquid Gallons | 0.2642 |
| Liters | U.S. Liquid Pints | 2.113 |

CHAPTER 13-THE METRIC SYSTEM

| | | |
|---------------------------------|--------------------------------|------------------------|
| Meters | Feet | 3.281 |
| Meters | Miles (Nautical) | 5.396×10^{-4} |
| Meters | Miles (Statute) | 6.214×10^{-4} |
| Miles (Nautical) | Kilometers | 1.853 |
| Miles (Statute) | Kilometers | 1.609 |
| Millimeters | Inches | 3.937×10^{-2} |
| Ounces | Grams | 28.349527 |
| Pounds | Kilograms | 0.4536 |
| Pounds per Square Inch (PSI) | Grams per Square Centimeter | 70.31 |
| Quarts (Liquid) | Liters | 0.9463 |
| Tons (long) | Kilograms | 1,016.00 |
| Tons (short) | Kilograms | 907.1848 |
| Watts | B.T.U./hour | 3.4129 |
| Watts | Horsepower | 1.341×10^{-3} |
| Yards | Meters | 0.914 |

CHAPTER 14 - MISCELLANEA

ENERGY TERMS - CONVERSIONS - HEAT VALUES

A. PREFIXES

Prefixes indicate orders of magnitude in steps. Prefixes provide a convenient way to express large and small numbers and to eliminate nonsignificant digits and leading zeros in fractions. The following are the more commonly used decimal prefixes:

| English System | |
|----------------|------------|
| Symbol | Represents |
| C | 100 |
| M | 1,000 |
| MM | 1,000,000 |

| Metric System | |
|---------------|-----------|
| K | 1,000 |
| M* | 1,000,000 |

*Caution: make certain of the system.

B. NATURAL GAS ENERGY MEASUREMENTS IN BTU

| | | |
|-----------------------|---|------------------------------|
| 1,000 | = | 1 Cubic Foot |
| 100,000 | = | 1 therm or 1 CCF or 100 MBTU |
| 1,000,000 | = | 1 MMBTU or 1,000 CF or 1 MCF |
| 1,000,000,000 | = | 1 billion BTU or 1 MMCF |
| 1,000,000,000,000 | = | 1 trillion BTU or 1 BCF |
| 1,000,000,000,000,000 | = | 1 quadrillion BTU or 1 TCF |

C. ENERGY MEASUREMENTS OF OTHER FUELS IN BTU

| | | | |
|----------------|----------------|------------------|--------------------------|
| Propane | 2,550 btu | = | 1 CF |
| | 21,650 btu | = | 1 Lb. |
| | 91,800 btu | = | 1 Gal. |
| Butane | 3,200 btu | = | 1 CF |
| | 21,500 btu | = | 1 Lb. |
| | 102,400 btu | = | 1 Gal. |
| Oil | 5,825,000 btus | = | 1 barrel crude |
| | 6,200,000 btus | = | 1 barrel #5 (Residual) |
| | 6,400,000 btus | = | 1 barrel #6 (heating) |
| | 134,000 btus | = | 1 Gal. #1 oil (kerosene) |
| | 139,000 btus | = | 1 Gal. #2 oil |
| | 146,800 btus | = | 1 Gal. #4 oil |
| | 150,000 btus | = | 1 Gal. #6 oil |
| (1 U.S. Barrel | = | 42 U.S. Gallons) | |

CHAPTER 14-MISCELLANEA

| | | | |
|----------|-----------------|---|-------------------------|
| Electric | 3,412 btus | = | 1 Kilowatt |
| Coal | 21,700,000 btus | = | 1 Ton Bituminous (soft) |
| | 22,800,000 btus | = | 1 Ton Anthracite (hard) |
| Wood | 3,500 btus | = | 1 Lb. (mixed average) |
| | 14,000,000 btus | = | 1 Cord (mixed average) |

D. QUICK FUEL COMPARISONS BASED ON 1,000,000 BTUS

| | | |
|------------------|---|--------------------|
| Natural Gas | = | 10 Therms |
| Propane Gas | = | 10.89 Gal. |
| Butane Gas | = | 9.77 Gal. |
| #1 Oil | = | 7.46 Gal. |
| #2 Oil | = | 7.19 Gal. |
| #4 Oil | = | 6.81 Gal. |
| #6 Oil | = | 6.67 Gal. |
| Bituminous Coal | = | .046 Ton (92 Lbs.) |
| Anthracite Coal | = | .044 Ton (88 Lbs.) |
| Wood (mixed avg) | = | 286 Lbs. |

E. SPECIFIC HEATING VALUE OF WOODS

(Based on U.S. Dept. of Agriculture Bulletin No. 753)

| | Weight per cord. lb. | | Heating Value BTU per lb. | | Equivalent lb. of Coal of 13,500 BTU per lb. | |
|---------------|----------------------|---------|---------------------------|---------|--|---------|
| | Green | Air-dry | Green | Air-dry | Green | Air-Dry |
| Ash-white | 4300 | 3800 | 4628 | 5395 | 0.343 | 0.400 |
| Beech | 5000 | 3900 | 3940 | 5359 | 0.292 | 0.397 |
| Birch, yellow | 5100 | 4000 | 3804 | 5225 | 0.282 | 0.387 |
| Chestnut | 4900 | 2700 | 2633 | 5778 | 0.195 | 0.428 |
| Cottonwood | 4200 | 2500 | 3024 | 6000 | 0.224 | 0.444 |
| Elm, white | 4400 | 3100 | 3591 | 5710 | 0.266 | 0.423 |
| Hickory | 5700 | 4600 | 4053 | 5391 | 0.300 | 0.399 |
| Maple, sugar | 5000 | 3900 | 4080 | 5590 | 0.302 | 0.414 |
| Maple, red | 4700 | 3200 | 3745 | 5969 | 0.277 | 0.442 |
| Oak, red | 5800 | 3900 | 3379 | 5564 | 0.250 | 0.412 |
| Oak, white | 5600 | 4300 | 3972 | 5558 | 0.294 | 0.412 |
| Pine, yellow | 3100 | 2300 | 7079 | 9174 | 0.526 | 0.680 |
| Pine, white | 3300 | 2200 | 4226 | 5864 | 0.313 | 0.434 |
| Walnut, black | 5100 | 4000 | 4078 | 4650 | 0.302 | 0.344 |
| Willow | 4600 | 2300 | 2370 | 5870 | 0.176 | 0.435 |

A cord of Wood is a pile 4' x 4' x 8' = 128 cu. ft. comprising approximately 56% Solid wood and 44% Interstitial spaces.

CHAPTER 14-MISCELLANEA

MEASURING GAS & OIL INPUT

Gas on pages 85 and 86 - Oil on page 87

1. GAS RATE - CUBIC FEET PER HOUR

| Seconds for One Revolution | Size of Test Dial | | Seconds for One Revolution | Size of Test Dial | |
|----------------------------------|-------------------|-----------|----------------------------------|-------------------|-----------|
| | 2 cu. ft. | 5 cu. ft. | | 2 cu. ft. | 5 cu. ft. |
| 10 | 720 | 1800 | 55 | 131 | 327 |
| 11 | 655 | 1636 | 56 | 129 | 321 |
| 12 | 600 | 1500 | 57 | 126 | 316 |
| 13 | 555 | 1385 | 58 | 124 | 310 |
| 14 | 514 | 1286 | 59 | 122 | 305 |
| 15 | 480 | 1200 | 60 | 120 | 300 |
| 16 | 450 | 1125 | 62 | 116 | 290 |
| 17 | 424 | 1059 | 64 | 112 | 281 |
| 18 | 400 | 1000 | 66 | 109 | 273 |
| 19 | 379 | 947 | 68 | 106 | 265 |
| 20 | 360 | 900 | 70 | 103 | 257 |
| 21 | 343 | 857 | 72 | 100 | 250 |
| 22 | 327 | 818 | 74 | 97 | 243 |
| 23 | 313 | 783 | 76 | 95 | 237 |
| 24 | 300 | 750 | 78 | 92 | 231 |
| 25 | 288 | 720 | 80 | 90 | 225 |
| 26 | 277 | 692 | 82 | 88 | 220 |
| 27 | 267 | 667 | 84 | 86 | 214 |
| 28 | 257 | 643 | 86 | 84 | 209 |
| 29 | 248 | 621 | 88 | 82 | 205 |
| 30 | 240 | 600 | 90 | 80 | 200 |
| 31 | 232 | 581 | 92 | 78 | 196 |
| 32 | 225 | 563 | 94 | | 192 |
| 33 | 218 | 545 | 96 | 75 | 188 |
| 34 | 212 | 529 | 98 | | 184 |
| 35 | 206 | 514 | 100 | 72 | 180 |
| 36 | 200 | 500 | 102 | | 176 |
| 37 | 195 | 486 | 104 | 69 | 173 |
| 38 | 189 | 474 | 106 | | 170 |
| 39 | 185 | 462 | 108 | 67 | 167 |
| 40 | 180 | 450 | 110 | | 164 |
| 41 | 176 | 439 | 112 | 64 | 161 |
| 42 | 172 | 429 | 116 | 62 | 155 |
| 43 | 167 | 419 | 120 | 60 | 150 |
| 44 | 164 | 409 | 125 | | 144 |
| 45 | 160 | 400 | 130 | | 138 |
| 46 | 157 | 391 | 135 | | 132 |
| 47 | 153 | 383 | 140 | | 129 |
| 48 | 150 | 375 | 145 | | 124 |
| 49 | 147 | 367 | 150 | | 120 |
| 50 | 144 | 360 | 155 | | 116 |
| 51 | 141 | 353 | 160 | | 113 |
| 52 | 138 | 346 | 165 | | 109 |
| 53 | 136 | 340 | 170 | | 106 |
| 54 | 133 | 333 | 175 | | 103 |
| 55 | 131 | 327 | 180 | | 100 |

CHAPTER 14-MISCELLANEA

| Seconds for One Revolution | Size of Test Dial | | Seconds for One Revolution | Size of Test Dial | |
|----------------------------------|-------------------|-----------|----------------------------------|-------------------|-----------|
| | 1/2 cu. ft. | 1 cu. ft. | | 1/2 cu. ft. | 1 cu. ft. |
| 10 | 180 | 360 | 35 | 103 | |
| 11 | 164 | 327 | 36 | 50 | 100 |
| 12 | 150 | 300 | 37 | 97 | |
| 13 | 138 | 277 | 38 | 47 | 95 |
| 14 | 129 | 257 | 39 | 92 | |
| 15 | 120 | 240 | 40 | 45 | 90 |
| 16 | 113 | 225 | 41 | | |
| 17 | 106 | 212 | 42 | 43 | 86 |
| 18 | 100 | 200 | 43 | | |
| 19 | 95 | 189 | 44 | 41 | 82 |
| 20 | 90 | 180 | 45 | 40 | 80 |
| 21 | 86 | 171 | 46 | 78 | |
| 22 | 82 | 164 | 47 | 38 | |
| 23 | 78 | 157 | 48 | 75 | |
| 24 | 75 | 150 | 49 | | |
| 25 | 72 | 144 | 50 | 36 | 72 |
| 26 | 69 | 138 | 51 | | |
| 27 | 67 | 133 | 52 | | 69 |
| 28 | 64 | 129 | 53 | 34 | |
| 29 | 62 | 124 | 54 | | 67 |
| 30 | 60 | 120 | 55 | | |
| 31 | | 116 | 56 | 32 | 64 |
| 32 | 56 | 113 | 57 | | |
| 33 | | 109 | 58 | 31 | 62 |
| 34 | 53 | 106 | 59 | | |
| 35 | | 103 | 60 | 30 | 60 |

CHAPTER 14-MISCELLANEA

2. OIL RATE -- GALLON PER HOUR

Nozzle Delivery Rates at Various Pressures for No. 2 Fuel Oil

Pump Pressure (PSI)

| | 80 | 90 | 100 | 125 | 150 | 175 | 200 | 225 | 250 | 275 | 300 |
|------|------|------|------|------|------|------|------|------|------|------|------|
| 1.0 | 0.9 | .95 | 1.0 | 1.1 | 1.2 | 1.3 | 1.4 | 1.5 | 1.6 | 1.7 | 1.75 |
| 1.5 | 1.3 | 1.4 | 1.5 | 1.7 | 1.8 | 2.0 | 2.1 | 2.3 | 2.4 | 2.5 | 2.6 |
| 2.0 | 1.8 | 1.9 | 2.0 | 2.2 | 2.5 | 2.6 | 2.9 | 3.0 | 3.2 | 3.3 | 3.5 |
| 2.5 | 2.2 | 2.4 | 2.5 | 2.8 | 3.1 | 3.3 | 3.5 | 3.8 | 4.0 | 4.1 | 4.3 |
| 3.0 | 2.7 | 2.8 | 3.0 | 3.3 | 3.7 | 3.9 | 4.2 | 4.5 | 4.7 | 5.0 | 5.2 |
| 3.5 | 3.1 | 3.3 | 3.5 | 3.9 | 4.3 | 4.6 | 5.0 | 5.3 | 5.5 | 5.8 | 6.0 |
| 4.0 | 3.6 | 3.8 | 4.0 | 4.5 | 4.9 | 5.3 | 5.6 | 6.0 | 6.3 | 6.6 | 6.9 |
| 4.5 | 4.0 | 4.2 | 4.5 | 5.0 | 5.5 | 5.9 | 6.3 | 6.8 | 7.1 | 7.4 | 7.8 |
| 5.0 | 4.5 | 4.7 | 5.0 | 5.6 | 6.1 | 6.6 | 7.1 | 7.5 | 7.9 | 8.3 | 8.7 |
| 5.5 | 4.9 | 5.2 | 5.5 | 6.2 | 6.7 | 7.3 | 7.8 | 8.3 | 8.7 | 9.1 | 9.5 |
| 6.0 | 5.3 | 5.6 | 6.0 | 6.7 | 7.3 | 7.9 | 8.5 | 9.0 | 9.5 | 9.9 | 10.4 |
| 7.0 | 6.2 | 6.6 | 7.0 | 7.8 | 8.5 | 9.2 | 9.9 | 10.5 | 11.0 | 11.6 | 12.0 |
| 8.0 | 7.2 | 7.5 | 8.0 | 8.9 | 9.8 | 10.5 | 11.3 | 12.0 | 12.6 | 13.2 | 13.8 |
| 9.0 | 8.0 | 8.5 | 9.0 | 10.0 | 11.0 | 11.8 | 12.6 | 13.5 | 14.2 | 14.8 | 15.6 |
| 10.0 | 8.4 | 9.4 | 10.0 | 11.2 | 12.2 | 13.2 | 14.1 | 15.0 | 15.8 | 16.6 | 17.3 |
| 11.0 | 9.8 | 10.3 | 11.0 | 12.3 | 13.4 | 14.5 | 15.6 | 16.5 | 17.4 | 18.2 | 19.0 |
| 12.0 | 10.7 | 11.2 | 12.0 | 13.4 | 14.6 | 15.8 | 16.9 | 18.0 | 18.9 | 19.8 | 20.7 |
| 13.0 | 11.6 | 12.2 | 13.0 | 14.5 | 16.0 | 17.2 | 18.4 | 19.5 | 20.6 | 21.5 | 22.5 |
| 14.0 | 12.4 | 13.1 | 14.0 | 15.6 | 17.0 | 18.4 | 19.8 | 21.0 | 22.0 | 23.1 | 24.0 |
| 15.0 | 13.3 | 14.0 | 15.0 | 16.8 | 18.4 | 19.8 | 21.3 | 22.5 | 23.6 | 24.8 | 25.9 |
| 16.0 | 14.3 | 15.0 | 16.0 | 17.8 | 19.5 | 21.0 | 22.6 | 24.0 | 25.3 | 26.4 | 27.7 |
| 17.0 | 15.1 | 15.9 | 17.0 | 19.0 | 20.7 | 22.5 | 24.1 | 25.5 | 26.9 | 28.0 | 29.3 |
| 18.0 | 16.1 | 17.0 | 18.0 | 20.1 | 22.0 | 23.7 | 25.3 | 27.0 | 28.4 | 29.7 | 31.2 |
| 20.0 | 17.8 | 18.8 | 20.0 | 22.3 | 24.4 | 26.4 | 28.2 | 30.0 | 31.7 | 33.1 | 34.6 |
| 22.0 | 19.6 | 20.6 | 22.0 | 24.6 | 26.9 | 29.0 | 31.2 | 33.0 | 34.8 | 36.4 | 38.0 |

Notes: These delivery rates are approximate values.

Actual rates will vary slightly between different nozzles of the same nominal rating.

Delivery tends to increase with higher viscosity, or lower oil temperature, or lower specific gravity.

CHAPTER 14-MISCELLANEA

ACIDITY AND ALKALINITY - (pH)

pH is the logarithm of the reciprocal of the Hydrogen concentration.

$$\text{pH} = \log_{10} \frac{1}{\text{H}^+} \quad (\text{gram/liter equiv.})$$

Since 1 hydroxyl ion is formed whenever a hydrogen ion develops, the pH scale also is used to measure the formation of hydroxyl ions.

Typical pH values, relative strength:

| | | |
|----|-----------|--------------------------|
| 1 | 1,000,000 | 1/10 normal HCl sol |
| 2 | 100,000 | Gastric Fluid |
| 3 | 10,000 | Vinegar (4% acetic acid) |
| 4 | 1,000 | Orange or grape juice |
| 5 | 100 | Molasses |
| 6 | 10 | Milk (6.8 pH) |
| 7 | 1 | Pure Water, saliva |
| 8 | 10 | Baking Soda, sea water |
| 9 | 100 | Borax solution |
| 10 | 1,000 | Soap - Milk of magnesia |
| 11 | 10,000 | Tri-sodium phosphate |
| 12 | 100,000 | Household ammonia (10%) |
| 13 | 1,000,000 | Lye - Na O H |

Manufacturers of hydronic and air conditioning equipment recommend a pH value of 7.5 to 8.5 in the circulating water.

The pH levels in all industrial cooling and heating systems are closely supervised. Excessive acidity (pH 5-6) causes corrosion and high alkalinity (pH 8-10) results in heavy scale deposits on hot heat transfer surfaces.

SOUND LEVELS

The human ear can accept an enormous range of sound levels. From a whisper (20 dB) to a police siren at 50 ft. (120dB) represents a sound pressure level ratio of 1 to 1 million. The ear can take it for short periods, but prolonged exposure to high noise levels will cause permanent hearing impairment.

Because of the very large audio level ratios and resulting cumbersome numbers, the decibel system was devised.

Honoring Alexander Graham Bell, the telephone industry named the unit of sound the Bel. This unit was found to be too large for convenience, so 1/10 Bel or the decibel (dB) was adopted internationally.

CHAPTER 14-MISCELLANEA

TYPICAL SOUND PRESSURE LEVELS

| <u>Db</u> | <u>Noise Source and Distance</u> |
|-----------|----------------------------------|
| 130 | Jet plane take-off at 200 ft. |
| 120 | Police siren at 50 ft. |
| 110 | Pneumatic hammer at 50 ft. |
| 100 | Millworking machines at 50 ft. |
| 90 | Manufacturing plants, average |
| 80 | Symphony orchestra, fortissimo |
| 70 | Vacuum cleaner at 10 ft. |
| 60 | Business machine areas |
| 50 | Normal business offices |
| 40 | Quiet residential areas at night |
| 30 | Normal speech at 3 ft. |
| 20 | Soft whisper at 3 ft. |

Doubling of sound pressure from a single source results in a 3 dB rise in sound pressure level.

Doubling of the distance from the source of a sound to the receiver will reduce its level 6 dB.

A 10 dB rise in sound level indicates that the sound perceived by the ear will be twice as loud as the original sound.

A police siren developing 100 dB at 100 ft. will produce 94 dB at 200 ft.; 88 dB at 400 ft., etc.

GAS PIPE SIZING - RESIDENTIAL

Interior gas piping must be examined before installing a new gas boiler. A typical winter load and even a summer load (indirect hot water supply) must be considered. Listed below are average inputs of various gas appliances that may be on-line. Also listed is a chart to determine whether the existing piping can carry the seasonal load.

For specific appliances or appliances not shown below, the input should be determined from the manufacturer's rating.

CHAPTER 14-MISCELLANEA

Approximate Gas Input for Some Common Appliances

| Appliance | Input BTU Per Hr. (Approx.) |
|--|--------------------------------|
| Range, Free Standing, Domestic | 65,000 |
| Built-In Oven or Broiler Unit, Domestic | 25,000 |
| Built-In Top Unit, Domestic | 40,000 |
| Water Heater, Automatic Storage - 30 to 40 Gal. Tank | 45,000 |
| Water Heater, Automatic Storage - 50 Gal. Tank | 55,000 |
| Fireplace/Gas Log | 21,000 - 55,000 |
| Grille | 30,000 - 40,000 |
| Boiler | ---- |
| Clothes Dryer, Type 1 (Domestic) | 20,000 - 35,000 |
| Gas Light | 2,500 |
| Incinerator, Domestic | 35,000 |

Maximum Capacity of Pipe in Cubic Feet of Gas Per Hour

(Based on a pressure drop of 0.3 Inch Water Column and 0.6 Specific Gravity Gas)

| Length In Feet | Nominal Iron Pipe Size, Inches | | | | | | | | |
|----------------------|--------------------------------|-----|-----|-------|-------|-------|-------|-------|--------|
| | 1/2 | 3/4 | 1 | 1-1/4 | 1-1/2 | 2 | 2-1/2 | 3 | 4 |
| 10 | 132 | 278 | 520 | 1,050 | 1,600 | 3,050 | 4,800 | 8,500 | 17,500 |
| 20 | 92 | 190 | 350 | 730 | 1,100 | 2,100 | 3,300 | 5,900 | 12,000 |
| 30 | 73 | 152 | 285 | 590 | 890 | 1,650 | 2,700 | 4,700 | 9,700 |
| 40 | 63 | 130 | 245 | 500 | 760 | 1,450 | 2,300 | 4,100 | 8,300 |
| 50 | 56 | 115 | 215 | 440 | 670 | 1,270 | 2,000 | 3,600 | 7,400 |
| 60 | 50 | 105 | 195 | 400 | 610 | 1,150 | 1,850 | 3,250 | 6,800 |
| 70 | 46 | 96 | 180 | 370 | 560 | 1,050 | 1,700 | 3,000 | 6,200 |
| 80 | 43 | 90 | 170 | 350 | 530 | 990 | 1,600 | 2,800 | 5,800 |
| 30 | 40 | 84 | 160 | 320 | 490 | 930 | 1,500 | 2,600 | 5,400 |
| 100 | 38 | 79 | 150 | 305 | 460 | 870 | 1,400 | 2,500 | 5,100 |
| 125 | 34 | 72 | 130 | 275 | 410 | 780 | 1,250 | 2,200 | 4,500 |
| 150 | 31 | 64 | 120 | 250 | 380 | 710 | 1,130 | 2,000 | 4,100 |
| 175 | 28 | 59 | 110 | 225 | 350 | 650 | 1,050 | 1,850 | 3,800 |
| 200 | 26 | 55 | 100 | 210 | 320 | 610 | 980 | 1,700 | 3,500 |

Correction Factors for Specific Gravity Other Than 0.60

CHAPTER 14-MISCELLANEA

ABBREVIATIONS USED IN HEATING

| | | | |
|------------------------------|----------|--|-------|
| Absolute | abs | Gallons per Second | gps |
| Alternating-Current | a-c | Gram | g |
| Ampere | amp | Horsepower | hp |
| Atmosphere | atm | Horsepower-Hour | hp-hr |
| Average | avg | Hour | hr |
| Avoirdupois | avdp | Inch | in |
| Barometer | bar | Inch-Pound | in-lb |
| Boiling Point | bp | Kilogram | kg |
| Brake Horsepower | bhp | Kilowatt | kw |
| Brake Horsepower, Hour | bhp-hr | Melting Point | mp |
| British Thermal Unit | Btu | Meter | m |
| per hour | Btuh | Miles per Hour | mph |
| Calorie | cal | Millimeter | mm |
| Centigram | cg | Minute | min |
| Centimeter | cm | Ounce | oz |
| Cubic | cu | Pound | lb |
| Cubic Centimeter | cc | Pounds per Square Inch | psi |
| Cubic Foot | cu ft | Pounds per Square Inch, Gage | psig |
| Cubic Feet per Minute | cfm | Pounds per Square Inch, Absolute | psia |
| Cubic Feet per Second | cfs | Revolutions per Minute | rpm |
| Degree | deg or ° | Revolutions per Second. | rps |
| Degree, Centigrade | C | Second | sec |
| Degree, Fahrenheit | F | Specific Gravity | sp gr |
| Diameter | diam | Specific Heat | sp ht |
| Direct-Current | d - c | Square Foot | sq ft |
| Feet per Minute | fpm | Square Inch | sq in |
| Feet per Second | fps | Volt | v |
| Foot | ft | Watt | w |
| Foot-Pound | ft lb | Watt Hour | whr |
| Freezing Point | fp | | |
| Gallon | gal | | |
| Gallons per Minute | gpm | | |

CHAPTER 14-MISCELLANEA

CAPACITY OF ROUND STORAGE TANKS

Number of Gallons

| Depth or Length in Ft. | Inside Diameter in Inches | | | | | | | | | |
|---------------------------------|---------------------------|------|------|------|------|------|------|-------|-------|-------|
| | 18 | 24 | 30 | 36 | 42 | 48 | 54 | 60 | 66 | 72 |
| 1 | 1.10 | 1.96 | 3.06 | 4.41 | 5.99 | 7.83 | 9.91 | 12.24 | 14.81 | 17.62 |
| 2 | 26 | 47 | 73 | 105 | 144 | 188 | 238 | 294 | 356 | 423 |
| 2-1/2 | 33 | 59 | 91 | 131 | 180 | 235 | 298 | 367 | 445 | 530 |
| 3 | 40 | 71 | 100 | 158 | 216 | 282 | 357 | 440 | 534 | 635 |
| 3-1/2 | 46 | 83 | 129 | 184 | 252 | 329 | 416 | 513 | 623 | 740 |
| 4 | 53 | 95 | 147 | 210 | 288 | 376 | 475 | 586 | 712 | 846 |
| 4-1/2 | 59 | 107 | 165 | 238 | 324 | 423 | 534 | 660 | 800 | 952 |
| 5 | 66 | 119 | 181 | 264 | 360 | 470 | 596 | 734 | 899 | 1057 |
| 5-1/2 | 73 | 130 | 201 | 290 | 396 | 517 | 655 | 808 | 978 | 1163 |
| 6 | 79 | 141 | 219 | 315 | 432 | 564 | 714 | 880 | 1066 | 1268 |
| 6-1/2 | 88 | 155 | 236 | 340 | 468 | 611 | 770 | 954 | 1156 | 1374 |
| 7 | 92 | 165 | 255 | 368 | 504 | 658 | 832 | 1028 | 1244 | 1480 |
| 7-1/2 | 99 | 179 | 278 | 396 | 540 | 705 | 889 | 1101 | 1335 | 1586 |
| 8 | 106 | 190 | 291 | 423 | 576 | 752 | 949 | 1175 | 1424 | 1691 |
| 9 | 119 | 212 | 330 | 476 | 648 | 846 | 1071 | 1322 | 1599 | 1903 |
| 10 | 132 | 236 | 366 | 529 | 720 | 940 | 1189 | 1463 | 1780 | 2114 |
| 12 | 157 | 282 | 440 | 634 | 864 | 1128 | 1428 | 1762 | 2133 | 2537 |
| 14 | 185 | 329 | 514 | 740 | 1008 | 1316 | 1666 | 2056 | 2490 | 2960 |
| 16 | 211 | 376 | 587 | 846 | 1152 | 1504 | 1904 | 2350 | 2844 | 3383 |
| 18 | 238 | 423 | 660 | 952 | 1296 | 1692 | 2140 | 2640 | 3200 | 3806 |
| 20 | 264 | 470 | 734 | 1057 | 1440 | 1880 | 2380 | 2932 | 3556 | 4230 |

CAPACITY OF RECTANGULAR TANKS

To find the capacity in U.S. gallons of rectangular tanks, reduce all dimensions to inches, then multiply the length by the width by the height and divide the product by 231.

Example: Tank 56" long x 32" wide x 20" deep

Then 56" x 32" x 20" = 35,840 cu. in.

35,840 ÷ 231 = 155 gallon capacity

CHAPTER 14-MISCELLANEA

GRAINS OF MOISTURE PER CUBIC FOOT OF AIR AT VARIOUS TEMPERATURES AND HUMIDITIES

| Temp. °F. | Relative Humidity, Percent | | | | | | | | | |
|--------------|----------------------------|------|------|------|------|------|------|------|------|------|
| | 100% | 90% | 80% | 70% | 60% | 50% | 40% | 30% | 20% | 10% |
| 75 | 9.35 | 8.42 | 7.49 | 6.55 | 5.61 | 4.68 | 3.74 | 2.81 | 1.87 | 0.94 |
| 72 | 8.51 | 7.66 | 6.81 | 5.96 | 5.11 | 4.25 | 3.40 | 2.55 | 1.70 | 0.85 |
| 70 | 7.98 | 7.18 | 6.38 | 5.59 | 4.79 | 3.99 | 3.19 | 2.39 | 1.60 | 0.80 |
| 67 | 7.24 | 6.52 | 5.79 | 5.07 | 4.35 | 3.62 | 2.90 | 2.17 | 1.45 | 0.72 |
| 65 | 6.78 | 6.10 | 5.43 | 4.75 | 4.07 | 3.39 | 2.71 | 2.04 | 1.36 | 0.68 |
| 60 | 5.74 | 5.17 | 4.60 | 4.02 | 3.45 | 2.87 | 2.30 | 1.72 | 1.15 | 0.57 |
| 50 | 4.08 | 3.67 | 3.26 | 2.85 | 2.45 | 2.04 | 1.63 | 1.22 | 0.82 | 0.41 |
| 40 | 2.85 | 2.56 | 2.28 | 1.99 | 1.71 | 1.42 | 1.14 | 0.86 | 0.57 | 0.29 |
| 30 | 1.94 | 1.74 | 1.55 | 1.35 | 1.16 | 0.97 | 0.78 | 0.58 | 0.39 | 0.19 |
| 20 | 1.23 | 1.11 | 0.99 | 0.86 | 0.74 | 0.62 | 0.49 | 0.37 | 0.25 | 0.12 |
| 10 | 0.78 | 0.70 | 0.62 | 0.54 | 0.47 | 0.39 | 0.31 | 0.23 | 0.16 | 0.08 |
| 0 | 0.48 | 0.43 | 0.39 | 0.34 | 0.29 | 0.24 | 0.19 | 0.14 | 0.10 | 0.05 |

7000 Grains of moisture = 1 pound of water

HEATING CONVERSION FACTORS

| | | | | |
|----------|---|----------|---|----------|
| BTUs | X | 1054.8 | = | Joules |
| BTUs | X | 0.2520 | = | Kg. Cal. |
| Calories | X | 4.186 | = | Joules |
| Joules | X | 0.23889 | = | Calories |
| Joules | X | .0002389 | = | Kg. Cal. |
| Kg. Cal. | X | 3.9685 | = | BTUs |
| KW-hr | X | 860.01 | = | Kg. Cal. |
| KW-hr | X | 3413.0 | = | BTUs |
| Watt-hr | X | 3.413 | = | BTUs |

CHAPTER 14-MISCELLANEA

QUANTITY OF HEAT IN BTU REQUIRED TO RAISE 1 CU. FT. OF AIR THROUGH A GIVEN TEMPERATURE INTERVAL

| External Temp. Degrees F | Temperature of Air in Room | | | | | | | |
|--------------------------|----------------------------|-------|-------|-------|-------|-------|-------|-------|
| | 40 | 50 | 60 | 70 | 80 | 90 | 100 | 110 |
| -40 | 1.802 | 2.027 | 2.252 | 2.479 | 2.703 | 2.928 | 3.154 | 3.379 |
| -30 | 1.540 | 1.760 | 1.980 | 2.200 | 2.420 | 2.640 | 2.860 | 3.080 |
| -20 | 1.290 | 1.505 | 1.720 | 1.935 | 2.150 | 2.365 | 2.580 | 2.795 |
| -10 | 1.051 | 1.262 | 1.473 | 1.684 | 1.892 | 2.102 | 2.311 | 2.522 |
| 0 | 0.822 | 1.028 | 1.234 | 1.439 | 1.645 | 1.851 | 2.056 | 2.262 |
| 10 | 0.604 | 0.805 | 1.007 | 1.208 | 1.409 | 1.611 | 1.812 | 2.013 |
| 20 | 0.393 | 0.590 | 0.787 | 0.984 | 1.181 | 1.378 | 1.575 | 1.771 |
| 30 | 0.192 | 0.385 | 0.578 | 0.770 | 0.963 | 1.155 | 1.345 | 1.540 |
| 40 | — | 0.188 | 0.376 | 0.564 | 0.752 | 0.940 | 1.128 | 1.316 |
| 50 | — | — | 0.184 | 0.367 | 0.551 | 0.735 | 0.918 | 1.102 |
| 60 | — | — | — | 0.179 | 0.359 | 0.538 | 0.718 | 0.897 |
| 70 | — | — | — | — | 0.175 | 0.350 | 0.525 | 0.700 |

MELTING POINTS OF METALS

| | Degrees F. | | Degrees F. |
|-------------------|------------|-------------------|------------|
| Aluminum | 1220 | Iron (Cast) Gray | 2460-2550 |
| Antimony | 1167 | Iron (Cast) White | 1920-2010 |
| Bismuth | 520 | Iron, Wrought | 2460-2640 |
| Brass (Red) | 1870 | Lead | 622 |
| Bronze | 1900 | Silver (Pure) | 1761 |
| Copper | 1981 | Steel | 2370-2550 |
| Glass | 2377 | Tin | 449 |
| Gold (Pure) | 1945 | Zinc | 787 |
| Solder (Lead-Tin) | 250-570 | | |

CHAPTER 14-MISCELLANEA

CONVERSION FACTORS

WATER

| | | | | |
|-----------------------------|---|----------|---|--------------|
| U.S. Gallons | x | 8.34 | = | Pounds |
| U.S. Gallons | x | 0.13368 | = | Cubic Feet |
| U.S. Gallons | x | 231.00 | = | Cubic Inches |
| U.S. Gallons | x | 3.78 | = | Liters |
| Imperial Gallons | x | 277.3 | = | Cubic Inches |
| Imperial Gallons at 62°F | = | 10.0 | = | Pounds |
| Cubic In. of Water (39.2°) | x | 0.036130 | = | Pounds |
| Cubic In. of Water (39.2°) | x | 0.004329 | = | U.S. Gallons |
| Cubic In. of Water (39.2°) | x | 0.576384 | = | Ounces |
| Cubic Feet of Water (39.2°) | x | 62.427 | = | Pounds |
| Cubic Feet of Water (39.2°) | x | 7.48 | = | U.S. Gallons |
| Cubic Feet of Water (39.2°) | x | 0.028 | = | Tons |
| Pounds of Water | x | 27.72 | = | Cubic Inches |
| Pounds of Water | x | 0.01602 | = | Cubic Feet |
| Pounds of Water | x | 0.12 | = | U.S. Gallons |

PRESSURE

| | | |
|---|---|---------------------------------------|
| 1 Pound Per Square Inch | = | 144 Pounds Per Square Foot |
| | | 2.0355 Inches of Mercury at 32°F. |
| | | 2.0416 Inches of Mercury at 62°F. |
| | | 2.309 Feet of Water at 62°F. |
| | | 27.71 Inches of Water at 62°F |
| | | 6.895 kPA (kilopascal) |
| 1 Ounce Per Square Inch | = | 0.1276 Inches of Mercury at 62°F. |
| | | 1.732 Inches of Water at 62°F. |
| 1 Atmosphere (14.7 Lbs. Per Sq. In.) | = | 2116.3 Pounds Per Square Foot |
| | | 33.947 Feet of Water at 62°F. |
| | | 30 Inches of Mercury at 62°F. |
| | | 29.922 Inches of Mercury at 32°F. |
| | | 760 Millimeters of Mercury at 32°F. |
| | | 101.3 kilopascal |
| 1 Inch Water (at 62°F.) | = | 0.03609 Lbs. or 0.5774 oz Per Sq. In. |
| | | 5.196 Pounds Per Square Foot |
| | | 0.248 kilopascal |
| 1 Foot Water (at 62°F.) | = | 0.433 Pounds Per Square Inch |
| | | 62.355 Pounds Per Square Foot |
| 1 Inch Mercury (at 62°F.) | = | 0.491 Lbs. or 7.86 oz. Per Sq. In |
| | | 1.132 Feet Water at 62°F. |
| | | 13.58 Inches Water at 62°F. |

CHAPTER 14-MISCELLANEA

EQUIVALENT VALUE IN DIFFERENT UNITS

1 H.P. = 746 watts
.746 K.W.
33,000 ft.-lbs. per minute
550 ft.-lbs. per second
2.64 lbs water evaporated per hour from and at
212°F.

1 H.P. Hour = 746 K.W. hours
2.64 lbs water evaporated per hour from and at
212°F.
17.0 lbs. water raised from 62° to 212°F.

1 Kilowatt = 1,000 watts
1.34 H.P.
3.53 lbs. water evaporated per hour from and at
212°F.

1 Watt = .00134 H.P.
.0035 lb. water evaporated per hour

1 K.W. Hour = 1,000 watt hours
1.34 H.P. Hours
3,600,000 joules
3.53 lbs. water evaporated from and at 212°F.
22.75 lbs. of water raised from 62°F to 212°F

1 Joule = 1 watt second
.000000278 K.W. hour

MJ = Megajoule = 1,000,000 Joule = 948 BTU
239 Kcal

EQUIVALENTS OF ELECTRICAL UNITS

1 Kilowatt = 1.34 H.P.
0.955 BTU per second
57.3 BTU per minute
3438 BTU per hour

1 Horse Power = 746 watts
42.746 BTU per minute
2564.76 BTU per hour

1 BTU = 17.452 watt minutes
0.2909 watt hour

CHAPTER 14-MISCELLANEA

STANDARD SYMBOLS IN HEATING SPECIFICATIONS

| | |
|------------------|---|
| D (delta) | = differential, difference in, change in, increment |
| DP | = pressure drop, differential, or loss usually in psi |
| DT | = difference, change or drop in temperature |
| P (pi) | = 3.141592654 = ratio of circumference of a circle to the diameter of that circle |
| S (sigma) | = sum, summation, total of |
| W (omega) | = ohm, unit of electric resistance = 1 V/A |
| ± | = plus or minus, tolerance |
| ≠ | = not equal to |
| < | = (is) less than |
| £ | = smaller than or equal to |
| > | = (is) greater than |
| ≥ | = greater than or equal to |
| << | = much smaller than |
| >> | = much larger than |
| T | = proportional to, similar to |
| ≈ | = approximately equal to |
| @ | = congruent to |
| \ | = therefore |
| ∠ | = angle |
| ∥ | = parallel to |
| ⊥ | = perpendicular to, at right angles to |
| ∞ | = infinity, infinite |

Series 2 Gas Boiler

- **Cast Iron Package**
- **Hot Water**
- **DOE Capacities 31 to 244 MBH**

The Series 2 hot water boiler combines the dependability of the a conventional proven atmospheric draft design with the economy of gas heat. It is completely factory assembled and fire tested and requires only minimal hookup to put into operation. At the heart of every Series 2 is a durable cast iron heat exchanger that enhances energy efficiency and improves boiler performance and has an added bonus in

the Lifetime True Blue Limited Warranty. Also, the sections are joined by push nipples that can expand and contract with the sections they join. The Series 2 is equipped with a fuel saving vent damper and achieves annual fuel utilization efficiencies up to 84% (Model 2h). Combined, the Series 2 is your long term investment in quality, dependable home heating. Certified for installation on combustible flooring. Do not install on carpeting.

Series 2PV Gas Boiler

- **Induced Draft**
- **Cast Iron Package**
- **Hot Water**
- **DOE Capacities 51 to 135 MBH**

The Series 2PV induced draft boiler is designed for new or replacement installations and electric heat conversions where a chimney is deteriorated or not available. It can be installed almost anywhere, such as a closet, utility room or garage. Available power vented directly through the wall. A cast iron heat exchanger, together with a draft inducing fan, provide for more heat with less fuel and eliminate heat loss through the

vent. No vent damper is required. Burnham also backs the heat exchanger on the Series 2PV with their Lifetime True-Blue Limited Warranty.

Spirit[®] Gas Boiler

- **Direct Vent**
- **Cast Iron Package**
- **Hot Water**
- **DOE Capacities 62 to 164 MBH**

The Spirit direct vent gas boiler is designed for new or replacement installations and electric heat conversions where a chimney is not available. It can be installed almost anywhere, such as in a closet, utility room or garage. Its sealed combustion, direct vent design helps regulate combustion air to help avoid operational and safety problems. A cast iron heat exchanger, together with a draft inducing fan, provide for more heat with less fuel and eliminate heat loss through the vent. No vent damper is required. Certified for installation on combustible flooring. Do not install on carpeting.

Independence[®]/Independence PV Gas Boiler

- **Cast Iron Package, Semi-Pak or Knockdown**
- **Steam or Hot Water**
- **DOE Capacities:**
 - Independence 51 to 317 MBH**
 - Independence PV 52 to 145 MBH**
- **Natural Draft or Power Vent Models Available**

The Independence and Independence PV gas-fired steam or hot water boilers offer the dependability and quality of cast iron construction. The Independence can be used for natural gas or LP gas (IN3-IN9). Burnham also offers the Independence as a power vented boiler for situations where a chimney doesn't exist or if an old chimney is deteriorated.

The Independence PV (steam only) induced draft boiler installs almost anywhere because no chimney is required for installation. Certified for installation on combustible flooring. Do not install on carpeting.

Revolution™ Gas Boiler

- Direct Vent
- Cast Iron Package
- Hot Water
- 87% AFUE
- DOE Capacities 55 to 160 MBH

As Burnham's most efficient gas-fired boiler, the Revolution is unmatched in versatility. Available in five sizes, the Revolution is easily combined with an indirect water heater and zoning controls

for total home comfort. And because of the variety of venting options, the Revolution is adaptable to the floorplan of virtually any home. Consider these features:

- Familiar and proven gas train and controls.
- No additional piping or accessories to run directly on radiant systems.
- Supply temperatures anywhere from 60 to 200 without worry of damage from condensation.
- A built-in injection pump to provide precise temperature control.
- A built-in primary/secondary loop to save time and simplify installation.
- All connections from top to allow for minimal clearances.

Minuteman® II Combination Boiler/Water Heater

- Cast Iron Construction
- Natural or LP Gas
- AFUE up to 83%
- Sealed Combustion
- 4 Sizes up to 140 MBH Input
- DOE Capacities 58-112 MBH

The Burnham Minuteman II provides heat and hot water "all in one." Because of its small footprint, it can be easily installed in confined areas or closets. The Minuteman II is ideal for all heating applications ranging from

single homes, multi-family units, to industrial or commercial applications. No chimney is necessary, which makes it the most versatile heating appliance for home comfort today.

Series 5B Gas Boiler

- **Cast Iron Knockdown Unit**
- **Steam or Hot Water**
- **Capacities 320 to 1,560 MBH**

The Series 5B boiler offers the capacities and features that make it ideal for commercial, institutional and high rise residential heating applications. The rear outlet draft hood is specifically suited to installations involving low ceiling heights.

The Burnham 5B is rated at 50 psi water or 15 psi steam. It has an 80% combustion efficiency. Plus, only Burnham boilers come with the exclusive Iron Nipple Seal of Dependability certifying the use of cast iron nipples which expand and contract with the sections to insure superior quality and reliable performance. Do not install on combustible flooring.

Series 8B Gas Boiler

- **Cast Iron Packaged or Knockdown Unit**
- **Hot Water**
- **Capacities 212 to 475 MBH**

The Burnham Series 8B gas boiler has a vertical flue design which provides for maximum heat transfer. Cast iron push nipples assure the integrity of the cast iron section assembly by expanding & contracting at the same rate and providing a water tight seal. The Series 8B

boilers can be installed 1" apart (subject to local codes) making them ideally suited for multiple boiler applications. Do not install on combustible flooring. A heat shield is required and available for combustible floor installations. Do not install on carpeting, even with a heat shield.

Series 8B Modular/Multiple System

- **Primary-Secondary Piping**
- **Capacities starting at 424 MBH**

The Burnham modular/multiple boiler system puts the efficiency, cost, and service advantages of compact gas boilers to work in space heating applications where a single commercial boiler would not be practical for reasons of budget or accessibility. The

Series 8B boilers can be installed 1" apart (subject to local codes), making them ideally suited for modular applications. And the modular/multiple system utilizes primary-secondary piping to maximize energy efficiency.

Modular/Multiple Control

Packages

- **Microprocessor-based**
- **User-programmable, User-friendly**

Burnham offers ten control packages for operating modular/multiple boiler systems. Control Package A and Penn Johnson A350 maintain a constant supply water temperature. Control Package B and Penn Johnson A350RN are the simplest system for step firing mod-

ules according to outdoor air temperature. Control packages D and E monitor supply water temperature and outdoor air temperature to step fire boiler modules for maximum fuel economy. Each has different levels of sophistication and added features. The Burnham microprocessor-based TMC panel senses supply water, return water, and outdoor air temperatures to automatically monitor and control the operation of up to eight boiler modules. The TMC Panel provides boiler status lights and digitally displays supply water, return water and outdoor air temperatures along with accumulated operating hours of each boiler. There are a number of programmable variables to allow accurate set-up for each installation, making the TMC Panel one of the most sophisticated controllers available. The tekmar 251, 252, 254 and 258 control offerings are also available.

V7 Series Oil Boiler

- **Cast Iron Package or Knockdown**
- **Hot Water or Steam**
- **DOE Capacities 68 to 299 MBH**

The fully factory-packaged Burnham V7 is available for steam or hot water with heating capacities ranging from 73 to 275 MBH. A knock-down model is available, as a water or steam unit, when access to the basement is confined.

The pinned heating surfaces of the V7's cast iron section assembly, together with a vertical flue design, extract maximum heat while maintaining low draft losses. This results in higher energy efficiencies and lower fuel costs. Cleaning can be easily accomplished from the top or side.

LE Series Oil Boiler/Natural Draft or Direct Vent

- **Oil Fired**
- **Steel Package**
- **Hot Water**
- **DOE Capacities 74 to 143 MBH**
- **AFUEs up to 86.7%**
- **LE Home Comfort System includes Energy Control 5000 Microprocessor and Alliance indirect water heater**

The Burnham LE boiler is the heart of the LE Home Comfort System, which includes the Energy Control 5000 and Alliance indirect water heater. The LE boiler allows quick, low energy start ups because it holds less water in the boiler vessel. A reversible cast iron swing door gives easy access to the combustion chamber for easy cleaning.

RSA Series Oil Boiler

- **Steel Package**
- **Hot Water**
- **Capacities: 100 to 318 MBH**

The RSA Series comes completely packaged, wired, and assembled for quick connection to water, fuel, and electrical service. The RSA comes equipped with split controls — a light sensing primary control and an operating/high limit control.

The high efficiency RSA has a large combustion area which promotes heating efficiency while maintaining low base temperatures. The RSA also features a new shell design for maximum heat transfer, and spiral turbulators which reduce stack temperatures while providing further heat transfer. A heat shield is required and available for combustible floor installations. Do not install on carpeting even with a heat shield.

Woodlander Solid Fuel Boiler

- **Steel Knockdown Unit**
- **Two Models Available**

The Burnham Woodlander line of solid fuel units gives you a variety of heating options. The Woodlander add-on boiler is furnished complete with all controls necessary for integration with your present boiler. It fires either seasoned hardwood, anthracite or bituminous coal.

If you are replacing your entire heating unit, Burnham has the Woodlander-3, named for its three fuel flexibility. It has a high efficiency oil burning section in addition to the wood or coal burning unit. Do not install on combustible flooring.

FD Series Oil, Gas, or Combination Fuel Boiler

- **Steel Package**
- **Hot Water**
- **Vertical or Optional
Horizontal Design**
- **Capacities 250 to 1,600 MBH**

The FD Series boiler is designed for forced draft firing. The rectangular shape of the FD provides a smaller footprint which can fit in areas where a sectional boiler will not.

The FD Series is ideal as an indirect hot water supply boiler for spas, car washes, laundries, commercial swimming pools and restaurants. It can also supply domestic hot water requirements for hotels, motels or apartments along with meeting their heating needs. Do not install on combustible flooring.

V9 Series Oil, Gas, or Combination Fuel Boiler

- **Cast Iron Packaged or
Knockdown Unit**
- **Hot Water or Steam**
- **Capacities 346 to 1,899 MBH**

Versatile enough to meet plan and spec job requirements, the new Burnham V9 boiler is ideal for use in light commercial installations. Applicable to both new construction and replacement jobs, the V9 provides the longevity of a cast iron boiler assembled with the time-proven integrity of cast iron nipples. The sectional design of the V9 makes for easy maneuvering and installation in existing boiler rooms. The sections and individual components fit through standard doorways and are assembled on-site with a simple ratchet, wrench, and short draw rods. Packaged and fire-tested units are also available. And the V9 can be installed in a confined area, requiring only nine inches from a right side wall and 6 1/2 feet of vertical clearance.

V11 Series Oil, Gas, Combination Fuel Boiler

- **Cast Iron Packaged or Knockdown Unit**
- **Steam or Hot Water**
- **Capacities 667 to 4,551 MBH**

Cast iron construction, sectional design, top or rear flue outlets, and a variety of options make the V11 commercial boiler the complete heating package for engineers

and contractors. Applicable to both new construction and replacement jobs, the V11 provides the longevity of a cast iron boiler assembled with the time-proven integrity of cast iron nipples. The sections and individual components fit through a standard doorway and are assembled on-site with a simple ratchet, wrench, and short draw rods. Packaged and fire-tested units are also available. Apartment buildings, schools, hospitals, and churches to name a few, can all benefit from the V11.

Baseray®

- **The Original Cast Iron Baseboard**

Baseray is the original cast iron radiant baseboard. It is designed to be inconspicuous but effective. In fact, Baseray's design enables heat to be disbursed in both horizontal and vertical directions. And its cast iron construction provides radiant heat, lifetime durability, and dependability.

Baseray assembles up to and including 6 lineal feet are shipped in one piece. Longer

assemblies are shipped in two or more pieces or sub-assemblies of 6 feet or less for assembly on the job.

Radiators

- **Cast Iron**
- **Slenderized in 4 thru 48 sections**
- **Radiant in 4 thru 44 sections**

Burnham's line of cast iron radiators is suitable for both residential and commercial applications. Both the slenderized and the radiant models supply a continuous flow of radiant heat combined with convection heat.

Duo-Rad[®]

- **Fan Coil Units**
- **Heating and Cooling**
- **10,000 to 22,000 BTU/HR**

The Duo-Rad fan coil unit combines luxurious hydronic heating with cooling in one smartly styled heat distributor. Duo-Rads are ideal for both residential and commercial applications. They are constructed of rigid welded steel.

Alliance Indirect-Fired Water Heater

- 4 Sizes - 26, 40, 53, and 79 Gal.
- Commercial Capabilities
- Rapid Recovery
- Stainless Steel Tank and Coil
- Minimal Standby Losses

The Alliance indirect-fired water heater provides more hot water than tankless coils and low BTU capacity direct-fired water heaters can supply. Available in 4 sizes, it provides abundant hot water at a minimal cost.

Energy Control 5000

- Multiple Zone Heating Control
- Water Heater Control

The Energy Control 5000 microprocessor continually monitors outdoor air temperature, indoor supply water temperature, and indoor room air temperature, maintaining the lowest water temperature necessary for comfort and boiler efficiency. The EC5000 provides control for four zones,

with add-on modules available for additional requirements.

The EC5000 also utilizes post purge, night setback, and warm weather shut down. Finally, the EC5000 helps protect your heating system by blending hot boiler water with cooler water to avoid thermal shock, and by periodically exercising pumps during the off season to keep them lubricated.

VENTING GUIDE FOR BURNHAM GAS BOILERS

®

- Vent system sizing and installation must be in accordance with National Fuel Gas Code (ANSI Z223.1-NFPA 54), Part 7, Venting of Equipment, or applicable provisions of local building codes.
 - a. Masonry chimney construction must be in accordance with Standard for Chimneys, Fireplaces, Vents, and Solid Fuel-Burning Appliances (ANSI/NFPA 211), which includes fire clay flue lining (ASTM C 315, Specifications for Clay Flue Linings) or listed chimney liner system.
 - b. An indoor chimney has no outdoor exposure below the roof line. An outdoor chimney has one or more surfaces exposed to the outdoors below the roof line. A Type B vent or listed chimney lining system passing through an unused masonry chimney is considered to be an indoor chimney.
- Burnham will not authorize removal of the automatic vent damper on a boiler certified. "For use only with automatic vent damper device..." as stated on or near rating plate.
- Statements apply to Burnham boilers installed in the United States. A revised or separate Canadian Policy Statement will be issued at a later date.



BENEFITS OF BURNHAM'S HOME HEATING TEAM MEMBERSHIP



Burnham's Home Heating Team is a unique contractor program featuring unequalled benefits to members in promoting, selling, installing and servicing Burnham products.

As a member of the Home Heating Team Burnham will:

SUPPORT YOUR BUSINESS

- 1) Safest, most efficient, and most dependable products in the industry.
- 2) Toll-free telephone number for technical as well as sales and marketing assistance.
- 3) Team marketing manual to help you identify more prospects and to help increase your closure rate.
- 4) Experienced and competent sales force to give personal assistance in service and marketing.
- 5) Home Comfort Financing supports you with the easiest to offer and most effective consumer financing programs in the industry.

HELP YOU PROMOTE YOUR BUSINESS

- 1) Co-op advertising - provides personal funding as well as distributor assisted funding to support your promotional efforts.
- 2) Advertising assistance - ad samples for newspapers, radio, yellow pages, billboards and direct mail programs as well as prepared copy for local public relations submissions.
- 3) Tie-ins to regional ad campaigns. Team members will have the opportunity to join with Burnham in regional and local advertising campaigns.

PROVIDE EDUCATION TO YOUR EMPLOYEES

- 1) Burnham Reporter magazine - periodic magazine provides valuable information concerning new products, programs and membership activities.

2) Burnham Trade Talk videos - designed to assist team members in selling, applying and servicing Burnham products.

3) Training seminars - features two formats: a classic, full-day seminar that effectively educates your staff on steam, hot water, and modular boiler applications. Or participate in "mini" seminars, covering specialized topics like proper boiler venting, radiant panel systems, upgrading older systems, electric heat conversions, as well as selling Burnham products to consumers.

REWARD YOUR COMPANY

Burnham will provide substantial financial rewards to the team members such as:

- 1) Winter Warmth Assurance - lets you offer the most flexible 5 or 10-year extended parts and labor warranty in the industry. Burnham's WWA can be used as an add-on sale or can be included in the contractor's proposal to help close the sale.
- 2) Periodic promotions - designed to increase your profitability and make selling Burnham easier and more interesting.
- 3) Burnham Trade Council - (12) tradesmen are selected to represent each region of the country and Canada on the Trade Council. Burnham is committed to listen to and react to the needs and opportunities put forth at periodic meetings. Each region has a representative that is representing the trade in that area and the member is available to listen to your needs and desires.

To join the HHT, contact your local Burnham sales office and ask to have a Burnham representative contact you about membership. The offices are listed in the front of this book.

BURNHAM REGIONAL OFFICES

Northeast Region

(CT, ME, MA, NH, RI and VT)

Burnham Sales Corporation

19-27 Mystic Avenue

Somerville, MA 02145

617-625-9735

Metropolitan New York Region

(NY, NJ)

Middle Atlantic

(PA, DC, DE, MD, WV and Eastern OH)

Central and Western Regions

Burnham Corporation

Regional Sales Offices

PO Box 3079

Lancaster, PA 17604

717-481-8400

HOME HEATING TEAM MEMBERS:

Call your special 1-800#

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